

AP DYNAMICS INC.
VIBRATIONS AND ROTATING EQUIPMENT

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Acoustical and Mechanical Study Report

Husky 12517 & 31833

Prepared for:

Enerflex Systems Ltd.
David Campbell

1 Summary and Recommendations

2 Acoustical Study Detailed Results

3 Mechanical Study Detailed Results

4 Methodology

5 References

6 Appendix

APD Project ID:	APD-10-825-EFX-AM
Document No.:	R825-003 Rev 0
Date Issued:	October 18 2010

Equipment Summary

CLIENT:	Enerflex
PROJECT NAME:	Husky 12517 & 31833
APD PROJECT ID:	APD-10-825-EFL-AM
CLIENT PROJECT ID:	Husky 12517 & 31833
COMPRESSOR FRAME:	Ariel JGJ/2
NUMBER OF STAGES:	1 or 2
NUMBER OF CYLINDER:	2
THROW 1:	7-3/8RJ (7.375" bore)
THROW 2:	6-3/8RJ (6" bore)
SUCTION PRESSURE:	765 - 1265 kPa
SUCTION TEMP.:	124 - 127 °F
DISCHARGE PRESSURE:	1600 - 3500 kPa
DISCHARGE TEMP.:	120 °F
FLOW RATE:	2.845 - 5.092 MMSCFD
DRIVER:	Waukesha F18GL nat. gas
POWER:	298 kW
SPEED:	1400 - 1800 RPM

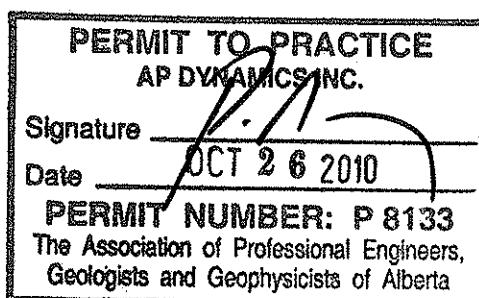
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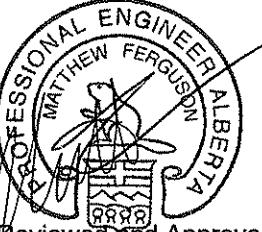
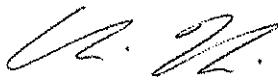
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REV	DESCRIPTION OF CHANGE	DATE ISSUED:	BY:	APRV'D.
0	Released to Client	October 25 2010	CMK	MGF



 Reviewed and Approved By: Matthew Ferguson, P.Eng.	 Acoustical Analysis By: Christopher Kowalski	 Mechanical Analysis By: Christopher Kowalski
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FLOW RATE:	2.845 – 5.092 MMSCFD
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POWER:	298 kW
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REV	DESCRIPTION OF CHANGE	DATE ISSUED:	BY:	APRVD.
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Reviewed and Approved By: Matthew Ferguson, P.Eng.	Acoustical Analysis By: Christopher Kowalski	Mechanical Analysis By: Christopher Kowalski

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1. SUMMARY AND RECOMMENDATIONS

1.1 SCOPE OF WORK

Enerflex Ltd. commissioned AP Dynamics Inc. to conduct an acoustical and mechanical study on a set of reciprocating air compressors operating in parallel for the Husky Project as shown in Figure 1.1 and Figure 1.2. The studies were performed as per API 618 4th edition M2-M5. Implementing the recommendations listed in section 1.2 and 1.3 will reduce pulsations and acoustical shaking forces to acceptable levels and ensure that no mechanical natural frequencies will be excited in the main process piping.

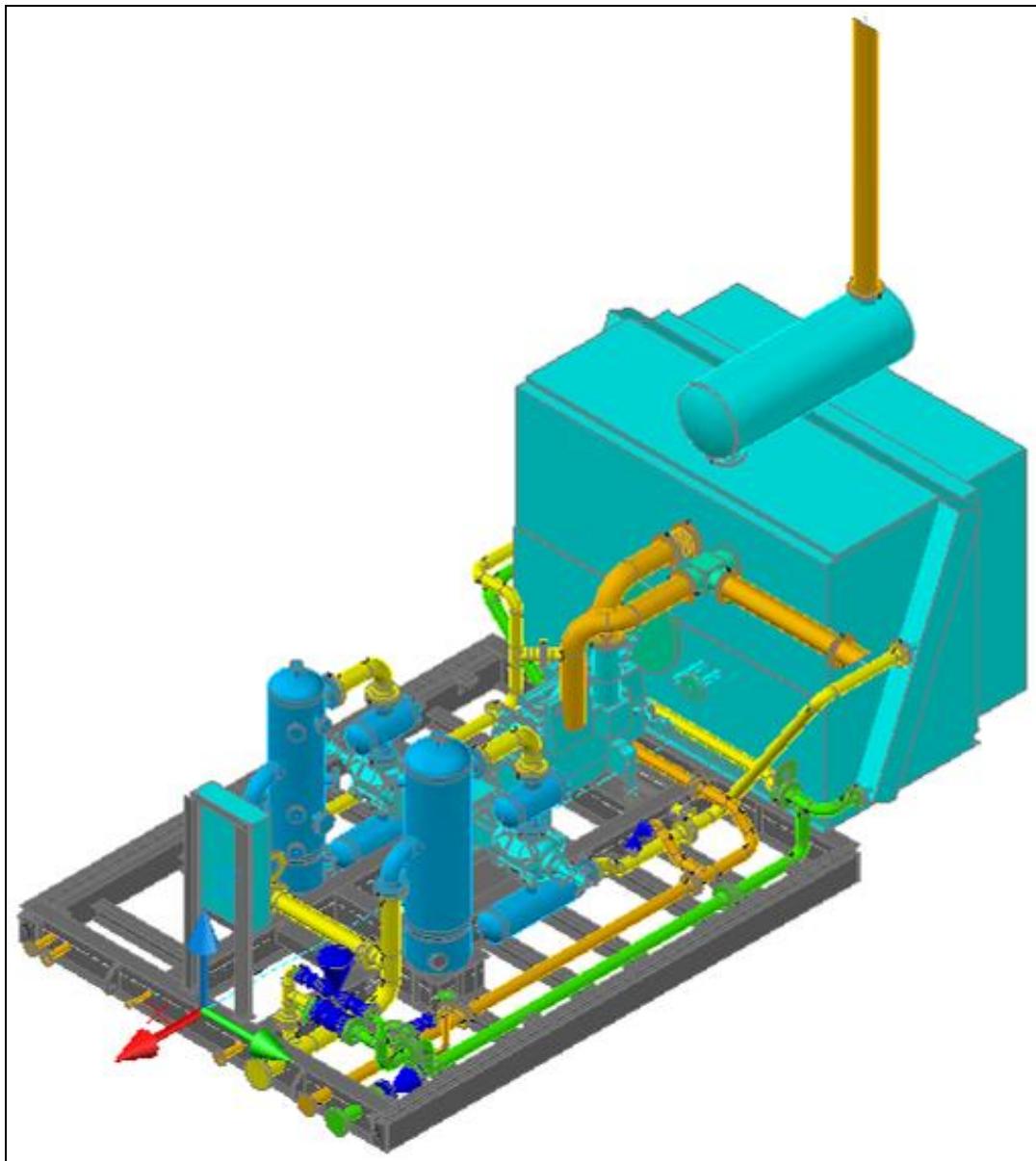


Figure 1.1 Isometric View of Proposed Unit

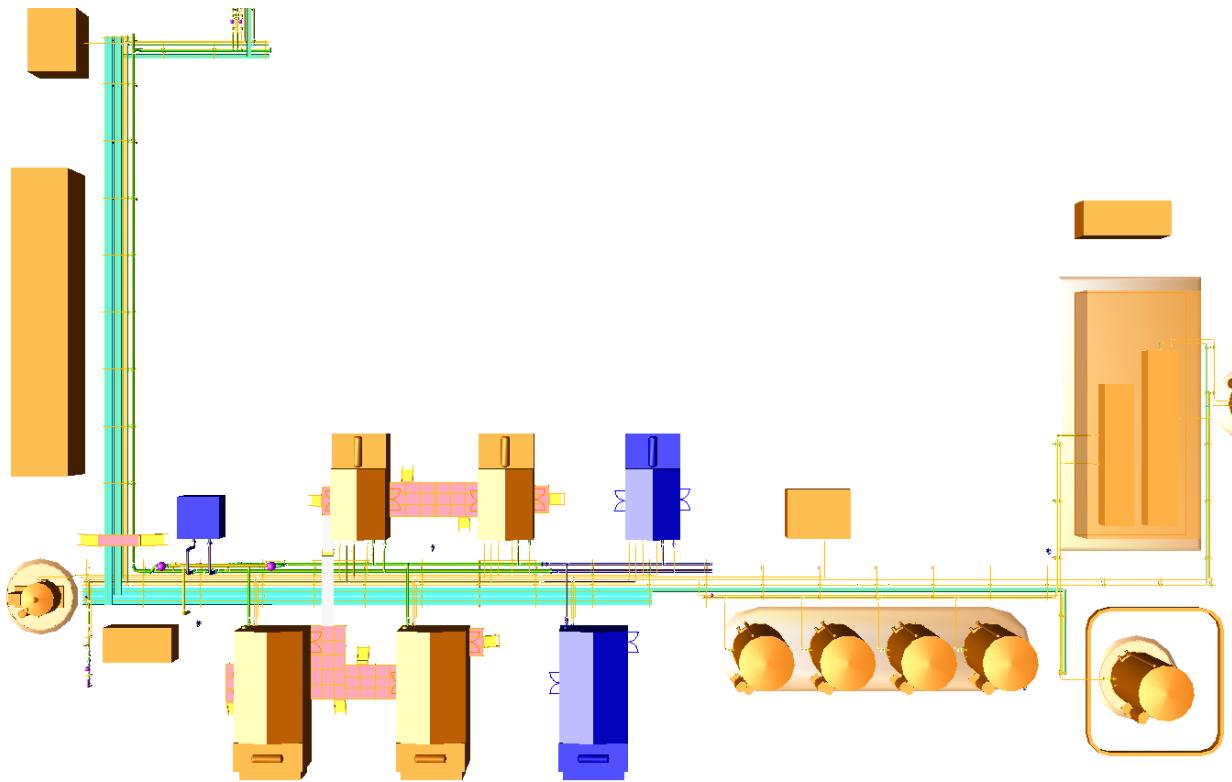


Figure 1.2 Plant Layout

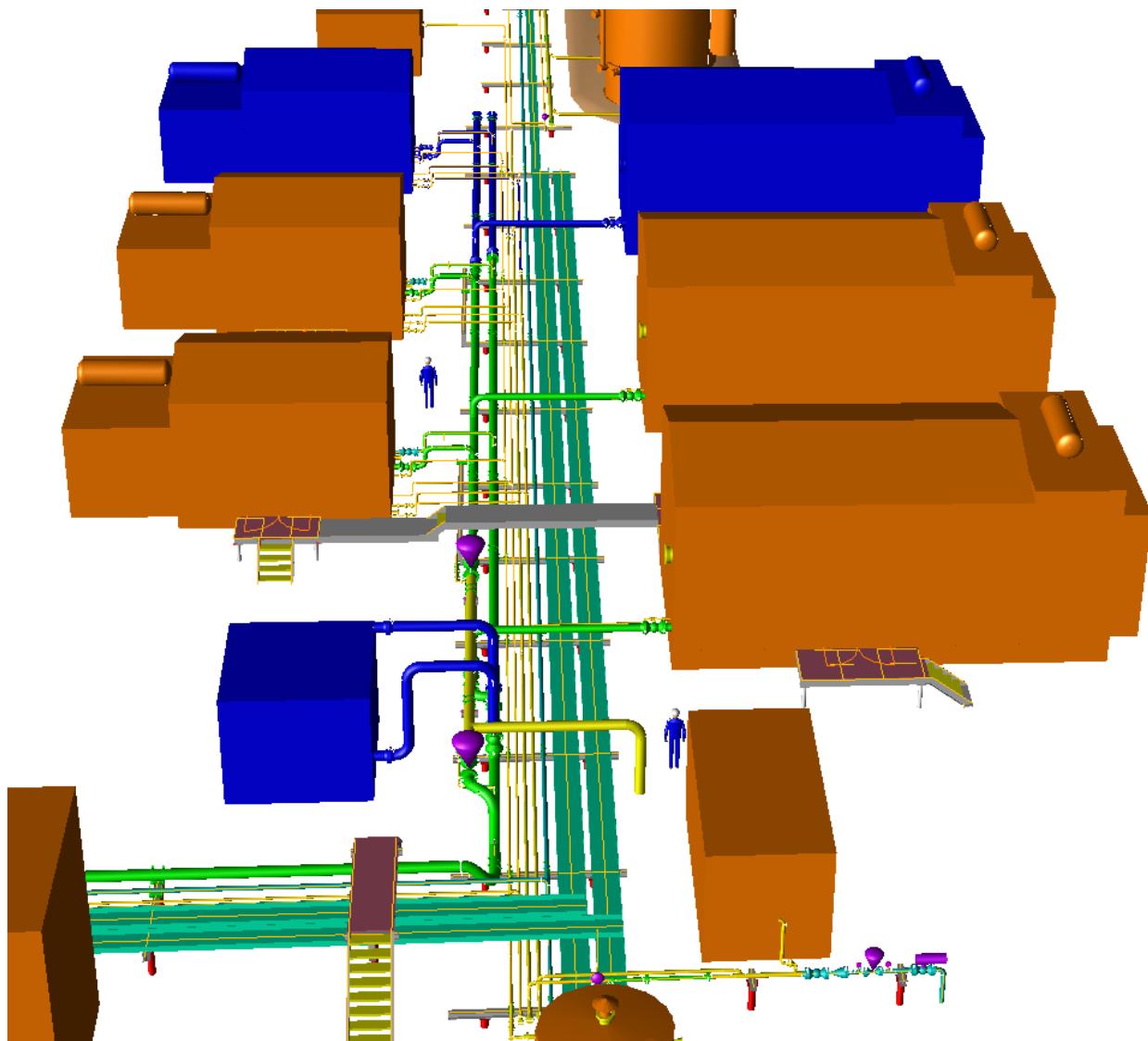


Figure 1.3 Pipe Rack layout 1

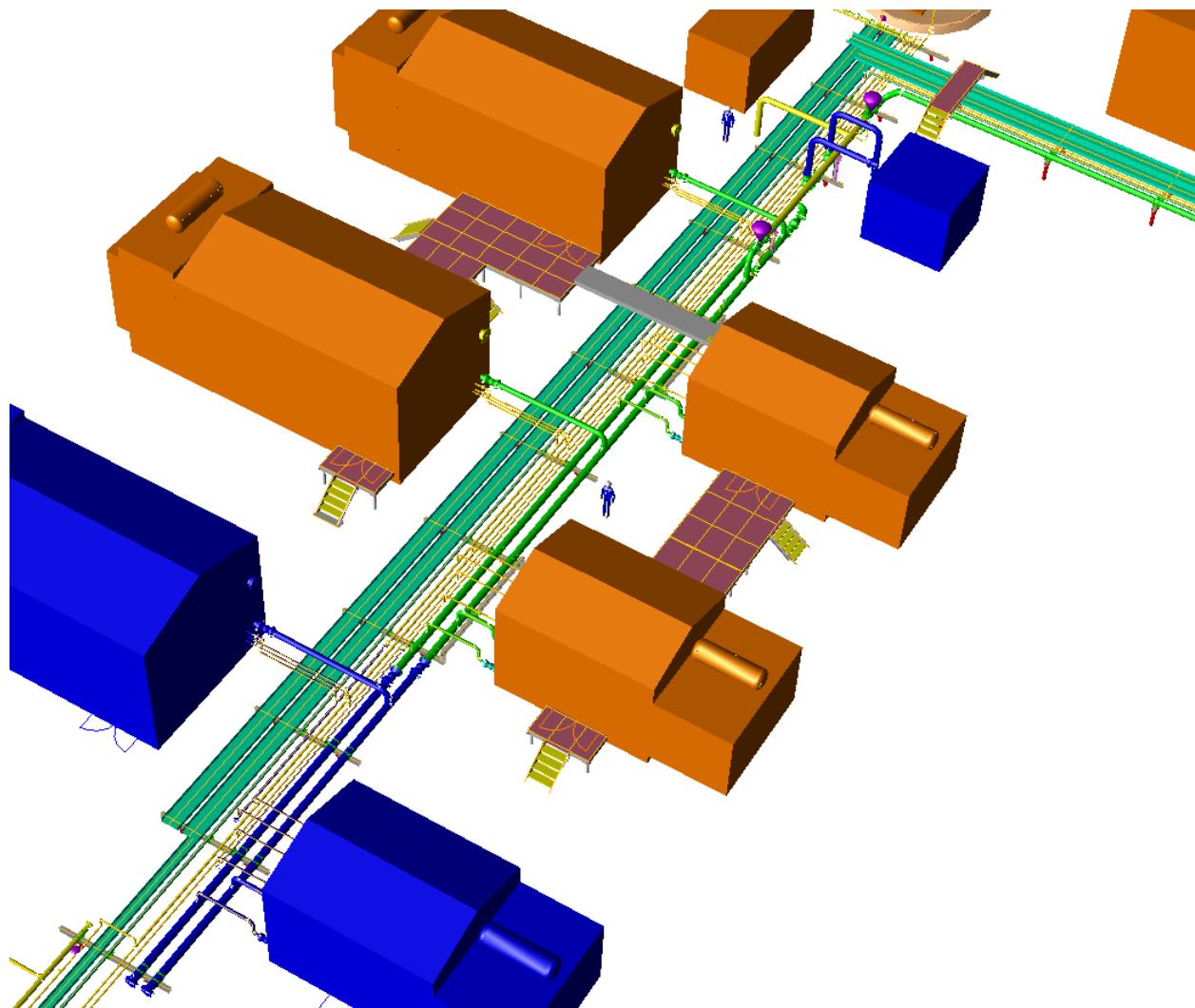


Figure 1.4 Pipe rack layout 2

1.2 ACOUSTICAL RECOMMENDATIONS

The recommended pulsation bottles seam to seam and outer diameters are given in Table 1.1 with corresponding drawing shown in Figure 1.5. Recommended orifice plate locations and inner diameters are given in Table 1.2. In addition to the bottle and orifice recommendations, the Stage 1 Cooler (IC-1) cooler outlet piping should be modified with expanded piping as shown in Figure 1.6. An additional set of flanges and orifice plates should be added at all 3 compressor discharge header tie-in spools as shown in Figure 1.7.

Table 1.1 Pulsation Bottles Summary

Bottle Location	OD	S/S	Notes
Suction Bottles	12-3/4"	20"	LWNF outlet nozzles
Cylinder 1 Discharge Bottle	12-3/4"	36"	2.5" XS choke tubes per Figure 1.5
Cylinder 2 Discharge Bottle	12-3/4"	36"	2.5" XS choke tubes per Figure 1.5

Table 1.2 Orifice Plate ID Summary

Location	ID
Cylinder 1 Suction Bottle Inlet Flange	3-1/4"
Cylinder 1 Discharge Flange	3"
Cylinder 2 Scrubber Inlet Flange	3-3/8"
Cylinder 2 Scrubber Outlet Flange	3-11/16"
Cylinder 2 Discharge Flange	2-3/4"
Compressor 1, 2 and 3 Discharge Header Tie-In Spools (see Figure 1.7)	3"

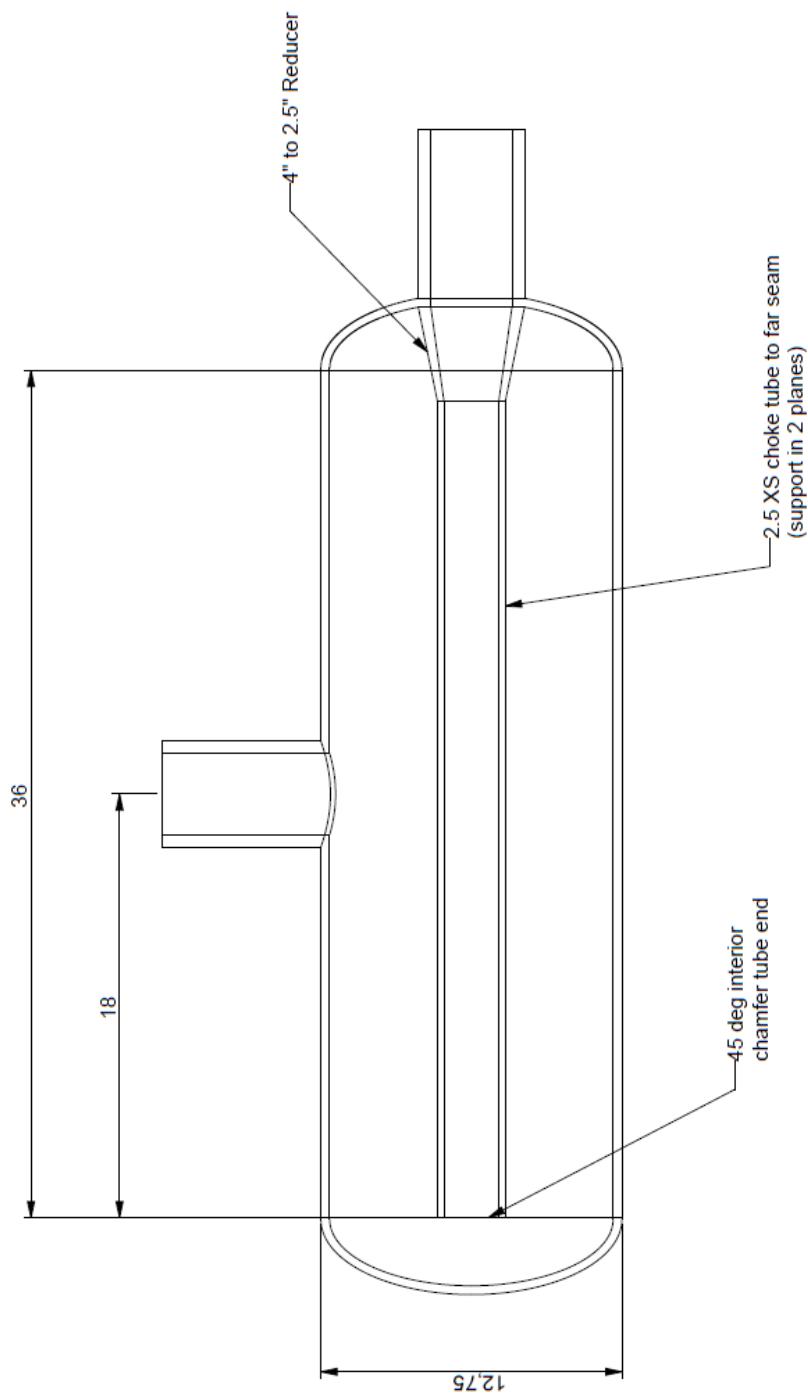


Figure 1.5 Stage 2 Discharge Bottle (see D825-002)

The stage one cooler (IC-1) outlet piping must be altered to minimize pulsations within the piping. The changes are shown below in Figure 1.6. Piping geometry (length) remains unchanged, only pipe diameter is altered.

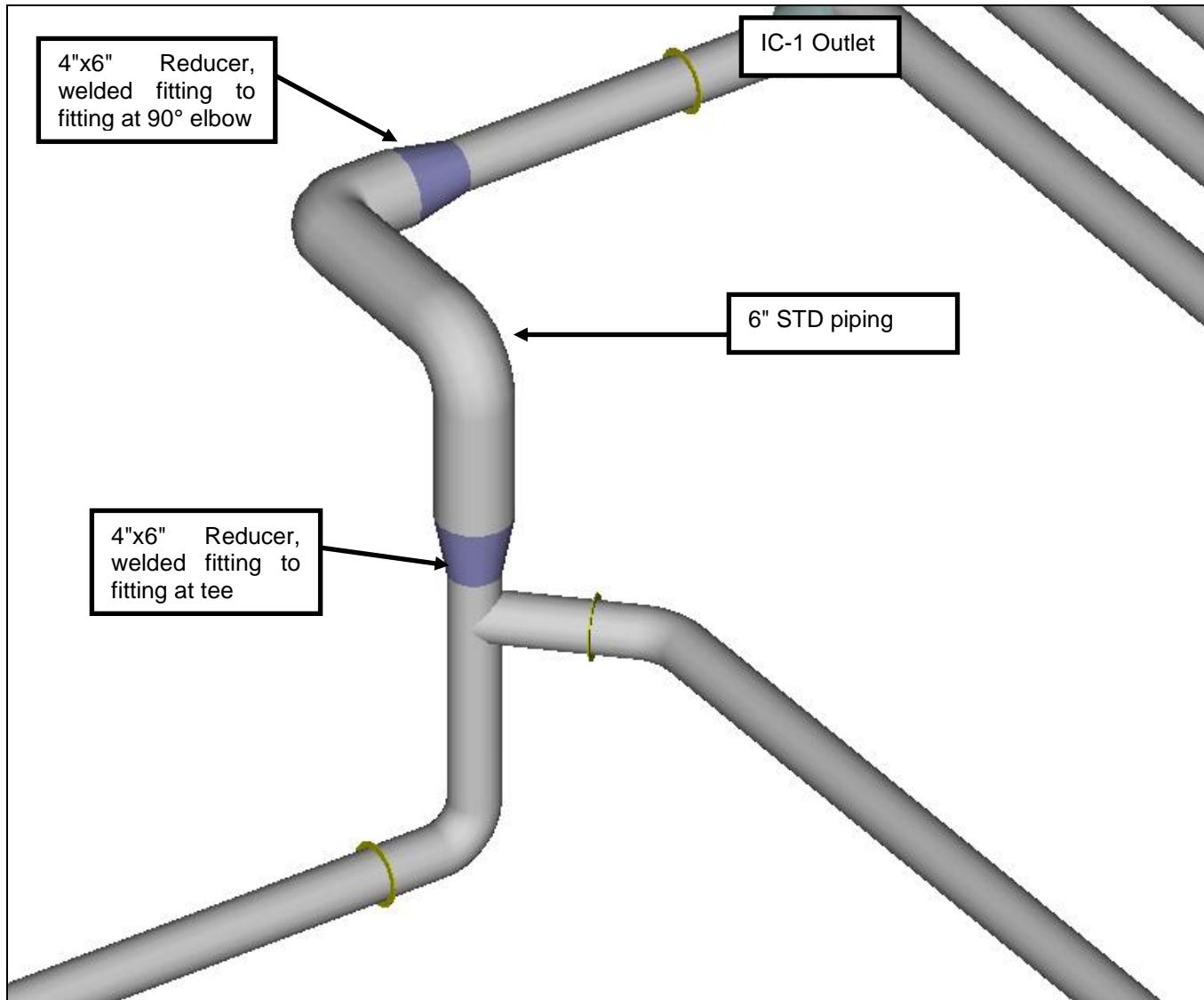


Figure 1.6 Stage 1 Cooler (IC-1) Outlet Piping Changes

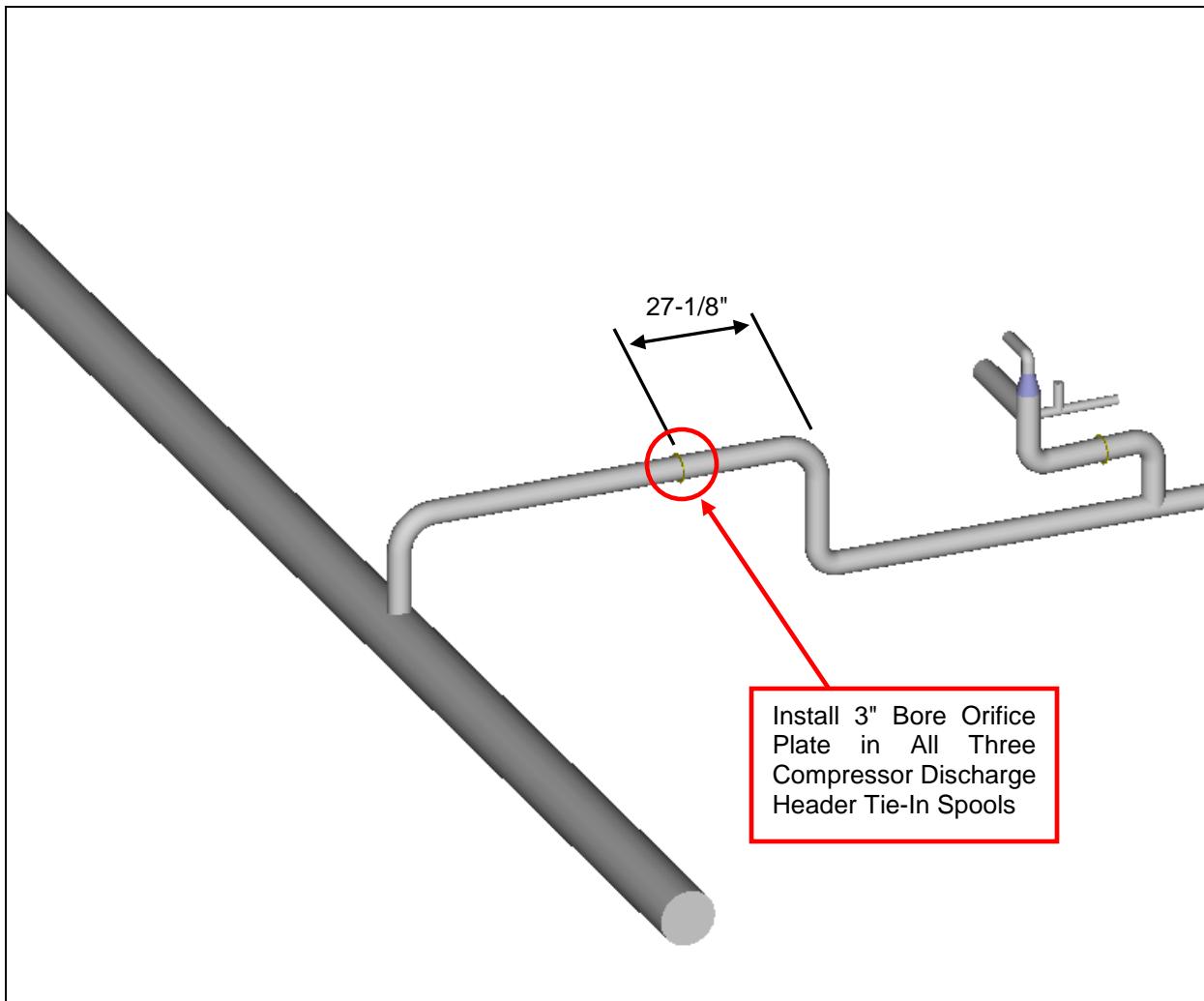


Figure 1.7 Location of Compressor Discharge Header Tie-In Orifice Plate(s)

1.3 MECHANICAL RECOMMENDATIONS

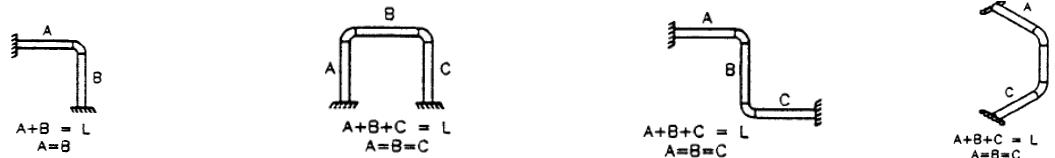
In order to meet the API 618 4th edition M4-M5 natural frequency guidelines the following modifications are required. Piping should be clamped per Table 1.3 through Table 1.4.

1. Long weld-neck flanges should be used for stage 1 and 2 suction bottle outlet nozzles.
2. Bottle wedge supports are required for both stage 1 and stage 2 discharge bottles as shown in Figure 1.8 and Figure 1.9. All wedges should be loose at start up and then tightened after the unit reaches operating temperature so that the thermal expansion of the cylinder and discharge bottle does not put a large vertical force on the cylinder during operation.
3. Add flange grabber as shown to stage 2 suction inlet pipe spool as shown in Figure 1.10.
4. Stage 1 scrubber skirt wall thickness should be increased to 25.4mm and skirt base should be welded to supporting wide flange beams.
5. Stage 2 scrubber skirt wall thickness should be increased to 25.4mm and skirt base should be welded to supporting wide flange beams.
6. The skid should have 4 piles under each corner of the scrubber bases.
7. Install 1/2" to 3/4" full depth gussets on both sides of the wide flange scrubber base beams.
8. Pile caps should be welded on all sides to the main skid members.
9. Ensure flanged connections meet ASME B31.3 2004 edition, paragraph 335.1, which prohibits distorting piping to bring it into alignment for joint assembly and specifies, before bolting up, flange faces shall be aligned to the design plane within 1/16"/ft ; flange bolt holes shall be aligned within 1/8" maximum offset.

Table 1.3 Pipe Support Spans (in), Straight Pipe

NPS	Cantilevered	Simply Supported	Fixed - Simply Supported	Fixed-Fixed
3/4	23.67	39.64	49.51	59.71
1	26.49	44.36	55.41	66.82
1-1/4	29.76	49.84	62.25	75.08
1-1/2	31.84	53.32	66.60	80.32
2	35.60	59.61	74.46	89.81
2.5	39.17	65.59	81.93	98.81
3	43.22	72.37	90.39	109.02
3.5	46.20	77.36	96.64	116.55
4	49.00	82.06	102.50	123.62
5	54.48	91.24	113.96	137.44
6	59.46	99.56	124.37	149.99
8	67.84	113.60	141.90	171.14
10	75.74	126.83	158.42	191.06
12	82.49	138.12	172.53	208.08
14	86.43	144.73	180.79	218.04
16	92.40	154.73	193.27	233.10
18	98.01	164.11	205.00	247.23
20	103.31	172.99	216.09	260.61
24	113.17	189.50	236.71	285.48

Note: Clamping main piping spools to each skid stringer beam should satisfy maximum pipe span clamping requirements.

Table 1.4 Pipe Support Spans (in), Spools With Elbows

NPS	L-Bend, Out of plane	L-Bend, In plane	U-Bend, out of Plane	U-bend, in plane	Z-Bend, Out of Plane	Z-Bend, In Plane	3D Bend
3/4	51.25	97.24	54.56	61.42	61.03	59.71	57.26
1	57.35	108.82	61.06	68.74	68.30	66.82	64.08
1-1/4	64.44	122.26	68.60	77.23	76.74	75.08	72.00
1-1/2	68.94	130.80	73.39	82.62	82.10	80.32	77.03
2	77.08	146.24	82.05	92.38	91.79	89.81	86.12
2.5	84.80	160.90	90.28	101.63	100.99	98.81	94.75
3	93.57	177.53	99.61	112.14	111.43	109.02	104.55
3.5	100.03	189.79	106.49	119.88	119.12	116.55	111.77
4	106.10	201.30	112.95	127.15	126.35	123.62	118.55
5	117.96	223.82	125.58	141.38	140.48	137.44	131.81
6	128.73	244.25	137.05	154.28	153.30	149.99	143.84
8	146.88	278.69	156.37	176.04	174.92	171.14	164.12
10	163.98	311.13	174.57	196.53	195.28	191.06	183.23
12	178.59	338.84	190.12	214.03	212.67	208.08	199.54
14	187.14	355.06	199.22	224.28	222.85	218.04	209.10
16	200.06	379.58	212.98	239.76	238.24	233.10	223.53
18	212.19	402.60	225.90	254.31	252.69	247.23	237.09
20	223.67	424.38	238.11	268.06	266.36	260.61	249.92
24	245.02	464.89	260.84	293.65	291.79	285.48	273.77

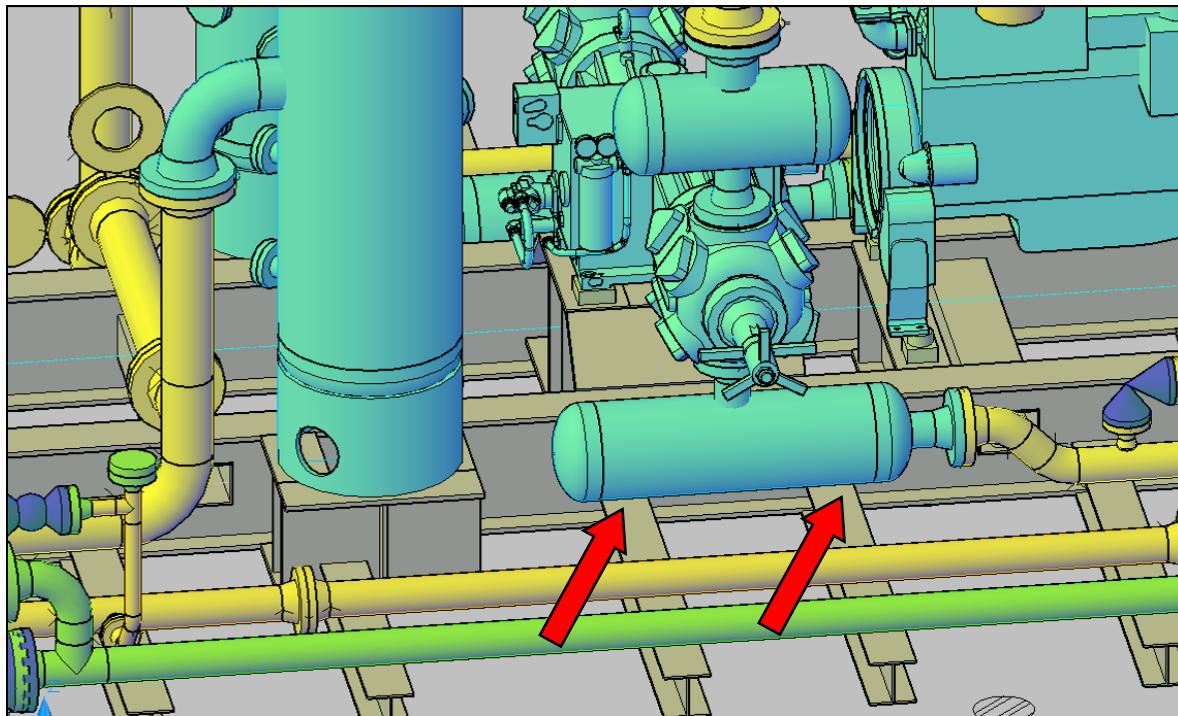


Figure 1.8 Stage 1 Discharge Bottle Wedge Locations

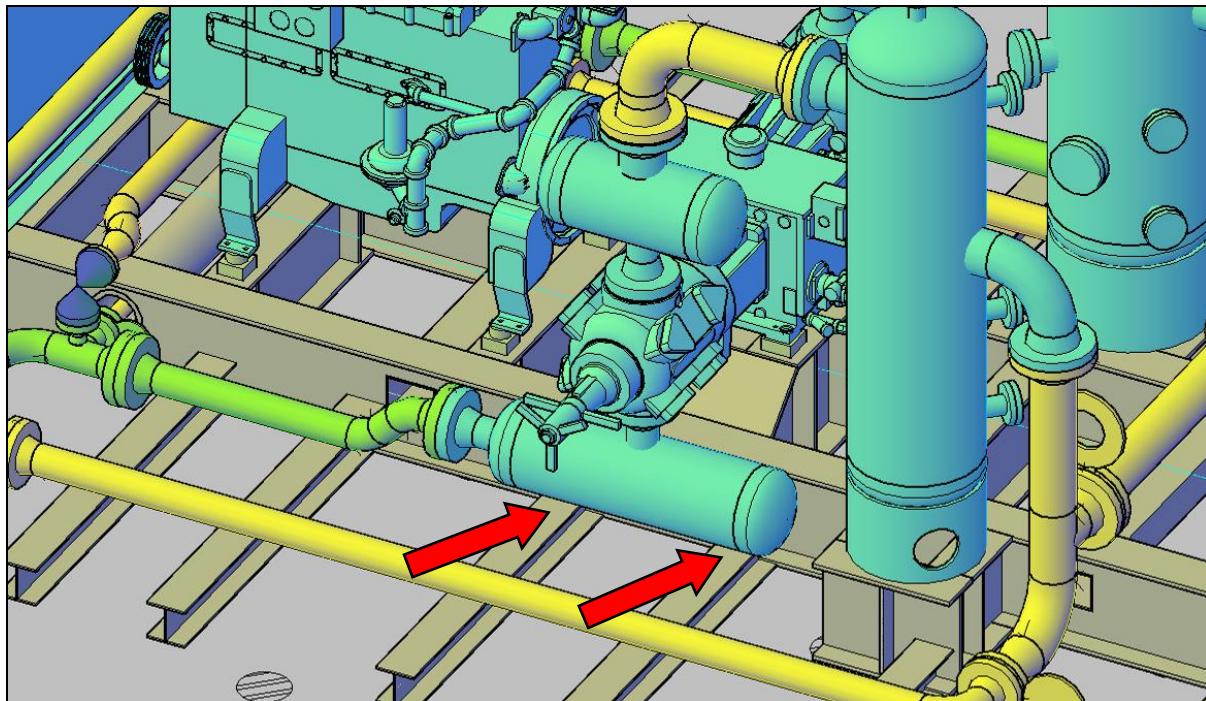


Figure 1.9 Stage 2 Discharge Bottle Wedge Locations

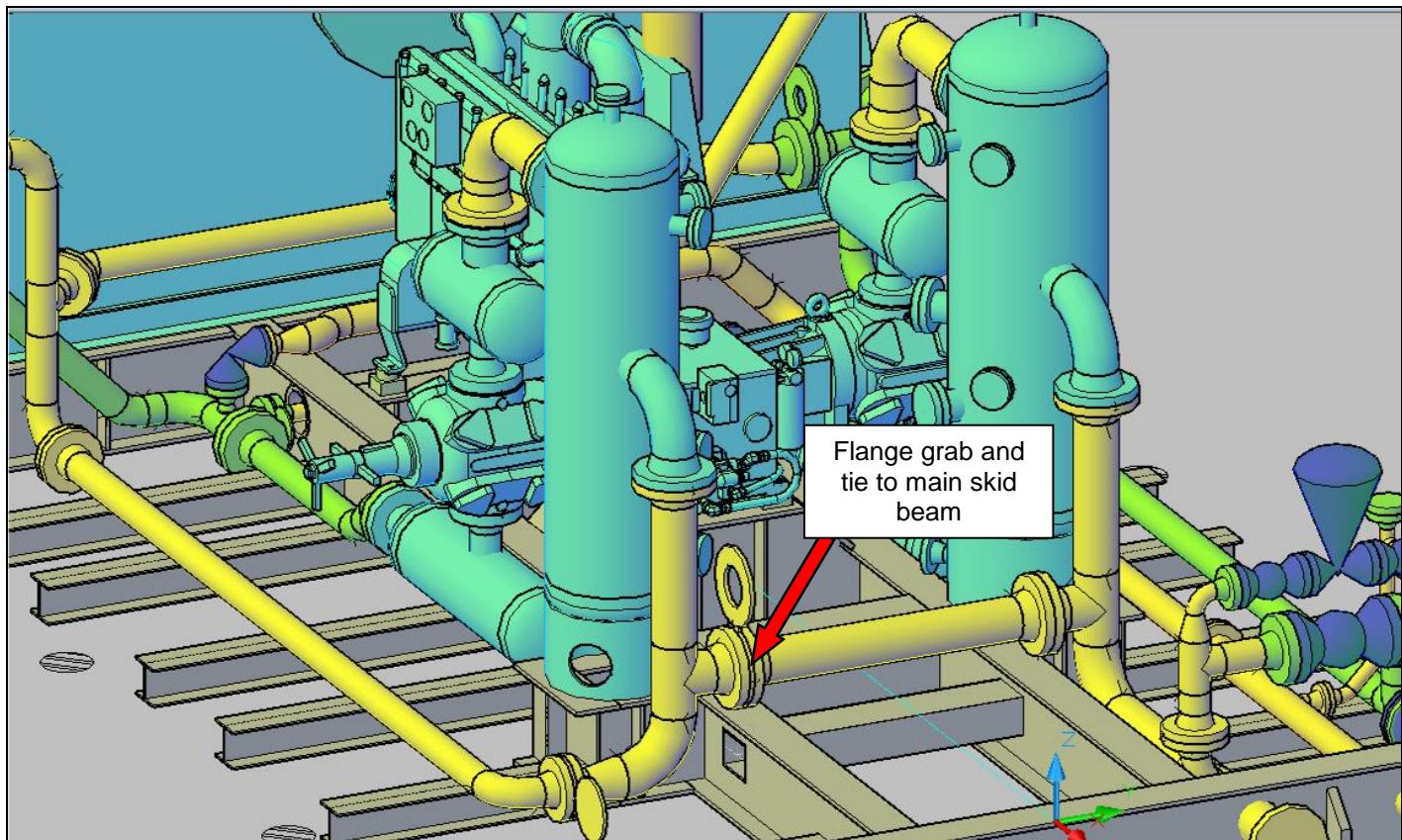


Figure 1.10 Stage 2 Scrubber Suction Spool Flange Grabber Location

2. ACOUSTICAL STUDY DETAILED RESULTS

2.1 LOAD CASES ANALYZED

The client provided 14 load cases to be analyzed, which were each split into 2 to cover the entire 1400 rpm to 1800 rpm range of the unit. The load cases are summarized in Table 2.1.

Table 2.1 Load Cases Summary

# Stages	Case	Speed, RPM	Flow Rate, MMSCFD	Suction Pressure, kPa	Suction Temp, °F	Discharge Pressure, kPa	Discharge Temp, °F	Throw 1 HE Clr, %	Throw 2 HE Clr, %
1	1	1800	5.08	965.53	125	1600	120	71.03	83.77
	2	1800	3.493	765.53	126	1600	120	71.03	83.77
	3	1800	5.092	1065.53	126	2000	120	71.03	83.77
	4	1800	3.915	915.53	127	2000	120	71.03	83.77
	1b	1600	4.516	965.53	125	1600	120	71.03	83.77
	2b	1600	3.105	765.53	126	1600	120	71.03	83.77
	3b	1600	4.526	1065.53	126	2000	120	71.03	83.77
	4b	1600	3.48	915.53	127	2000	120	71.03	83.77
2	5	1800	3.976	1065.53	124	2000	120	19.38	19.29
	6	1800	3.265	865.53	124	2000	120	19.38	19.29
	7	1800	4.665	1265.53	124	2400	120	19.38	19.29
	8	1800	3.243	865.53	124	2400	120	19.38	19.29
	9	1800	4.641	1265.53	124	2800	120	19.38	19.29
	10	1800	3.222	865.53	125	2800	120	19.38	19.29
	11	1800	4.618	1265.53	124	3200	120	19.38	19.29
	12	1800	3.201	865.53	125	3200	120	19.38	19.29
	13	1800	4.5	1265.53	124	3500	120	31.77	19.29
	14	1800	3.539	965.53	125	3500	120	19.38	19.29
	5b	1600	3.534	1065.53	124	2000	120	19.38	19.29
	6b	1600	2.902	865.53	124	2000	120	19.38	19.29
	7b	1600	4.147	1265.53	124	2400	120	19.38	19.29
	8b	1600	2.883	865.53	124	2400	120	19.38	19.29
	9b	1600	4.125	1265.53	124	2800	120	19.38	19.29
	10b	1600	2.864	865.53	125	2800	120	19.38	19.29
	11b	1600	4.105	1265.53	124	3200	120	19.38	19.29
	12b	1600	2.845	865.53	125	3200	120	19.38	19.29
	13b	1600	4	1265.53	124	3500	120	31.77	19.29
	14b	1600	3.146	965.53	125	3500	120	19.38	19.29

2.2 ACOUSTICAL SYSTEM DEFINITION

Every reciprocating compressor can be divided in $(n + 1)$ independent acoustical systems, where $n =$ the number of stages. The piston in the compressor's cylinder is the source of acoustical excitation and in order for pulsations to be transmitted into the piping, the relevant valves must be open (i.e. for the suction system, the cylinder's suction valves must be open). In order to achieve efficient compression, both suction and discharge valves should not be open at the same time thus there is no path for pulsations to travel across the cylinder from the suction into the discharge system and vice versa. Each system can be analyzed independently.

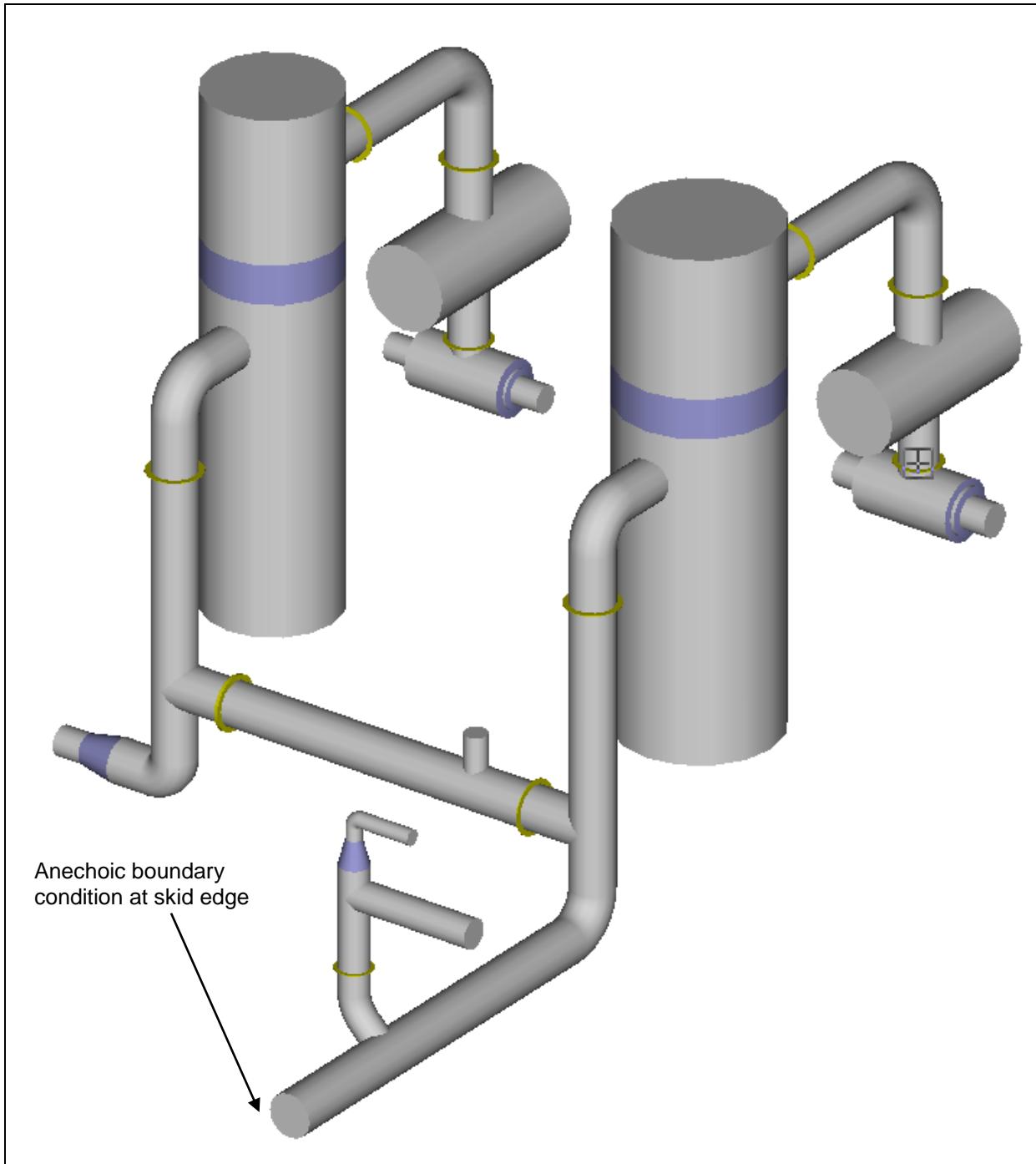


Figure 2.1 Single Stage Suction System

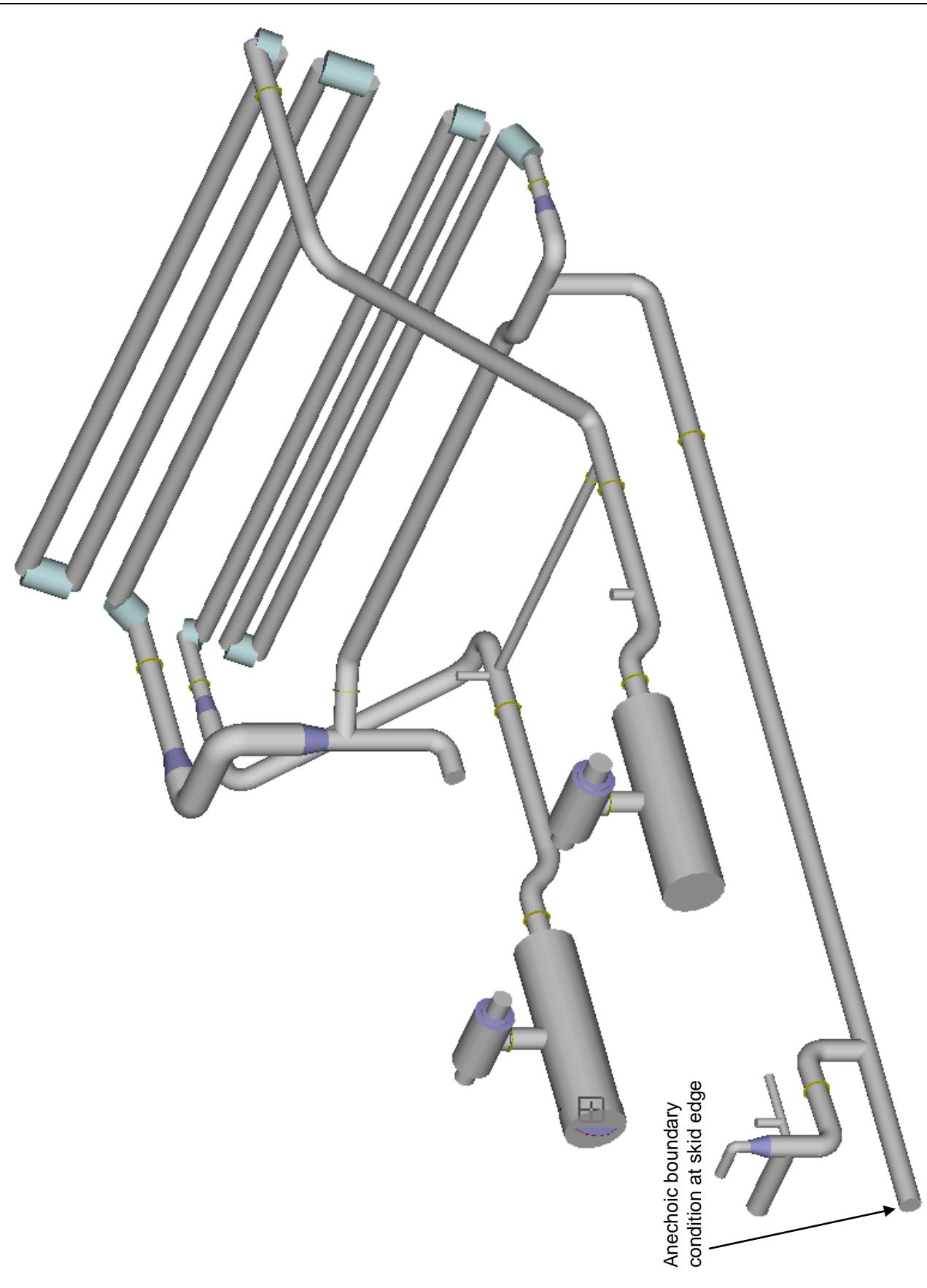


Figure 2.2 Single Stage Discharge System

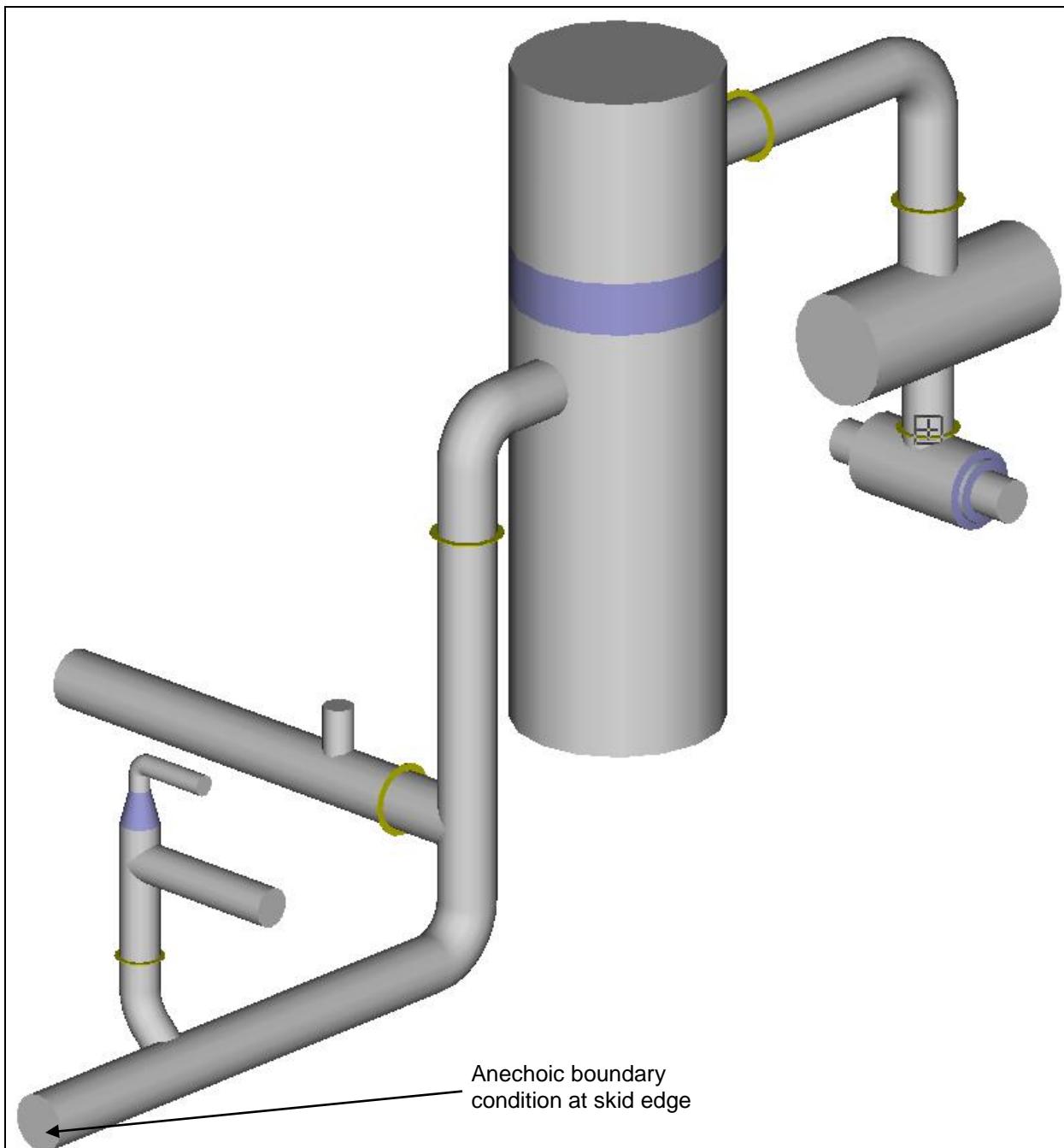


Figure 2.3 Two Stage Suction System

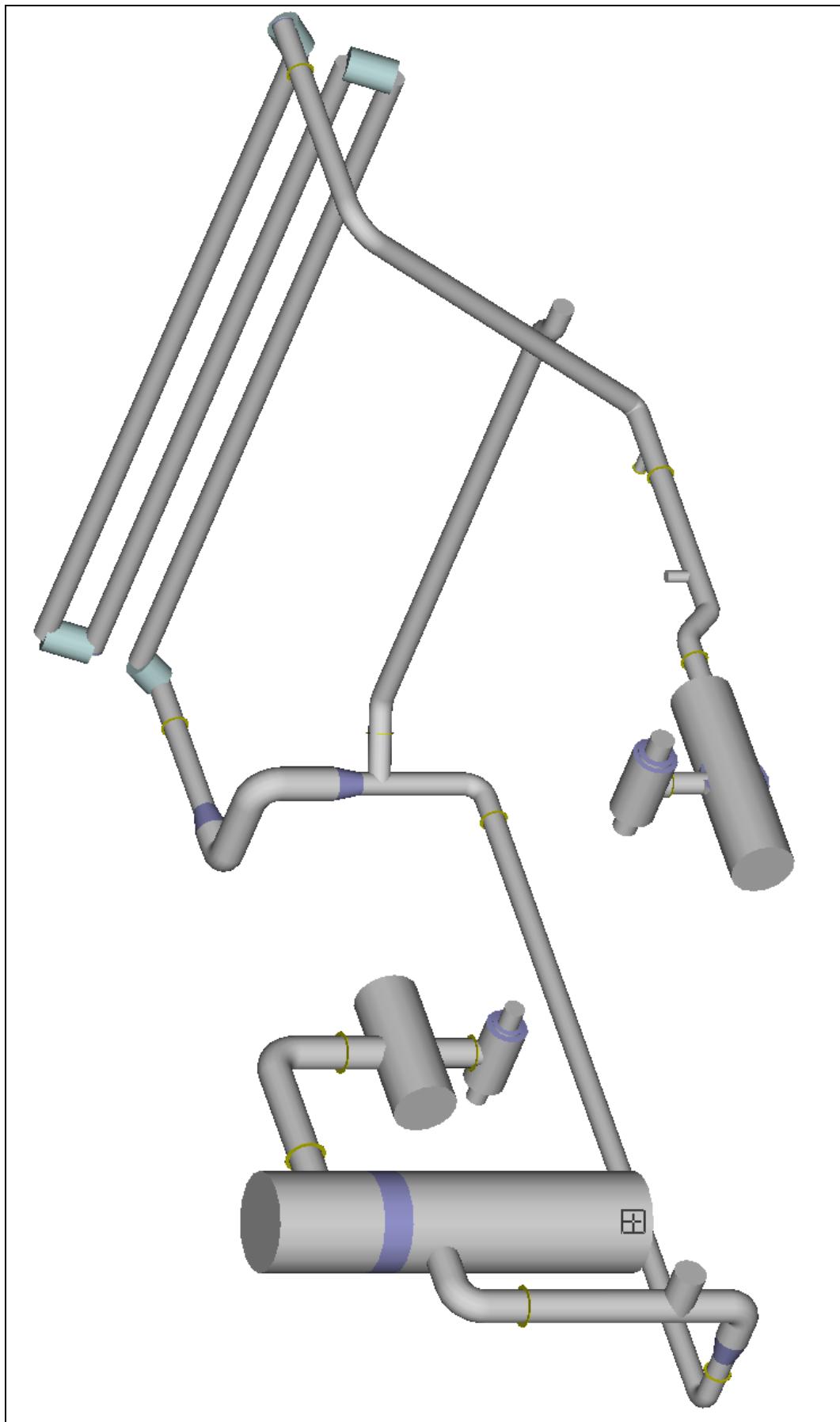


Figure 2.4 Two Stage Interstage System

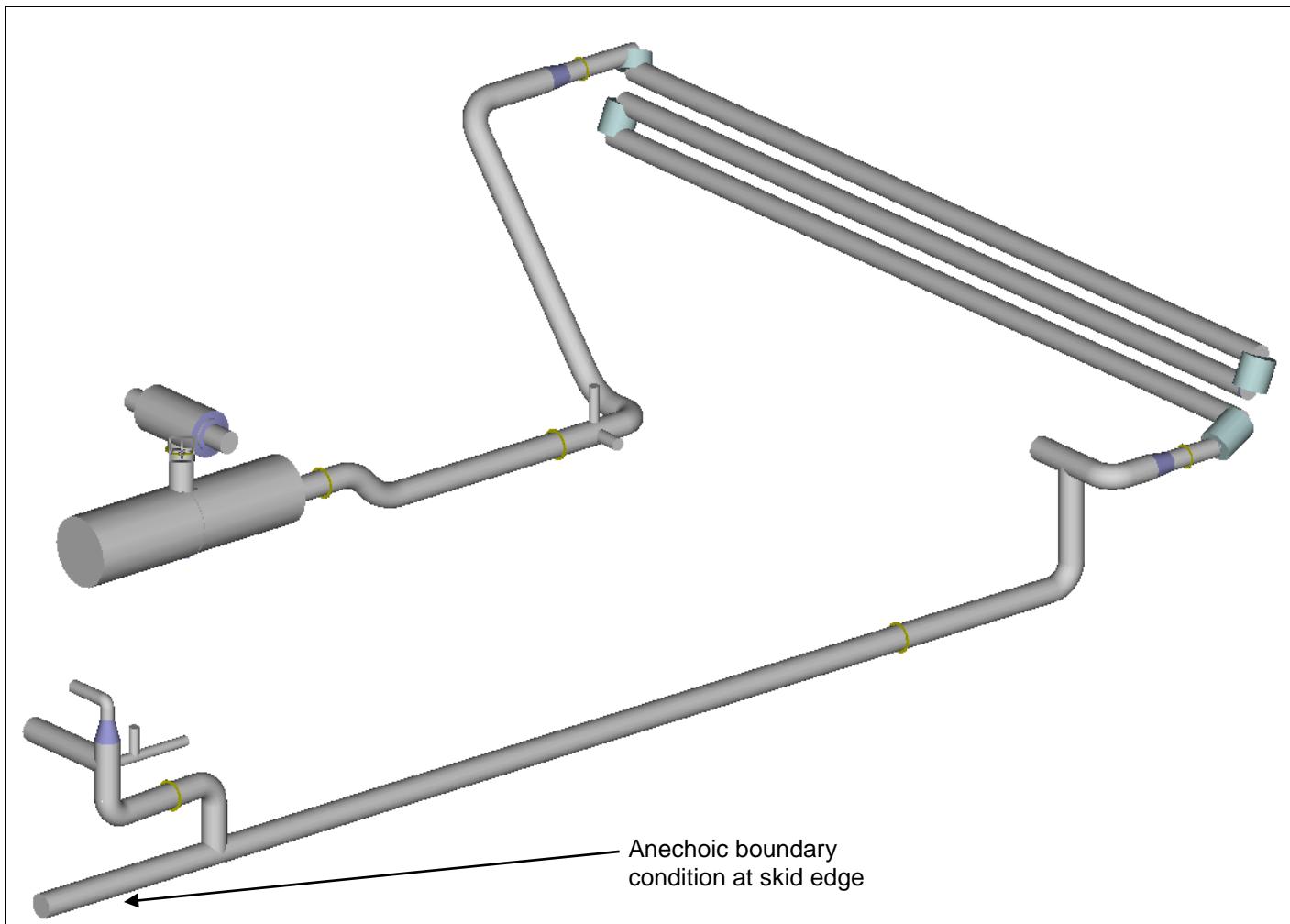


Figure 2.5 Two Stage Discharge System

2.3 SINGLE STAGE SUCTION RESULTS

Pulsation levels in the single stage suction system were 2.32x above API 618 4th edition guidelines for the worst as found case shown in Figure 2.6. After the installation of the recommended orifice plates pulsation levels are reduced to 1.01x guidelines as shown in Figure 2.7. Selected shaking forces before and after the recommendations are shown in Figure 2.8 through Figure 2.9. Shaking forces were as high as 550 N for the as found case. Shaking forces are well below guidelines before and after modifications.

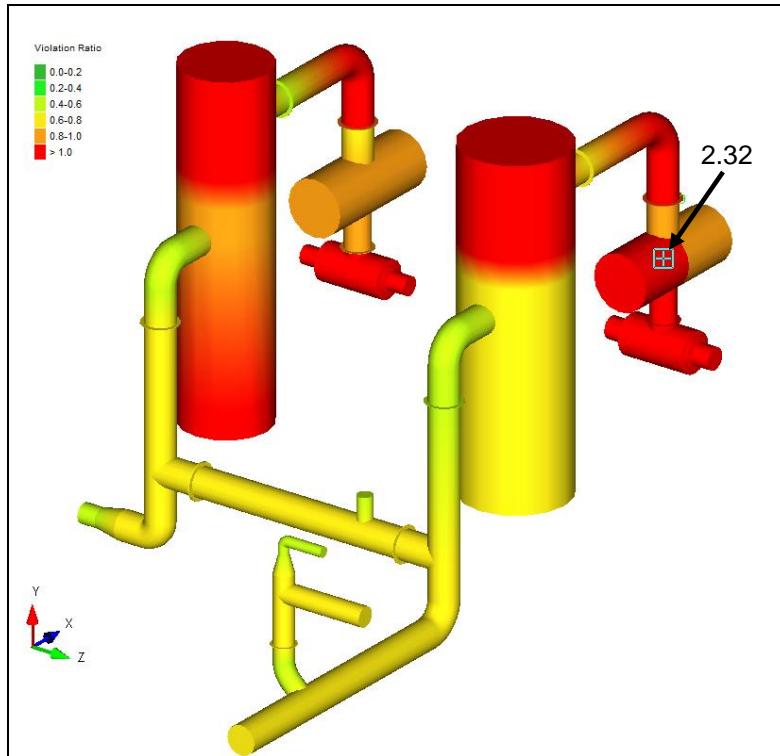


Figure 2.6 Pulsation Violation Ratio in Single Stage Suction System – As Found

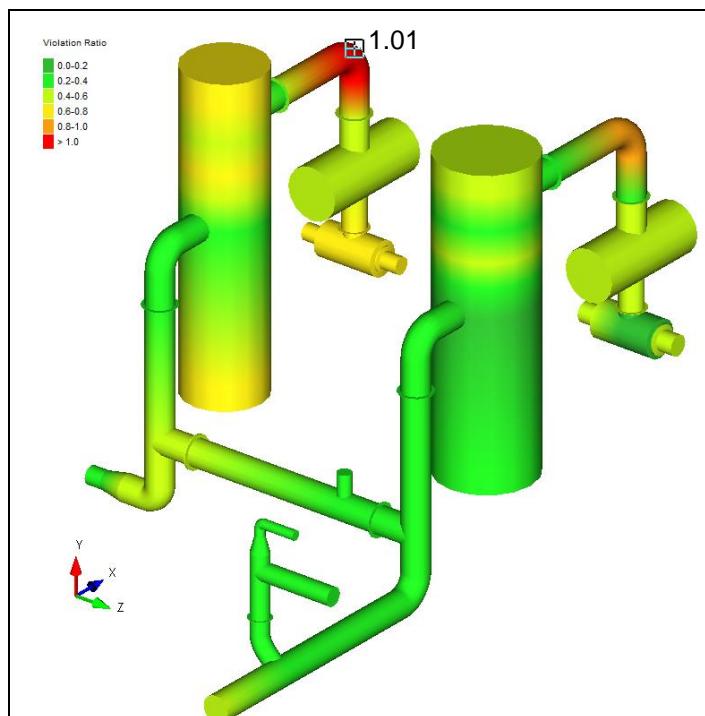


Figure 2.7 Pulsation Violation Ratio in Single Stage Suction System – After Modifications

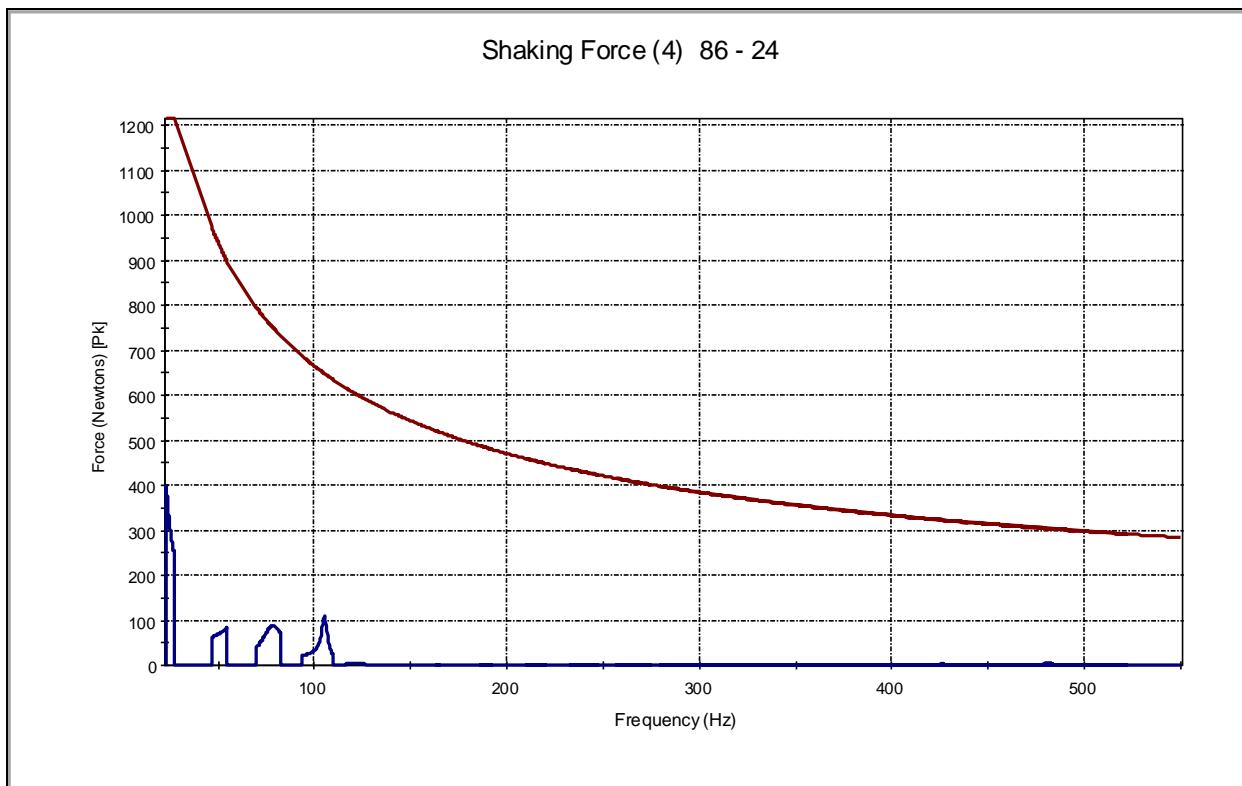


Figure 2.8 Shaking Force in Cylinder 1 Scrubber Outlet Spool – As Found

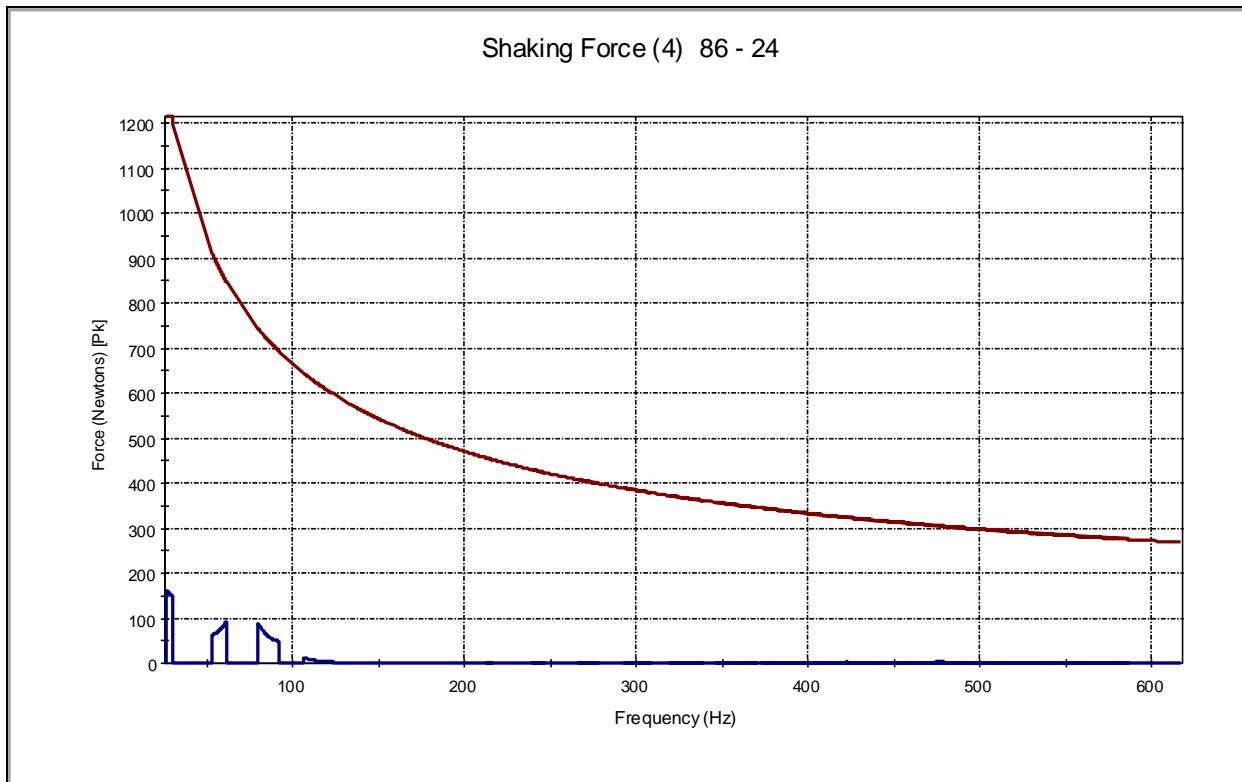


Figure 2.9 Shaking Force in Cylinder 1 Scrubber Outlet Spool – After Modifications

2.4 SINGLE STAGE DISCHARGE RESULTS

Pulsation levels in the single stage discharge system were 4.39x above API 618 4th edition guidelines for the worst as found case shown in Figure 2.10. After the implementation of the recommended changes pulsation levels are reduced to 1.73x guidelines shown in Figure 2.11. Selected shaking forces before and after the recommendations are shown in Figure 2.12 through Figure 2.16. Shaking forces were as high as 670 N for the as found case. After modifications, all shaking forces within the system are well below guideline.

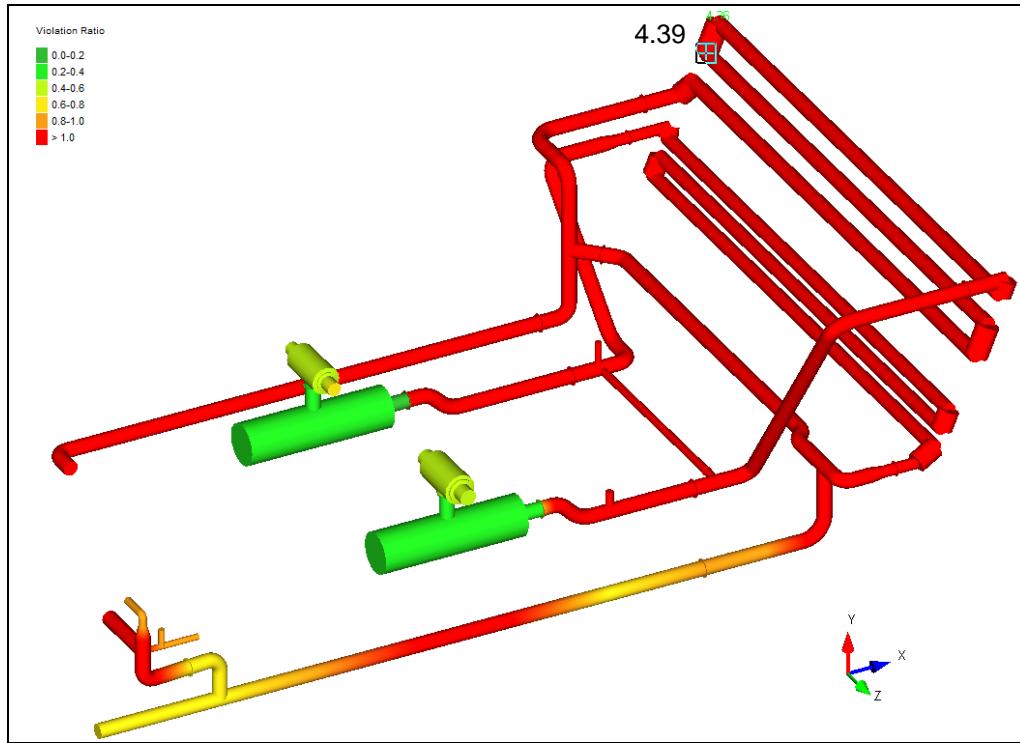


Figure 2.10 Pulsation Violation Ratio in Single Stage Discharge System – As Found

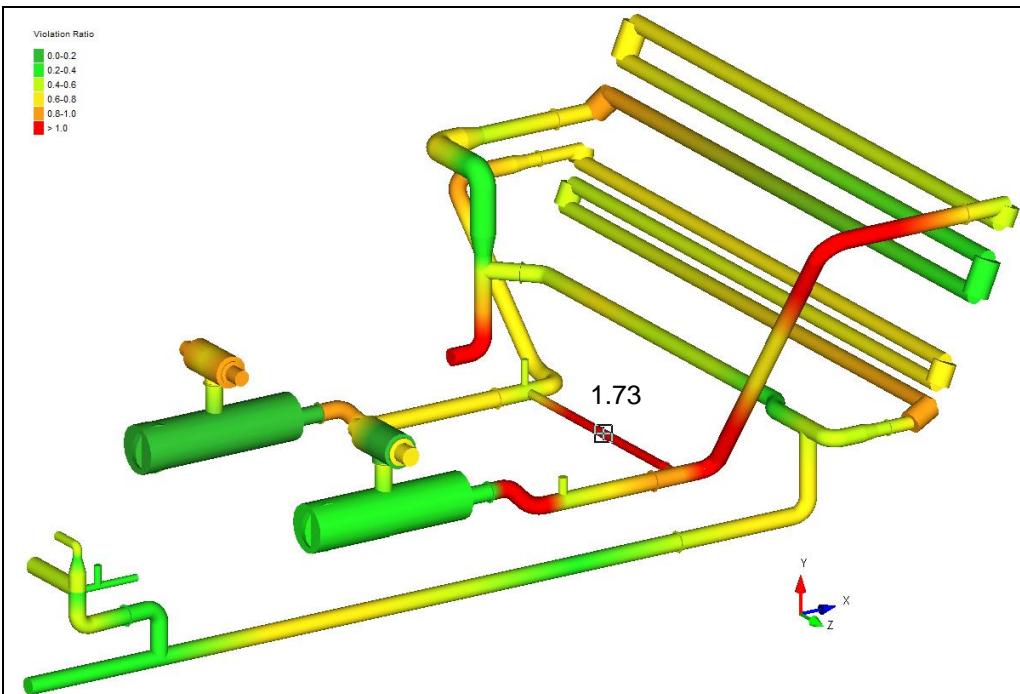


Figure 2.11 Pulsation Violation Ratio in Single Stage Discharge System – After Modifications

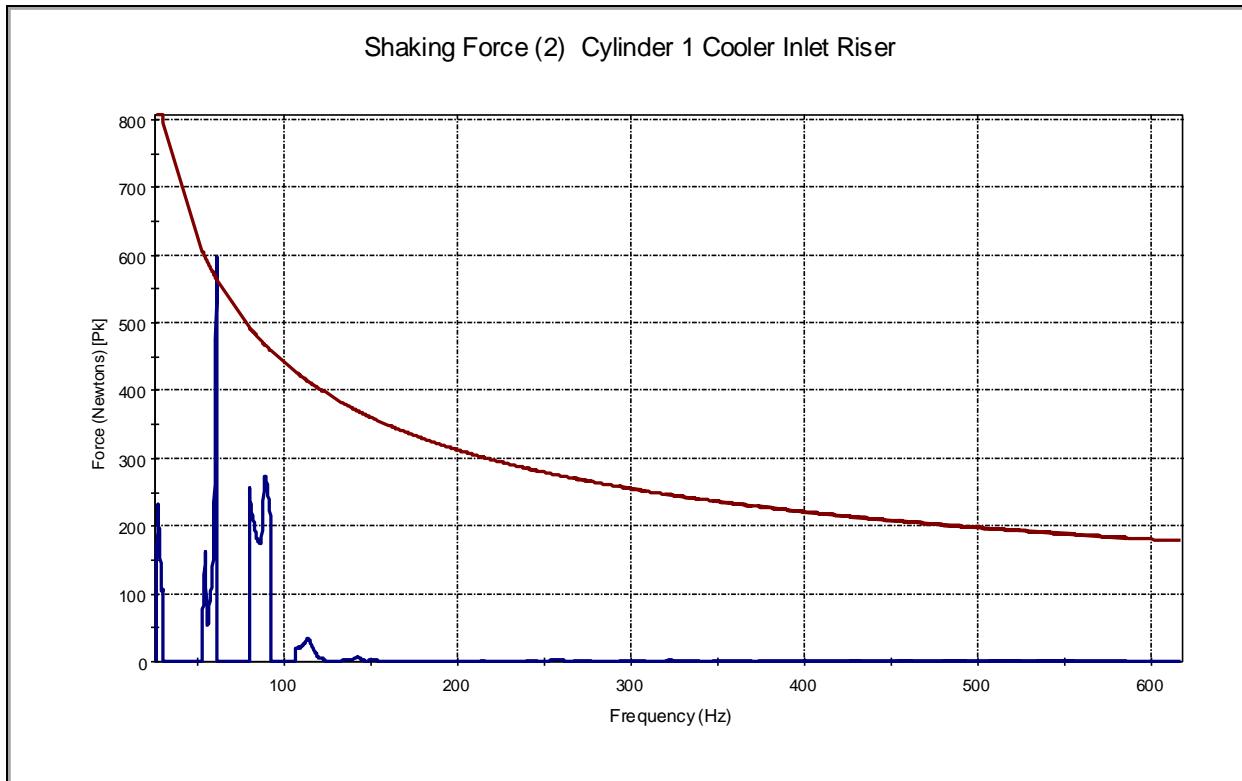


Figure 2.12 Shaking Force in Cylinder 1 Cooler Inlet Riser – As Found

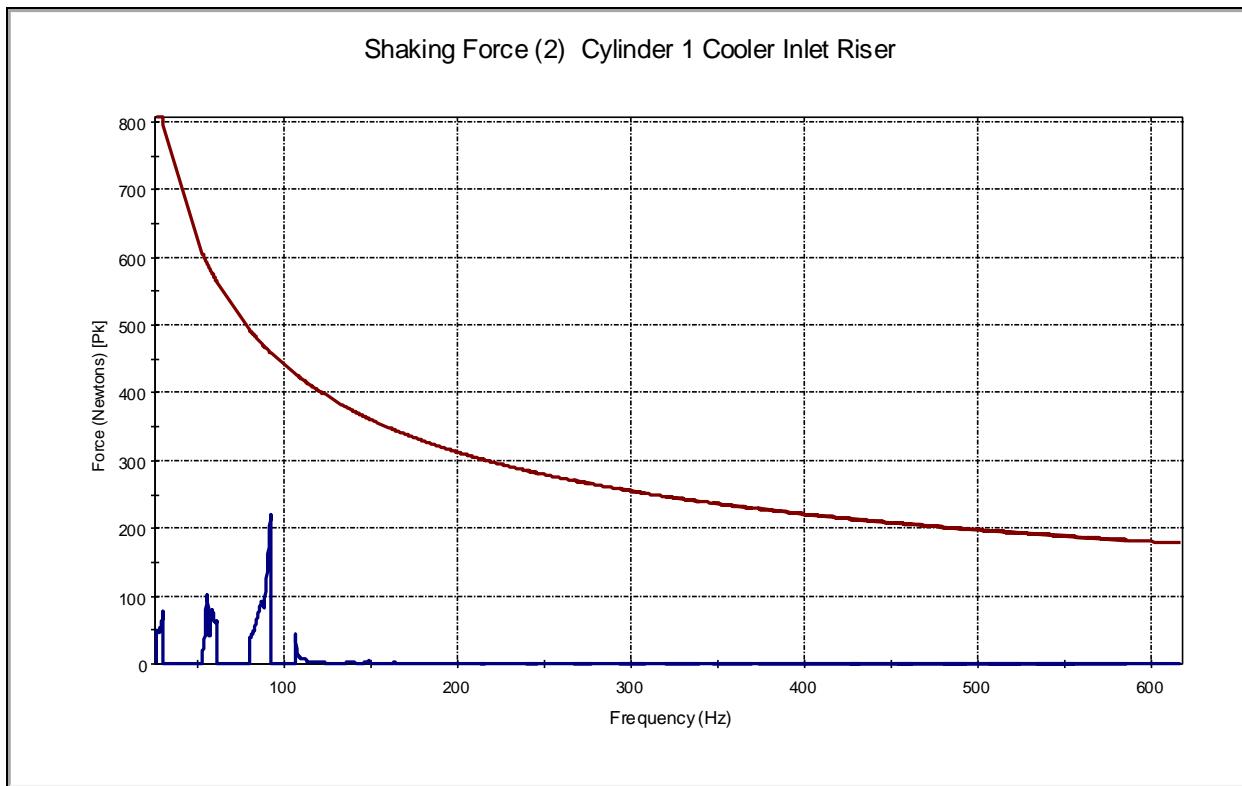


Figure 2.13 Shaking Force in Cylinder 1 Cooler Inlet Riser – After Modifications

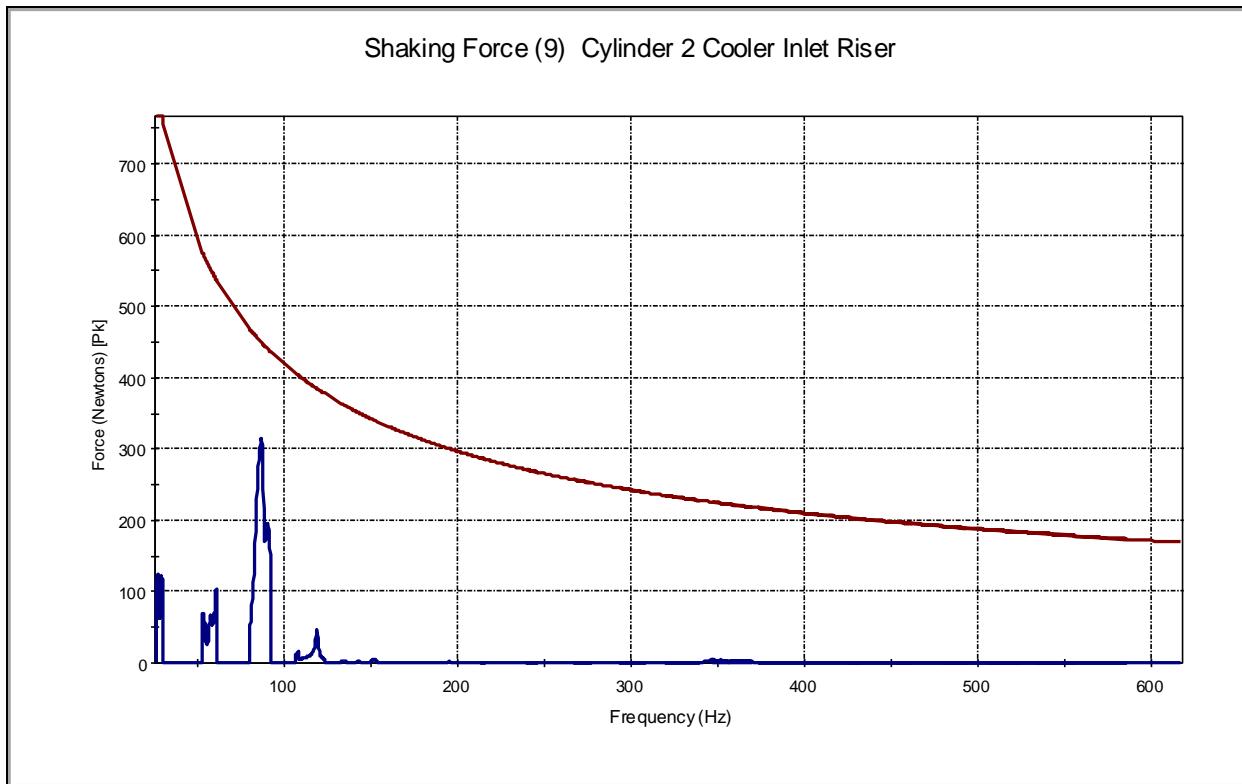


Figure 2.14 Shaking Force in Cylinder 2 Cooler Inlet Riser – As Found

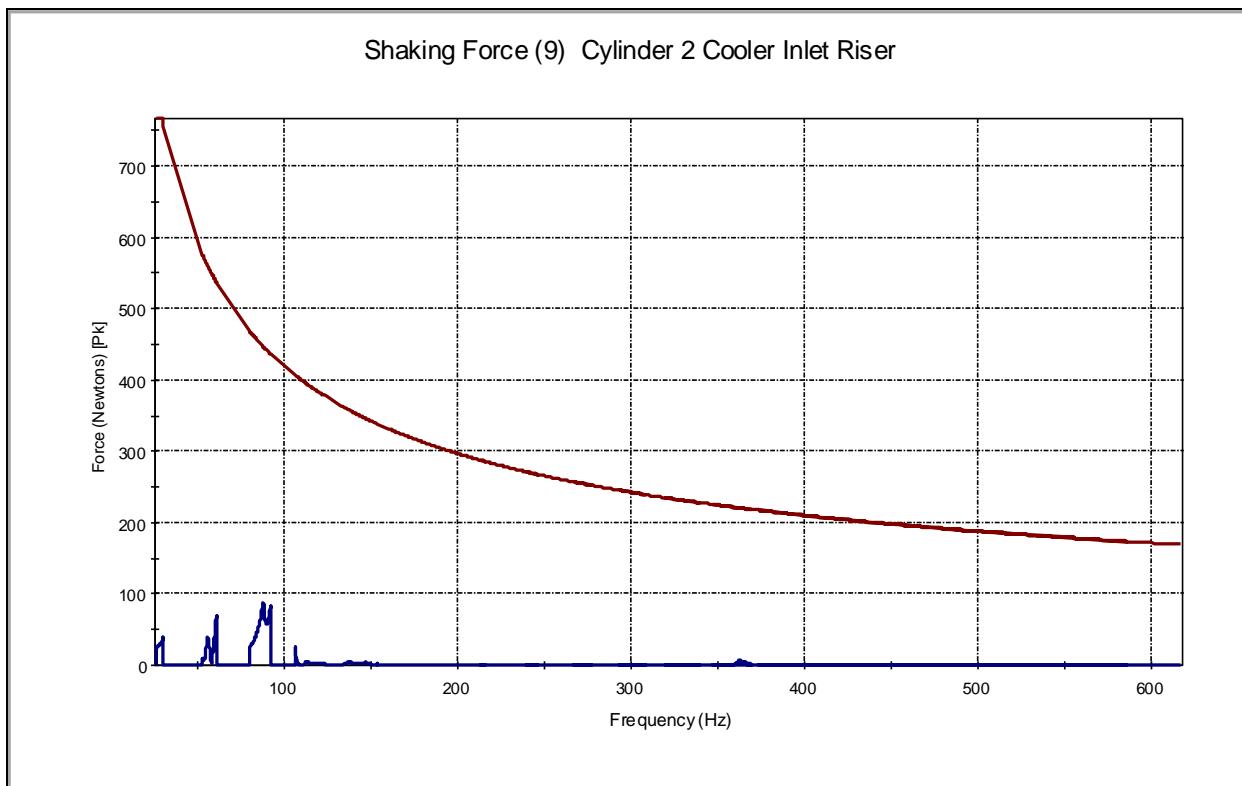


Figure 2.15 Shaking Force in Cylinder 2 Cooler Inlet Riser – After Modifications

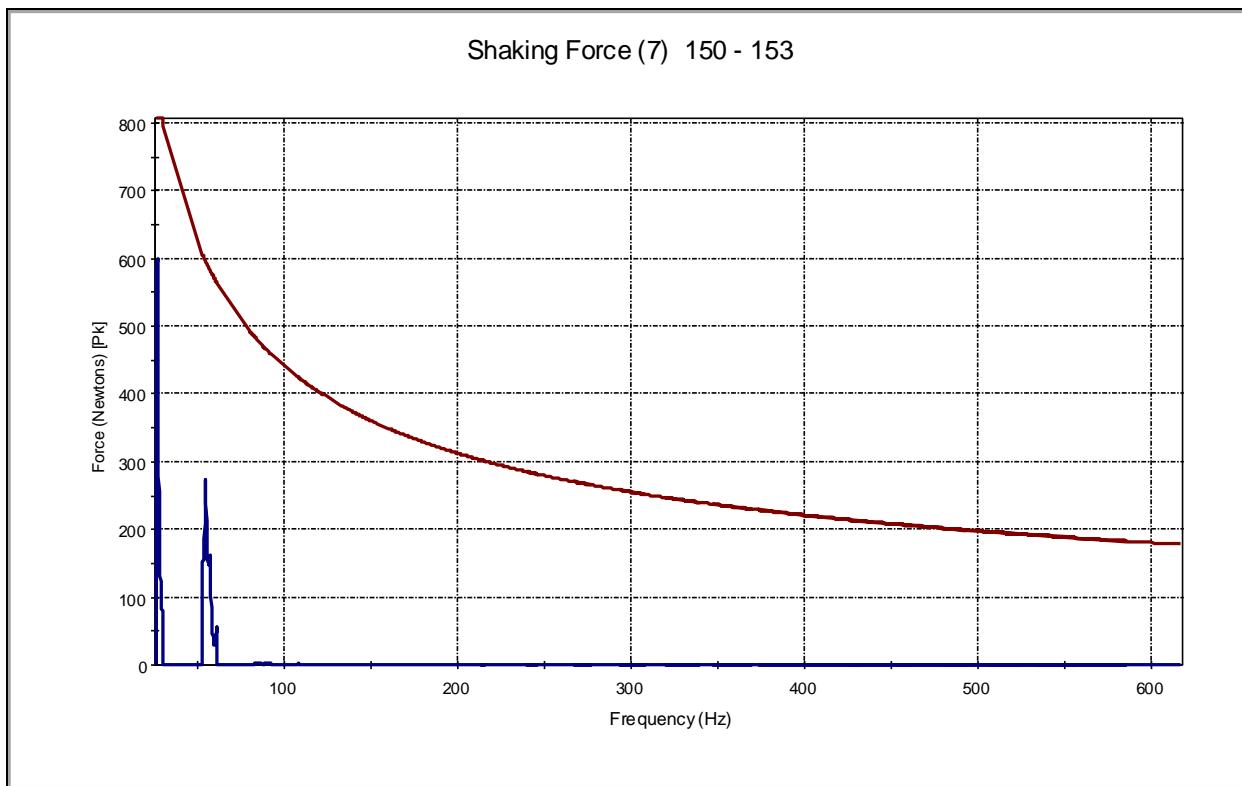


Figure 2.16 Shaking Force in Skid Level, Cylinder 2 Scrubber Feed Spool – As Found

Note: After modifications, the skid level cylinder 2 scrubber feed spool is completely isolated with blinds and has no shaking forces within the spool during 2 stage operation.

2.5 TWO STAGE SUCTION RESULTS

Pulsation levels in the two-stage suction system were 2.07x above API 618 4th edition guidelines for the worst as found case shown in Figure 2.17. After the installation of the recommended orifice plates pulsation levels are reduced to 1.48x guidelines shown in Figure 2.18. Selected shaking forces before and after the recommendations are shown in Figure 2.19 and Figure 2.20. Shaking forces are below guidelines before and after modifications.

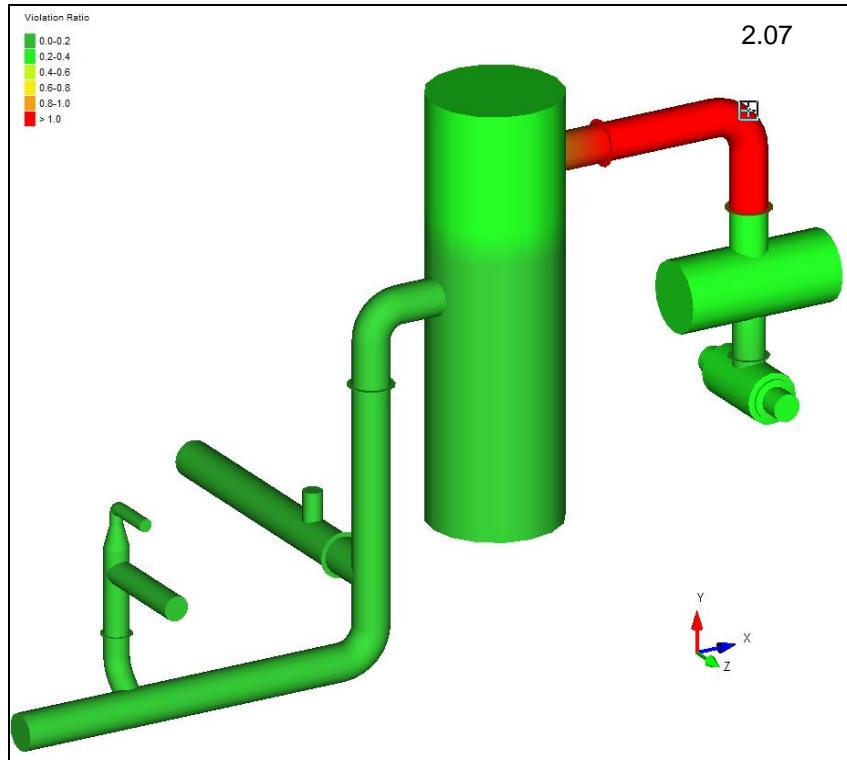


Figure 2.17 Pulsation Violation Ratio in Two Stage Suction System – As Found

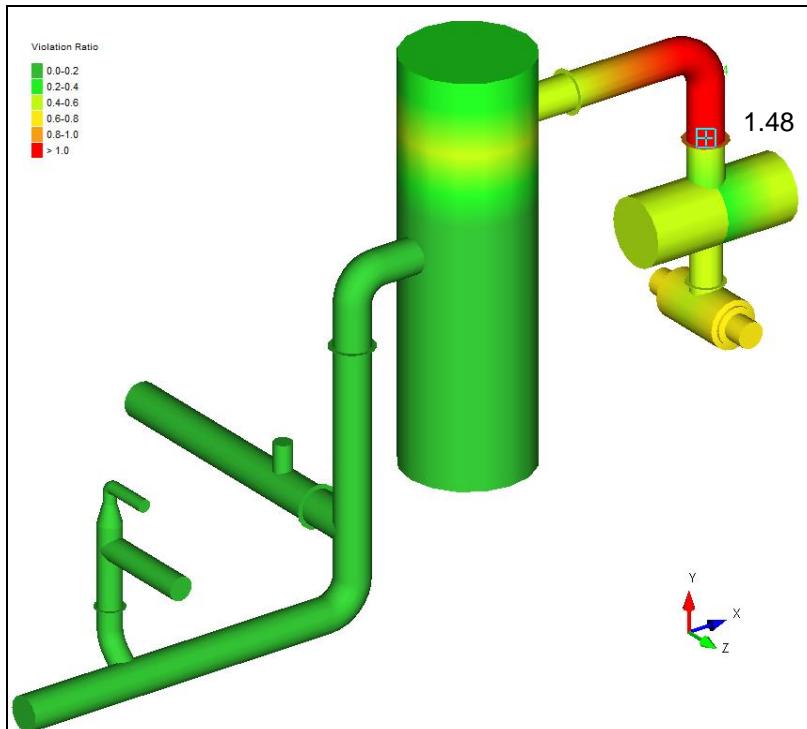


Figure 2.18 Pulsation Violation Ratio in Two Stage Suction System – After Modifications

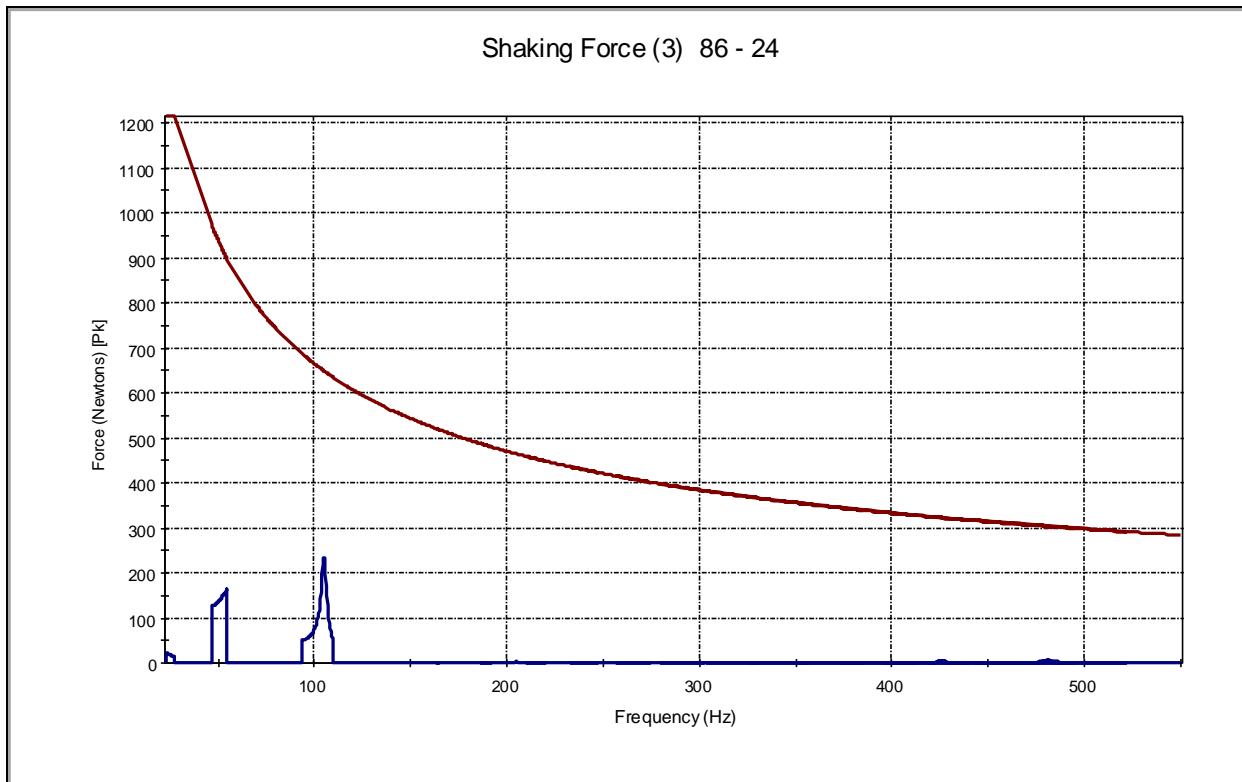


Figure 2.19 Shaking Force in Scrubber Outlet Spool – As Found

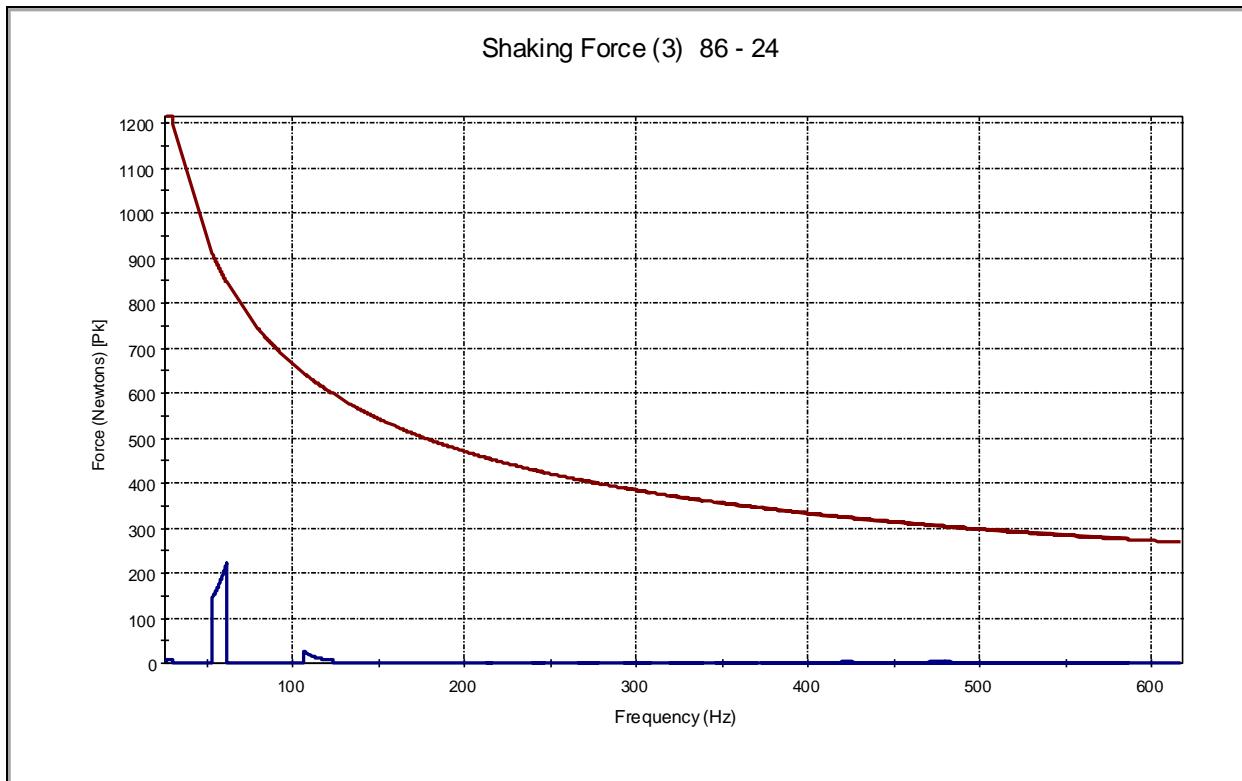


Figure 2.20 Shaking Force in Scrubber Outlet Spool – After Modifications

2.6 TWO STAGE INTERSTAGE RESULTS

Pulsation levels in the two-stage interstage system were 6.56x above API 618 4th edition guidelines for the worst as found case shown in Figure 2.21. After the implementation of the recommended changes pulsation levels are reduced to 0.98x guidelines shown in Figure 2.22. Selected shaking forces before and after the recommendations are shown in Figure 2.23 through Figure 2.26. Shaking forces were as high as 684N in the as found case; they are well below guidelines after modifications.

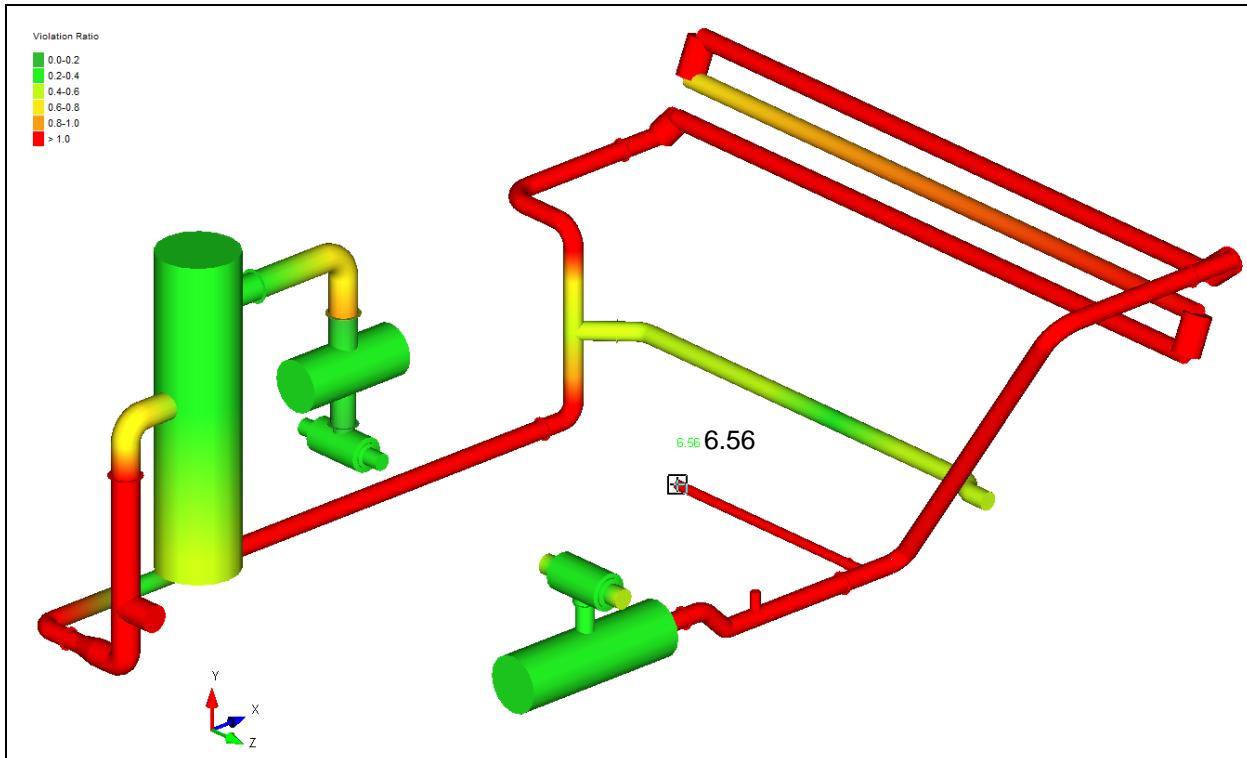


Figure 2.21 Pulsation Violation Ratio in Two Stage Interstage System – As Found

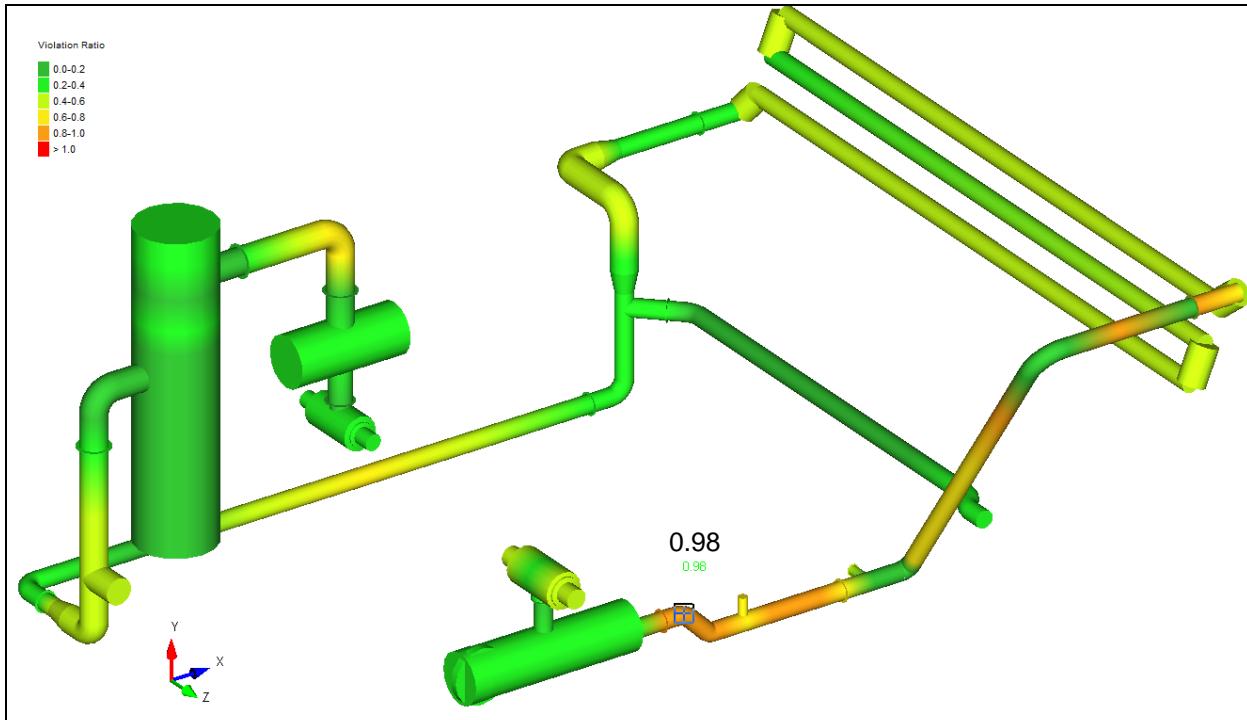


Figure 2.22 Pulsation Violation Ratio in Two Stage Interstage System – After Modifications

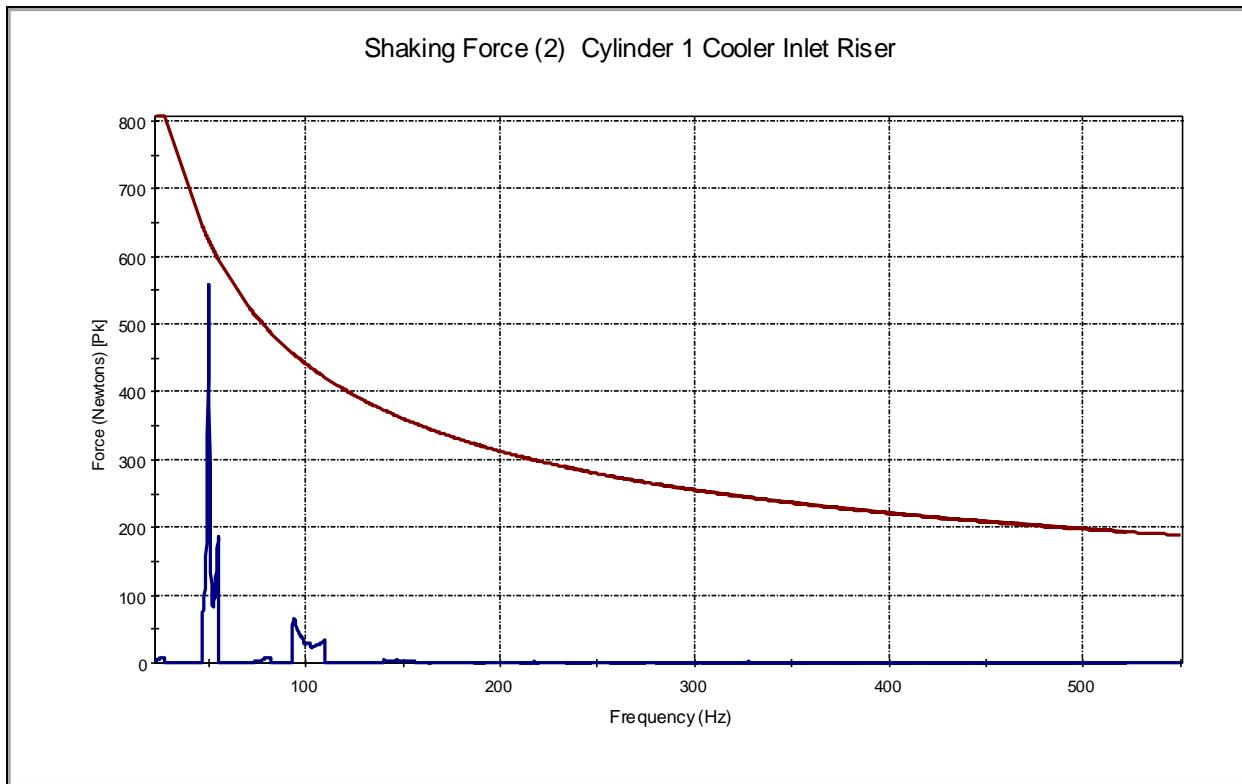


Figure 2.23 Shaking Force in Cylinder 1 Cooler Inlet Riser – As Found

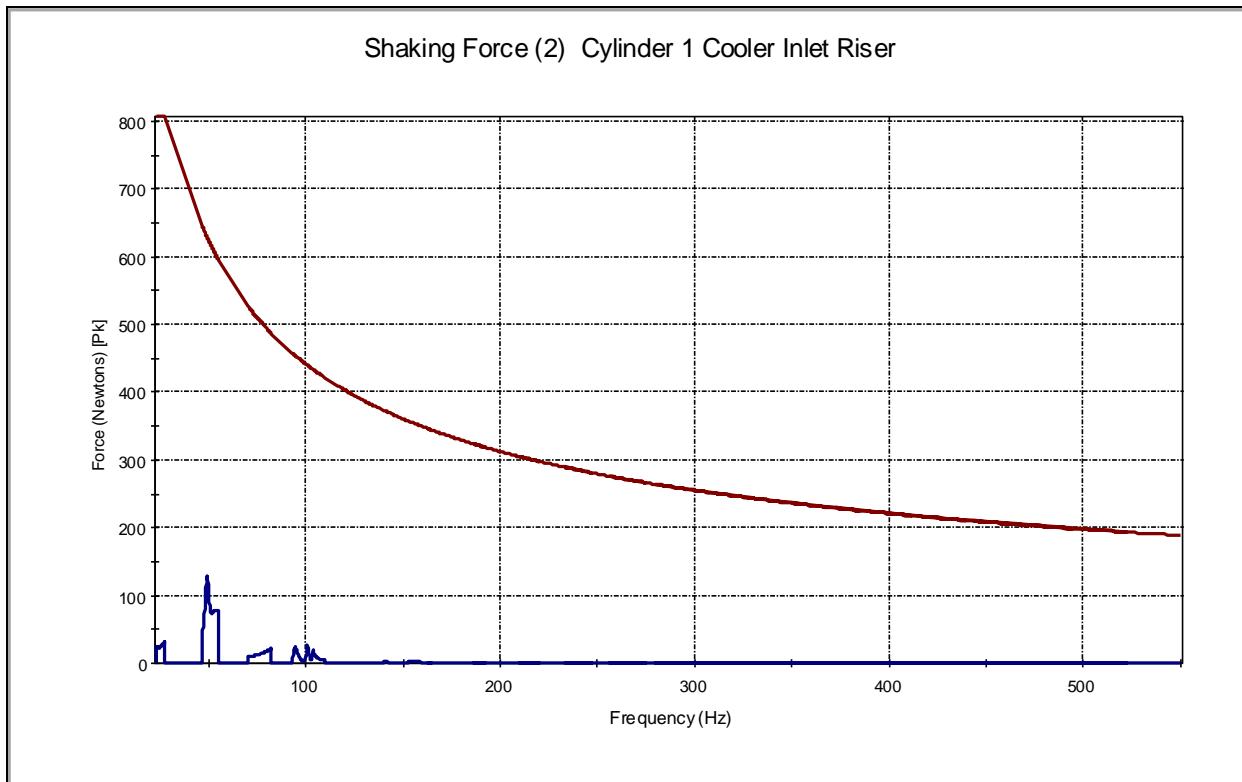


Figure 2.24 Shaking Force in Cylinder 1 Cooler Inlet Riser – After Modifications

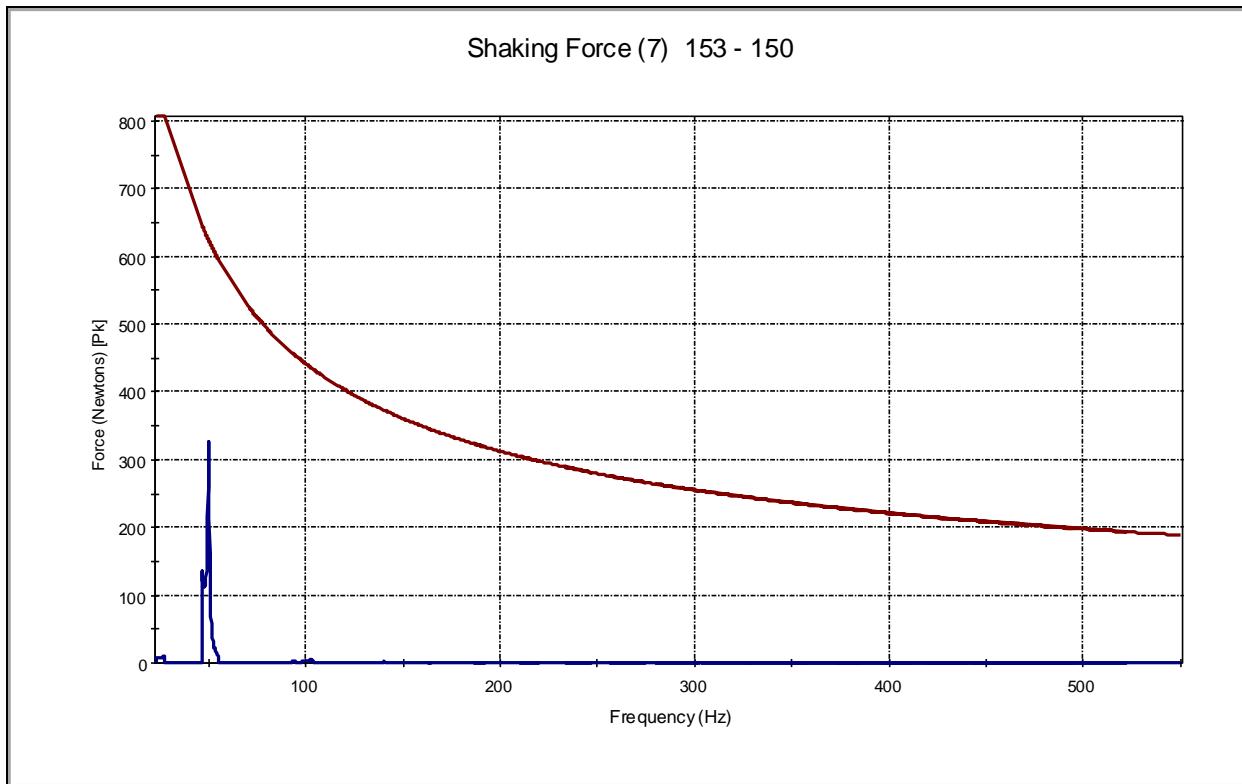


Figure 2.25 Shaking Force in Skid Level, Cylinder 2 Scrubber Feed Spool – As Found

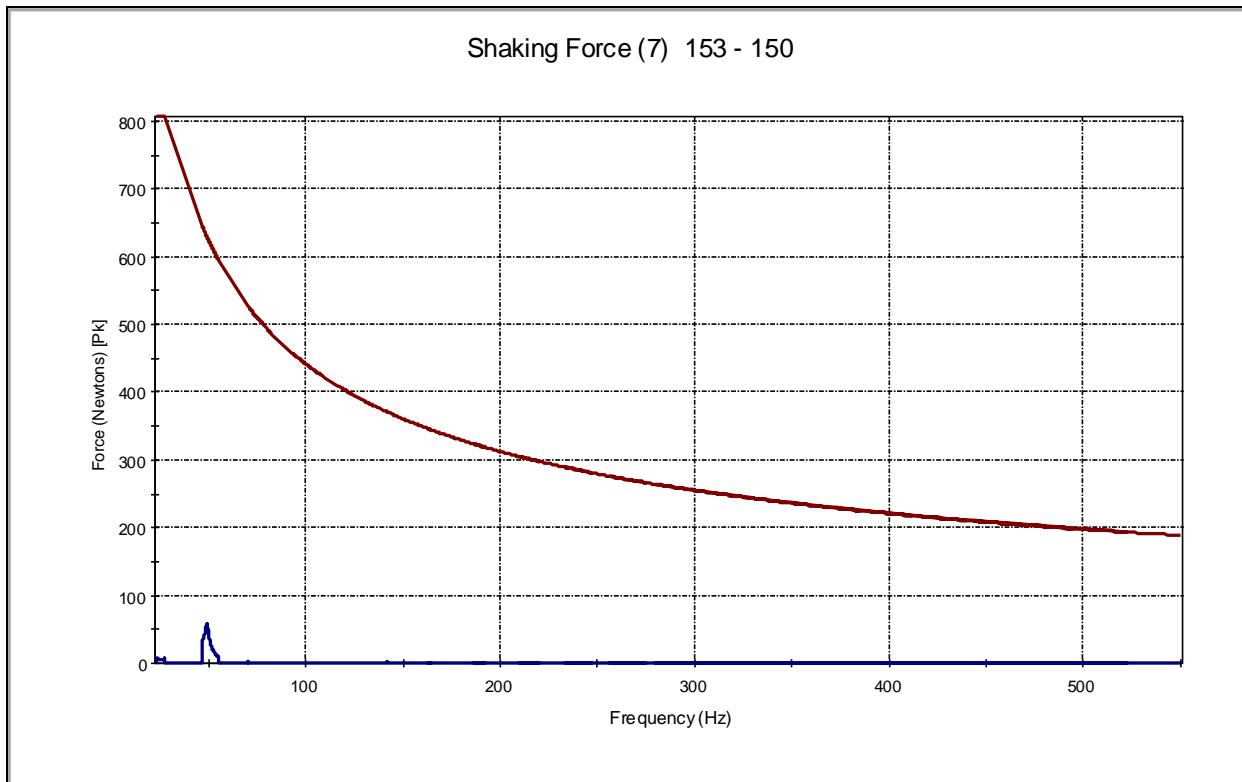


Figure 2.26 Shaking Force in Skid Level, Cylinder 2 Scrubber Feed Spool – After Modifications

2.7 TWO STAGE FINAL DISCHARGE RESULTS

Pulsation levels in the two stage final discharge system were 2.66x above API 618 4th edition guidelines for the worst as found case shown in Figure 2.27. After the implementation of the recommended changes pulsation levels are reduced to 1.54x guidelines shown in Figure 2.28. Selected shaking forces before and after the recommendations are shown in Figure 2.29 and Figure 2.30. Shaking forces are below guidelines before and after modifications.

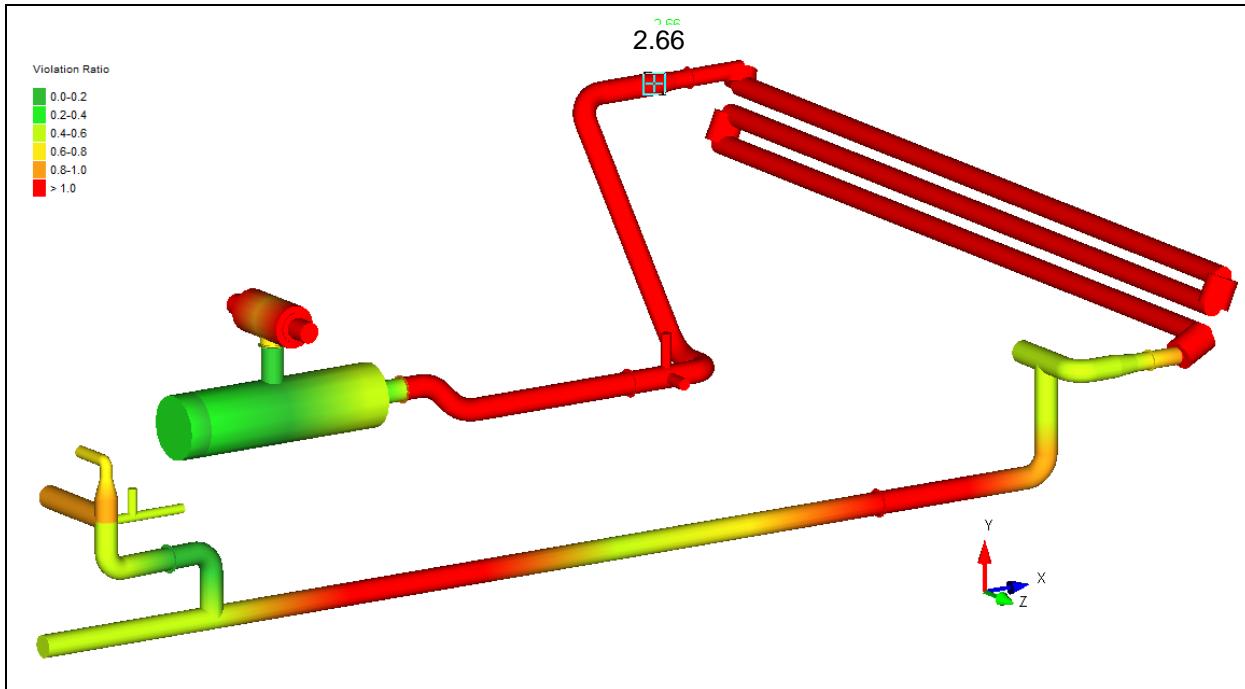


Figure 2.27 Pulsation Violation Ratio in Two Stage Discharge System – As Found

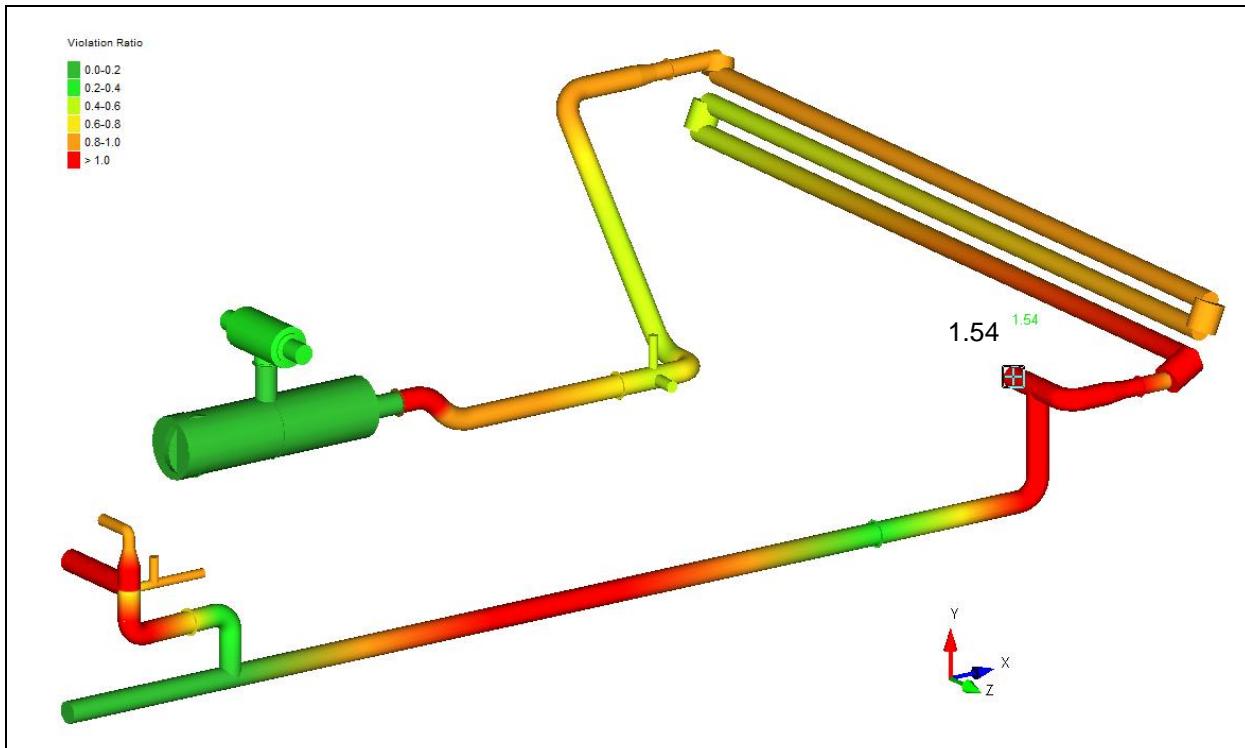
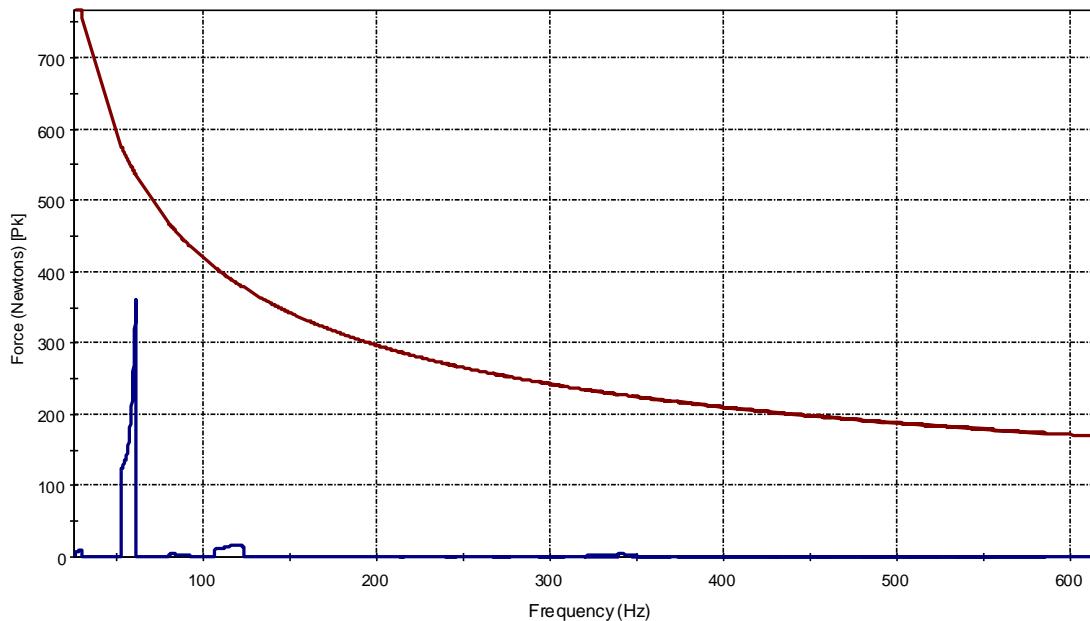
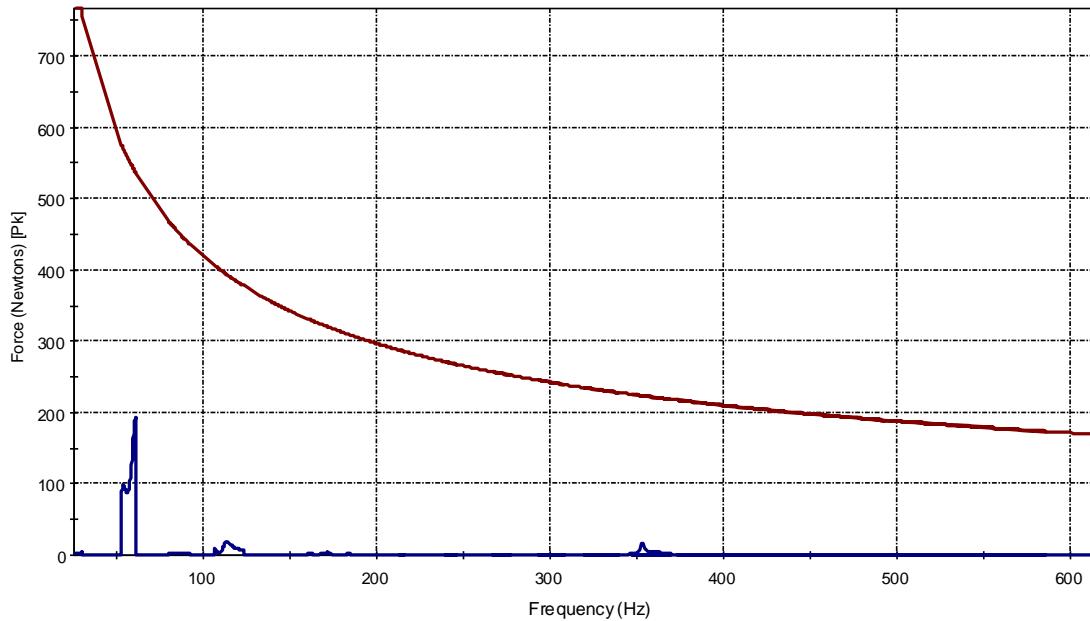


Figure 2.28 Pulsation Violation Ratio in Two Stage Discharge System – After Modifications

Shaking Force (2) Cylinder 2 Cooler Inlet Riser**Figure 2.29 Shaking Force in Cylinder 2 Cooler Inlet Riser – As Found****Shaking Force (2) Cylinder 2 Cooler Inlet Riser****Figure 2.30 Shaking Force in Cylinder 2 Cooler Inlet Riser – After Modifications**

2.8 PRESSURE DROPS AND HORSEPOWER LOSSES

The predicted pressure drops and horsepower losses, including the effects of cylinder orifice plates and bottle internals, are shown in the table below, together with the API 618 4th edition recommended maximum for the cases studied. In this respect, the design was dictated by the operating cases with the largest pressure drops. Where the pressure drops are different for cylinders sharing the same bottle (due, for example, to different clearance volumes or different paths through the bottle), the average is reported.

Table 2.2 Single Stage Pressure Drops and Power Losses

Case	System	Pressure Loss, kPa		Power Loss, kW	
		Calculated	Allowable	Calculated	Allowable
1	Single Stage Suction	6.41	13.7	0.67	1.34
1	Single Stage Discharge	8.97	22.23	0.72	1.62
1	Total	15.38	35.93	1.39	2.96
2	Single Stage Suction	3.74	14.44	0.35	1.24
2	Single Stage Discharge	4.55	28.88	0.28	1.58
2	Total	8.29	43.32	0.63	2.82
3	Single Stage Suction	5.9	17.71	0.58	1.61
3	Single Stage Discharge	7.6	32.45	0.52	2.01
3	Total	13.5	50.16	1.1	3.62
4	Single Stage Suction	4.01	17.79	0.36	1.46
4	Single Stage Discharge	4.71	37.44	0.27	1.89
4	Total	8.72	55.23	0.63	3.35
1b	Single Stage Suction	5.06	13.7	0.47	1.19
1b	Single Stage Discharge	7.1	22.23	0.51	1.44
1b	Total	12.16	35.93	0.98	2.63
2b	Single Stage Suction	2.95	14.44	0.24	1.1
2b	Single Stage Discharge	3.6	28.88	0.2	1.4
2b	Total	6.55	43.32	0.44	2.5
3b	Single Stage Suction	4.65	17.71	0.4	1.43
3b	Single Stage Discharge	6.01	32.45	0.37	1.79
3b	Total	10.66	50.16	0.77	3.22
4b	Single Stage Suction	3.17	17.79	0.25	1.3
4b	Single Stage Discharge	3.73	37.44	0.19	1.68
4b	Total	6.9	55.23	0.44	2.98

Table 2.3 Two Stage Pressure Drops and Power Losses, Cases 5-14

Case	System	Pressure Loss, kPa		Power Loss, kW	
		Calculated	Allowable	Calculated	Allowable
5	Stage 1 Suction	6.24	6.4	0.85	0.87
5	Stage 1 Discharge	11.26	9.56	1.21	1.03
5	Stage 2 Suction	6.64	5.47	0.61	0.5
5	Stage 2 Discharge	7.18	7.29	0.59	0.6
5	Total	31.32	28.72	3.26	3
6	Stage 1 Suction	5.08	5.62	0.68	0.75
6	Stage 1 Discharge	8.97	8.65	0.94	0.9
6	Stage 2 Suction	5.29	7.61	0.47	0.68
6	Stage 2 Discharge	5.08	11.53	0.36	0.81
6	Total	24.42	33.41	2.45	3.14
7	Stage 1 Suction	7.33	7.52	0.99	1.02
7	Stage 1 Discharge	13.2	11.25	1.42	1.21
7	Stage 2 Suction	7.8	6.82	0.71	0.62
7	Stage 2 Discharge	8.32	9.12	0.67	0.74
7	Total	36.65	34.71	3.79	3.59
8	Stage 1 Suction	5.01	5.9	0.67	0.79
8	Stage 1 Discharge	8.68	9.34	0.89	0.95
8	Stage 2 Suction	5.11	10.22	0.44	0.88
8	Stage 2 Discharge	4.4	17.62	0.27	1.09
8	Total	23.2	43.08	2.27	3.71
9	Stage 1 Suction	7.25	7.86	0.98	1.06
9	Stage 1 Discharge	12.87	12.01	1.36	1.27
9	Stage 2 Suction	7.61	10.27	0.68	0.92
9	Stage 2 Discharge	7.38	15.2	0.53	1.1
9	Total	35.11	45.34	3.55	4.35
10	Stage 1 Suction	4.96	6.16	0.66	0.82
10	Stage 1 Discharge	8.41	10.02	0.83	0.99
10	Stage 2 Suction	4.95	12.29	0.42	1.03
10	Stage 2 Discharge	3.88	23.74	0.21	1.3
10	Total	22.2	52.21	2.12	4.14
11	Stage 1 Suction	7.18	8.17	0.96	1.1
11	Stage 1 Discharge	12.55	12.75	1.3	1.32
11	Stage 2 Suction	7.42	13.05	0.65	1.14
11	Stage 2 Discharge	6.64	21.3	0.43	1.39
11	Total	33.79	55.27	3.34	4.95
12	Stage 1 Suction	4.89	6.42	0.65	0.85
12	Stage 1 Discharge	8.16	10.7	0.79	1.03
12	Stage 2 Suction	4.8	14	0.39	1.15
12	Stage 2 Discharge	3.47	29.88	0.17	1.49
12	Total	21.32	61	2	4.52
13	Stage 1 Suction	6.82	8.14	0.89	1.07
13	Stage 1 Discharge	11.93	12.68	1.21	1.29
13	Stage 2 Suction	7.05	14.92	0.6	1.28
13	Stage 2 Discharge	5.95	26.45	0.36	1.59
13	Total	31.75	62.19	3.06	5.23
14	Stage 1 Suction	5.42	7.06	0.72	0.93
14	Stage 1 Discharge	9.04	11.73	0.88	1.15
14	Stage 2 Suction	5.33	15.27	0.44	1.25
14	Stage 2 Discharge	3.89	32.29	0.19	1.61
14	Total	23.68	66.35	2.23	4.94

Table 2.4 Two Stage Pressure Drops and Power Losses, Cases 5b-14b

Case	System	Pressure Loss, kPa		Power Loss, kW	
		Calculated	Allowable	Calculated	Allowable
5b	Stage 1 Suction	4.92	6.4	0.59	0.77
5b	Stage 1 Discharge	8.9	9.56	0.85	0.91
5b	Stage 2 Suction	5.25	5.47	0.43	0.44
5b	Stage 2 Discharge	5.67	7.29	0.41	0.53
5b		24.74	28.72	2.28	2.65
6b	Stage 1 Suction	4.01	5.62	0.48	0.67
6b	Stage 1 Discharge	7.09	8.65	0.66	0.8
6b	Stage 2 Suction	4.17	7.61	0.33	0.6
6b	Stage 2 Discharge	4.01	11.53	0.25	0.72
6b		19.28	33.41	1.72	2.79
7b	Stage 1 Suction	5.78	7.52	0.7	0.91
7b	Stage 1 Discharge	10.44	11.25	1	1.07
7b	Stage 2 Suction	6.16	6.82	0.5	0.55
7b	Stage 2 Discharge	6.58	9.12	0.47	0.65
7b		28.96	34.71	2.67	3.18
8b	Stage 1 Suction	3.96	5.9	0.47	0.7
8b	Stage 1 Discharge	6.86	9.34	0.62	0.85
8b	Stage 2 Suction	4.04	10.22	0.31	0.79
8b	Stage 2 Discharge	3.48	17.62	0.19	0.97
8b		18.34	43.08	1.59	3.31
9b	Stage 1 Suction	5.72	7.86	0.69	0.94
9b	Stage 1 Discharge	10.17	12.01	0.95	1.12
9b	Stage 2 Suction	6	10.27	0.48	0.82
9b	Stage 2 Discharge	5.83	15.2	0.38	0.98
9b		27.72	45.34	2.5	3.86
10b	Stage 1 Suction	3.91	6.16	0.46	0.73
10b	Stage 1 Discharge	6.65	10.02	0.58	0.88
10b	Stage 2 Suction	3.91	12.29	0.29	0.92
10b	Stage 2 Discharge	3.07	23.74	0.15	1.15
10b		17.54	52.21	1.48	3.68
11b	Stage 1 Suction	5.67	8.17	0.68	0.97
11b	Stage 1 Discharge	9.92	12.75	0.91	1.17
11b	Stage 2 Suction	5.86	13.05	0.46	1.01
11b	Stage 2 Discharge	5.25	21.3	0.31	1.24
11b		26.7	55.27	2.36	4.39
12b	Stage 1 Suction	3.86	6.42	0.45	0.75
12b	Stage 1 Discharge	6.45	10.7	0.55	0.92
12b	Stage 2 Suction	3.79	14	0.28	1.02
12b	Stage 2 Discharge	2.75	29.88	0.12	1.32
12b		16.85	61	1.4	4.01
13b	Stage 1 Suction	5.38	8.14	0.63	0.95
13b	Stage 1 Discharge	9.43	12.68	0.85	1.14
13b	Stage 2 Suction	5.56	14.92	0.42	1.14
13b	Stage 2 Discharge	4.7	26.45	0.25	1.41
13b		25.07	62.19	2.15	4.64
14b	Stage 1 Suction	4.28	7.06	0.5	0.83
14b	Stage 1 Discharge	7.15	11.73	0.62	1.02
14b	Stage 2 Suction	4.21	15.27	0.31	1.11
14b	Stage 2 Discharge	3.07	32.29	0.14	1.43
14b		18.71	66.35	1.57	4.39

3. MECHANICAL STUDY DETAILED RESULTS

3.1 MECHANICAL MODEL SETUP

As required by API 618 4th edition M4 and M5, the natural frequencies of the pipe runs, cylinders, and manifolds were calculated and compared to the characteristic frequency of excitation corresponding to the running frequency of the compressor and engine. As specified by the client, a maximum running speed of 1800 RPM (corresponding to 30 Hz) was used in the study. The mechanical model of the pipeline and the supporting structure included in this analysis is shown in Figure 3.1.

3.2 MECHANICAL STUDY RESULTS

A mechanical system model that comprises piping, scrubber vessels, suction and discharge bottles and compressor cylinders, was created according to drawings and pictures provided by the client. The analysis was completed using AutoPIPE V8i, a piping stress analysis package that is widely accepted in the industry.

In determining locations and types of support used on pipes attention was paid to avoid multiples of operating speed.

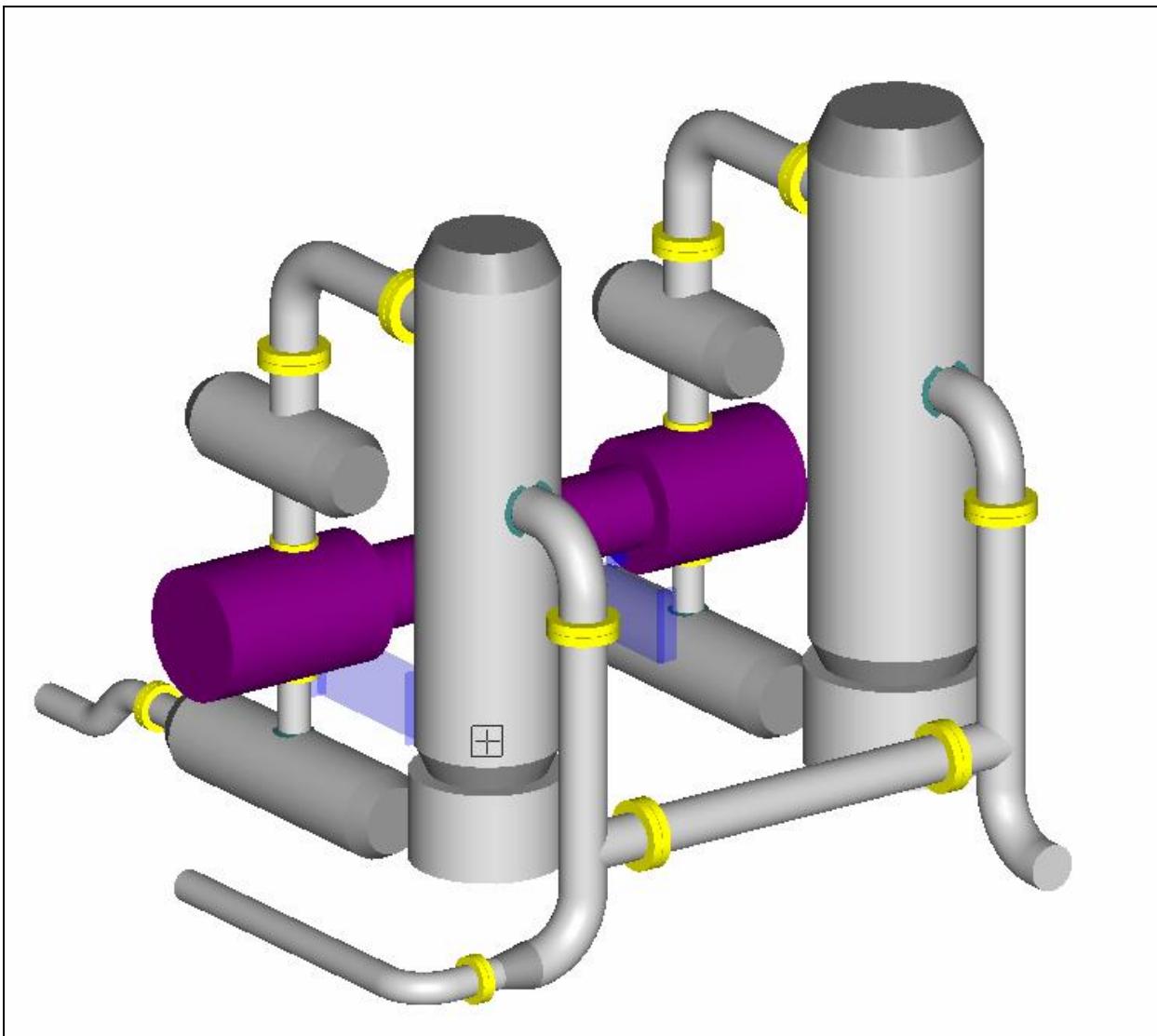


Figure 3.1 Mechanical Model General View

3.3 MECHANICAL MODEL MODAL ANALYSIS RESULTS

The natural frequencies and mode shape descriptions are given in Table 3.1 and the mode shapes of the modified piping are shown in Figure 3.2 through Figure 3.11.

Table 3.1 Natural Frequencies After Recommendations

Mode Number	Frequency	Running Speed Multiple	Location
1	77.3 Hz	2.58	Cylinder 2 Suction Bottle, Scrubber & Scrubber Inlet Piping
2	81.4 Hz	2.71	Cylinder 2 Suction Bottle, Scrubber & Scrubber Inlet Piping
3	84.2 Hz	2.8	Cylinder 1 Suction Bottle, Scrubber & Scrubber Inlet Piping
4	86.5 Hz	2.88	Cylinder 1 Suction Bottle, Scrubber & Scrubber Inlet Piping
5	93.6 Hz	3.12	Cylinders 1 & 2 Scrubber Inlet Piping
6	100.1 Hz	3.34	Cylinder 2 Suction Bottle
7	103 Hz	3.43	Cylinder 1 Suction Bottle
8	118.5 Hz	3.95	Cylinders 1 & 2 Scrubber Inlet Piping
9	126.3 Hz	4.21	Cylinders 1 & 2 Scrubber Inlet Piping
10	149 Hz	4.97	Cylinder 2 Scrubber Inlet Piping

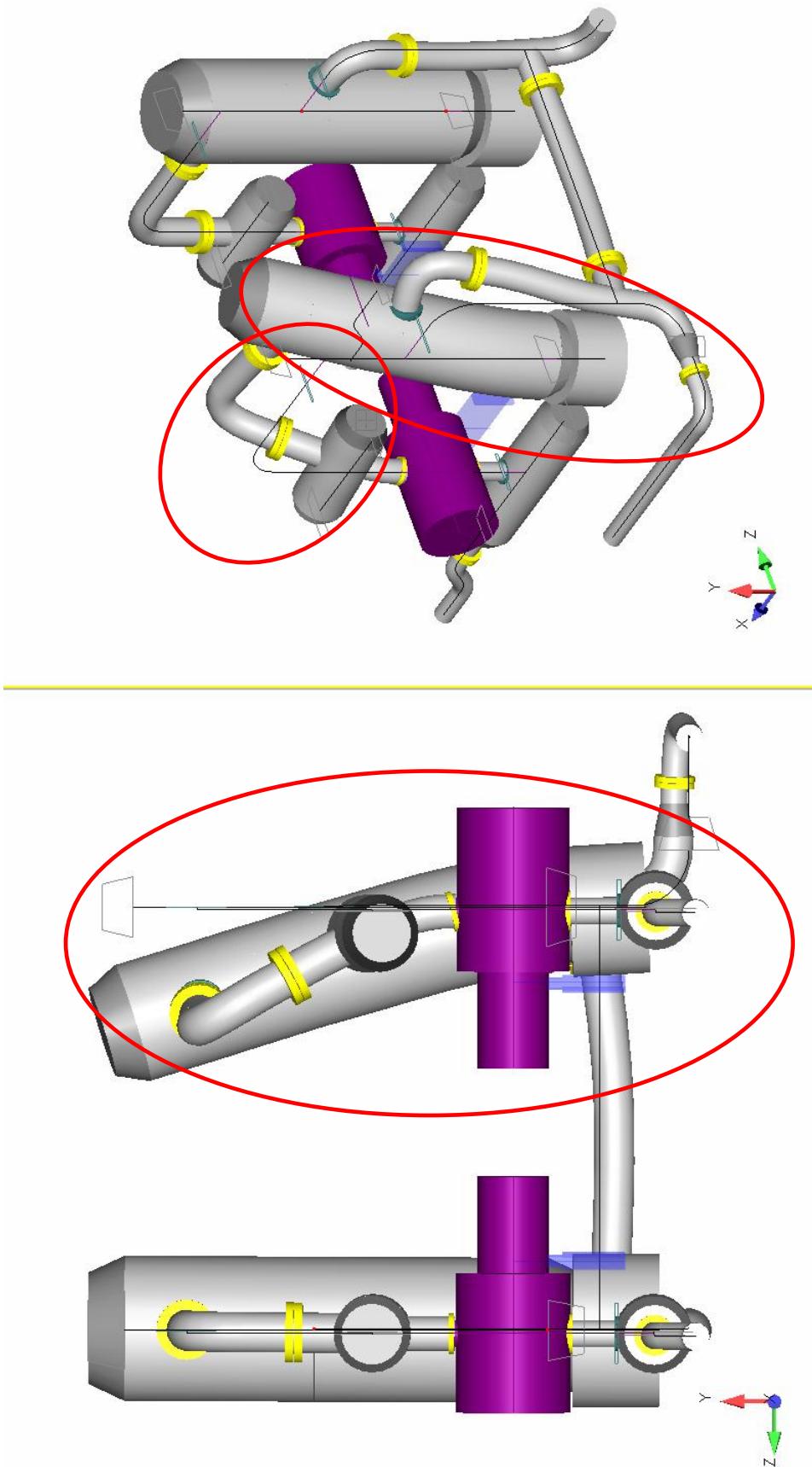


Figure 3.2 Mode Shape 1 – 77.3 Hz – Modified System

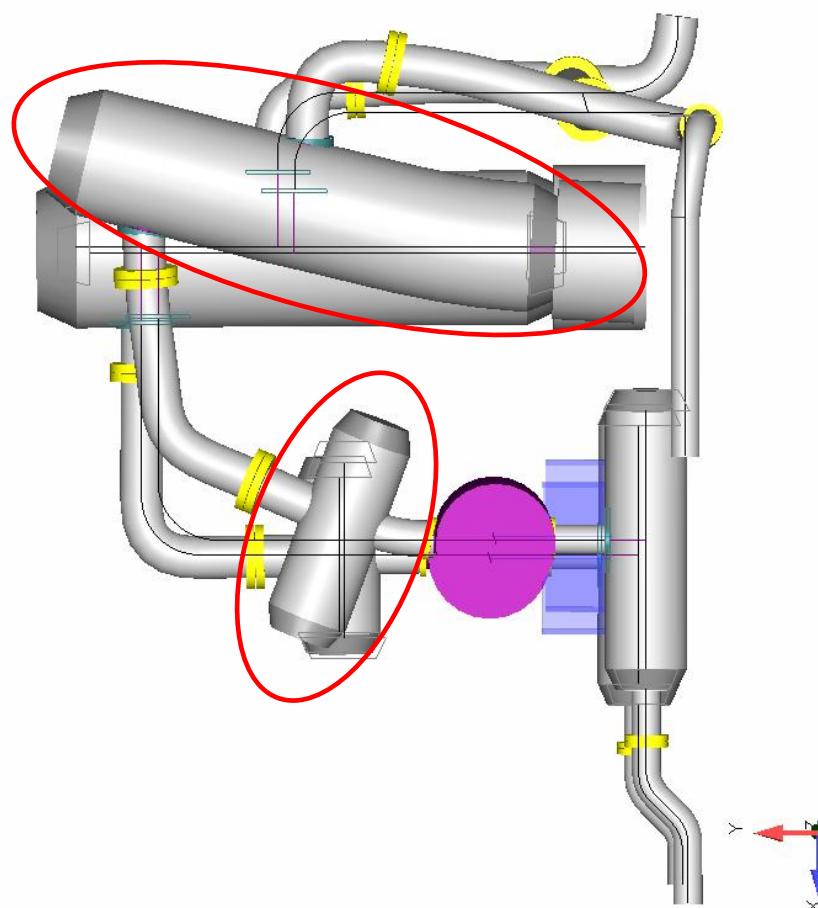
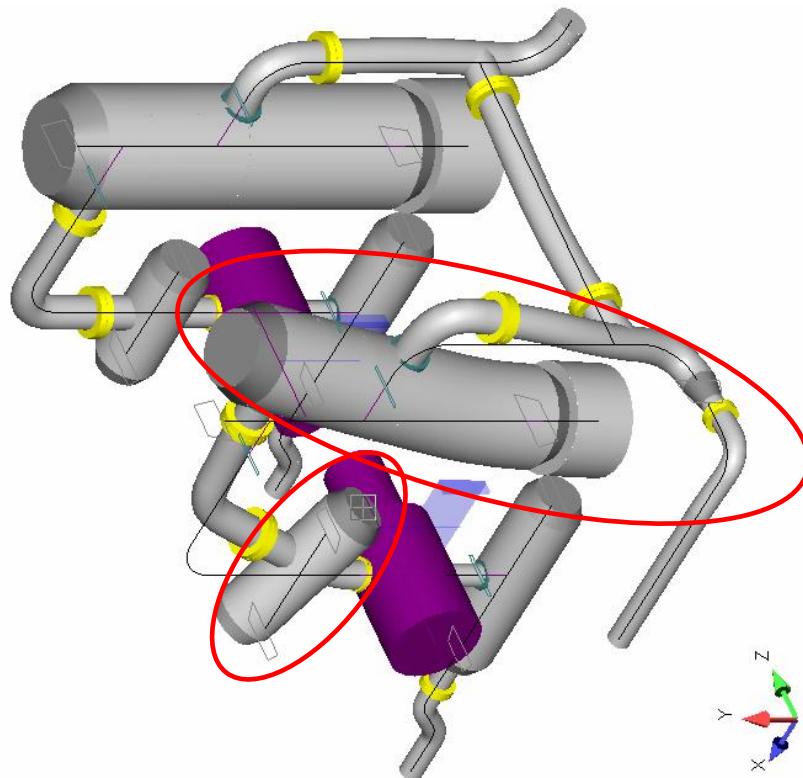


Figure 3.3 Mode Shape 2 – 81.4 Hz – Modified System

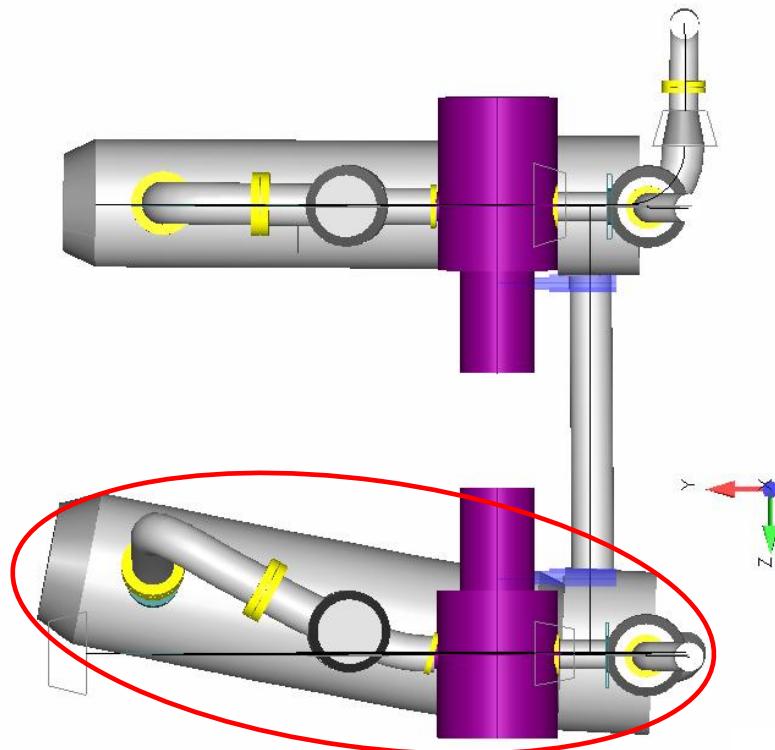
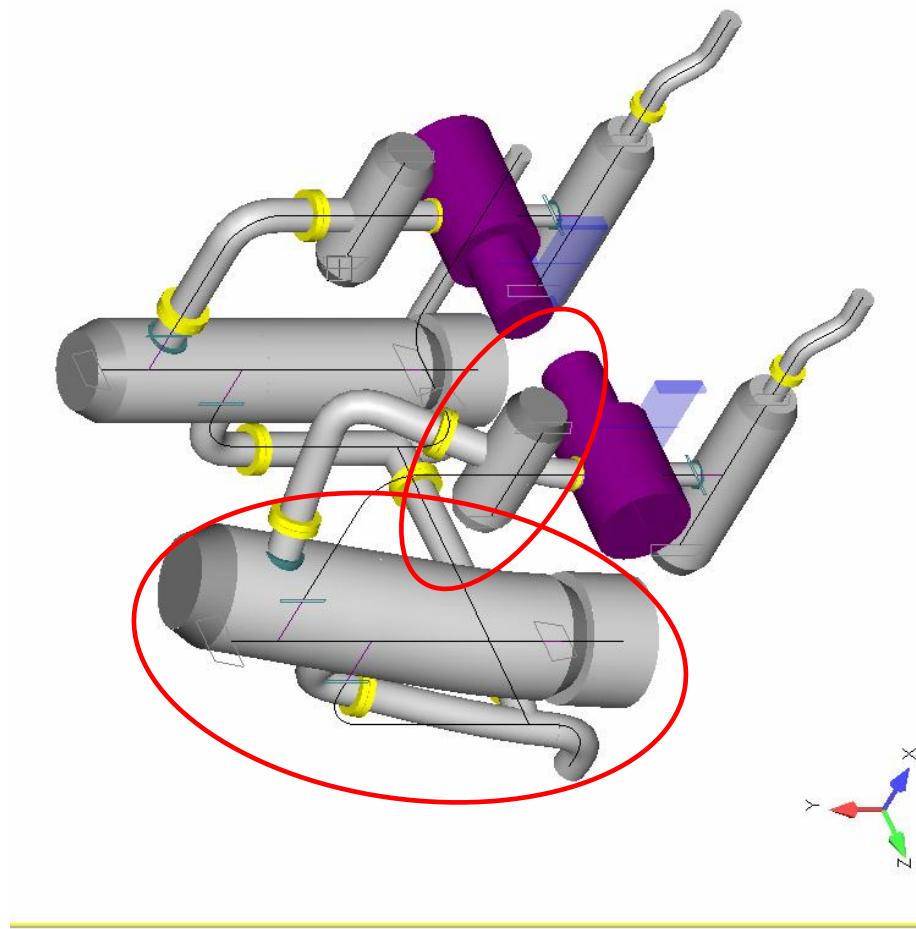


Figure 3.4 Mode Shape 3 – 84.2 Hz – Modified System

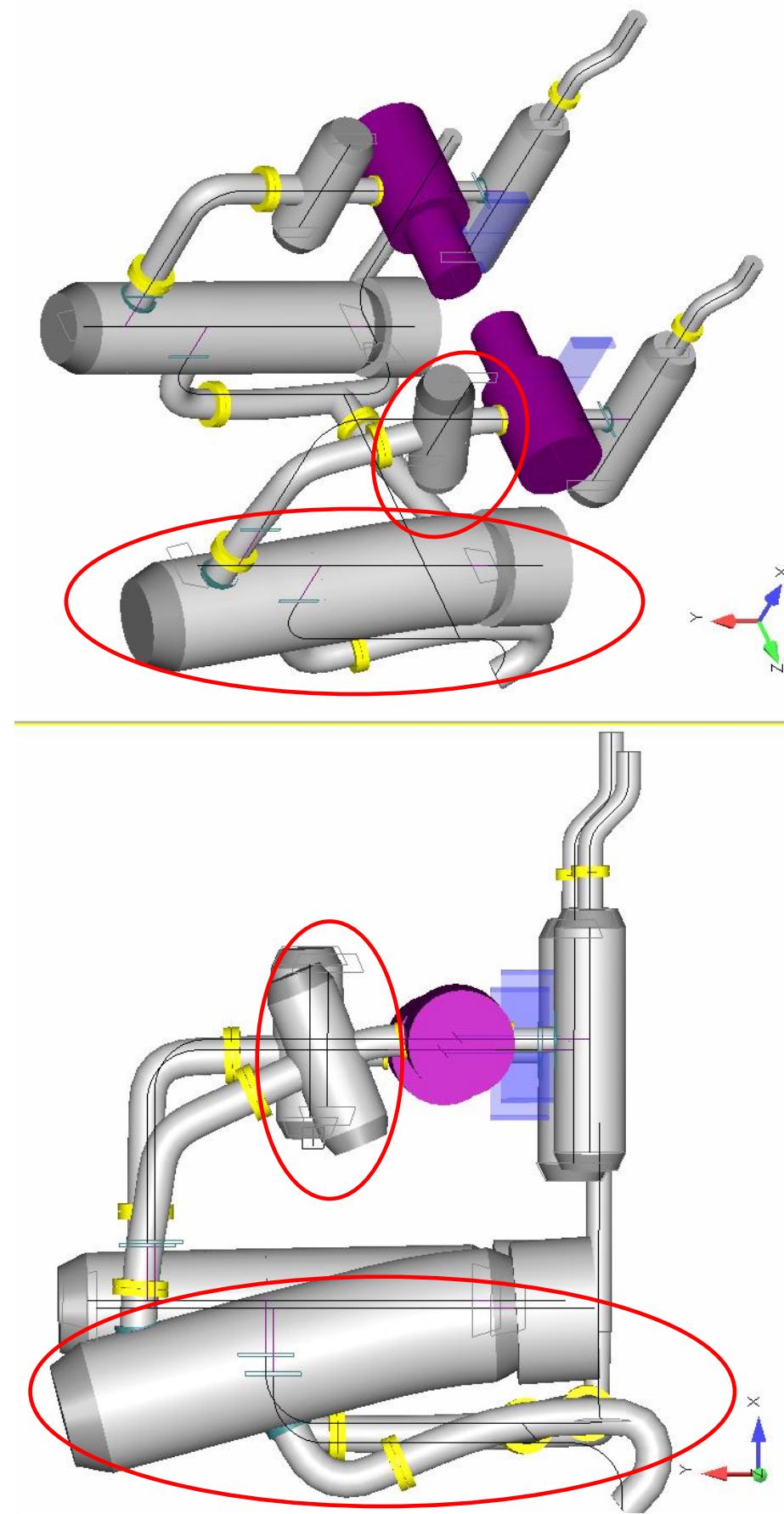


Figure 3.5 Mode Shape 4 – 86.5 Hz – Modified System

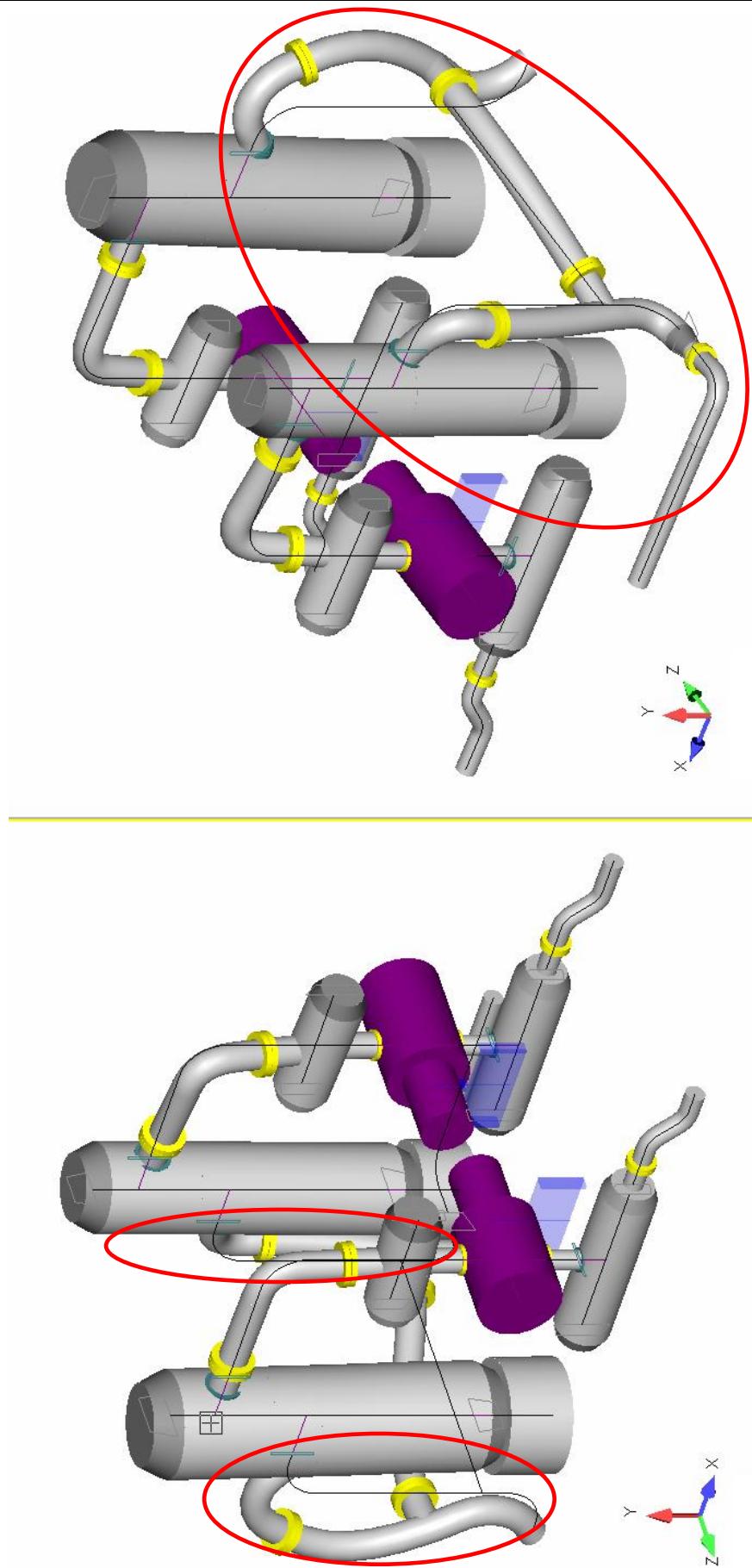


Figure 3.6 Mode Shape 5 – 93.6 Hz – Modified System

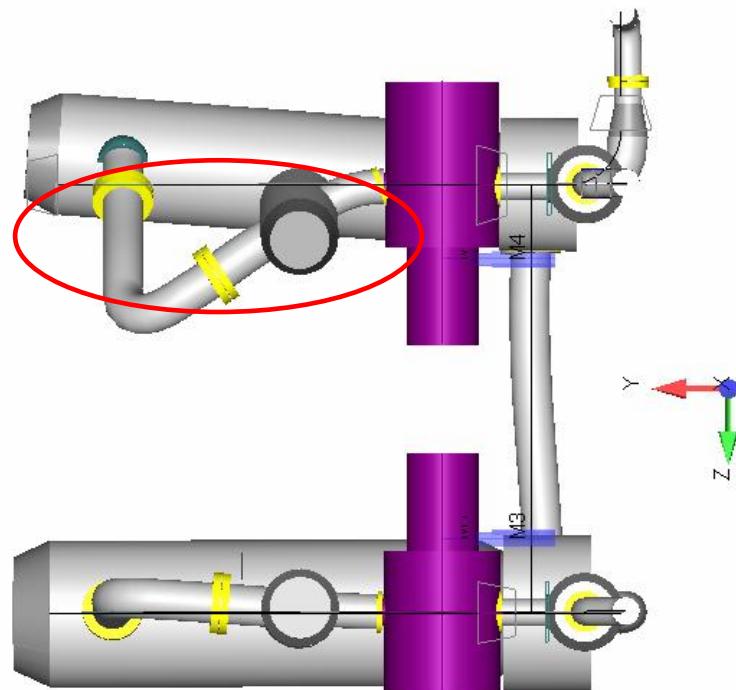
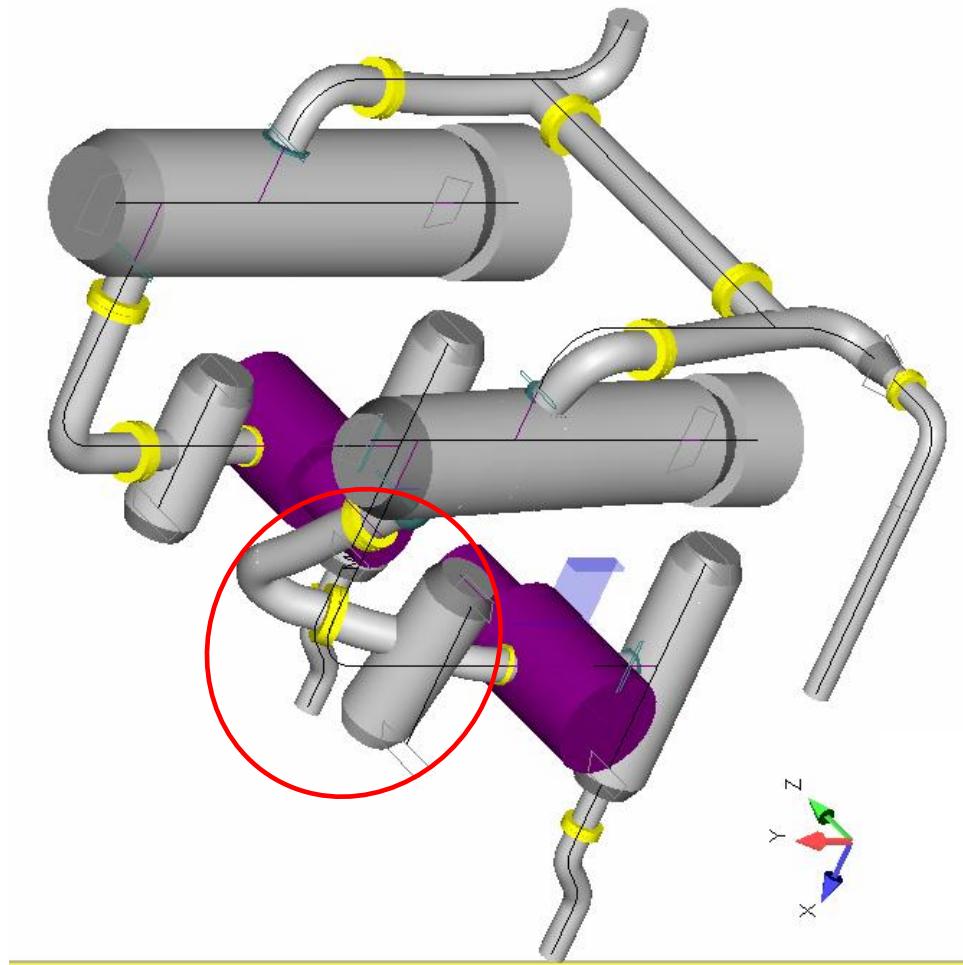


Figure 3.7 Mode Shape 6 – 100.1 Hz – Modified System

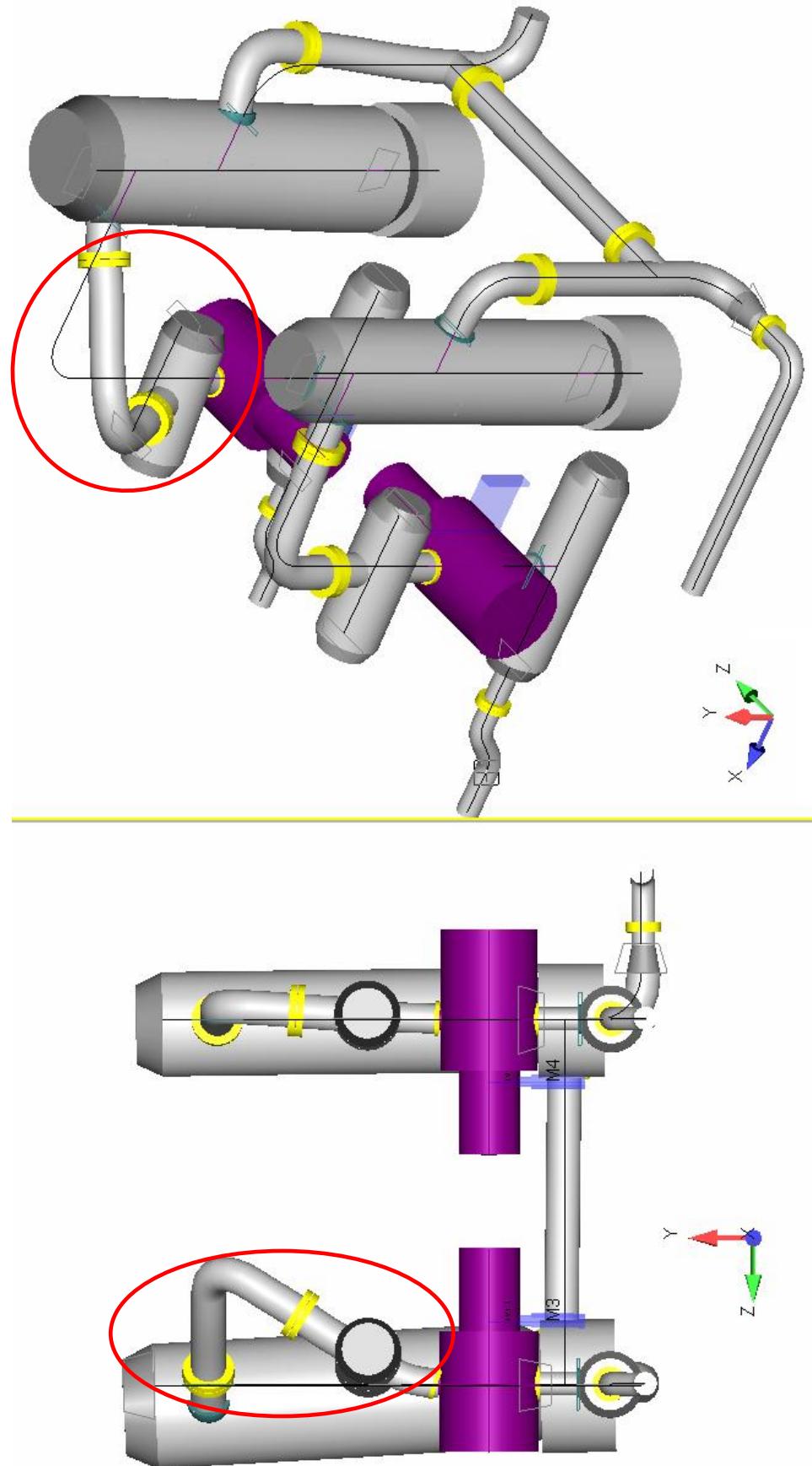


Figure 3.8 Mode Shape 7 – 103 Hz – Modified System

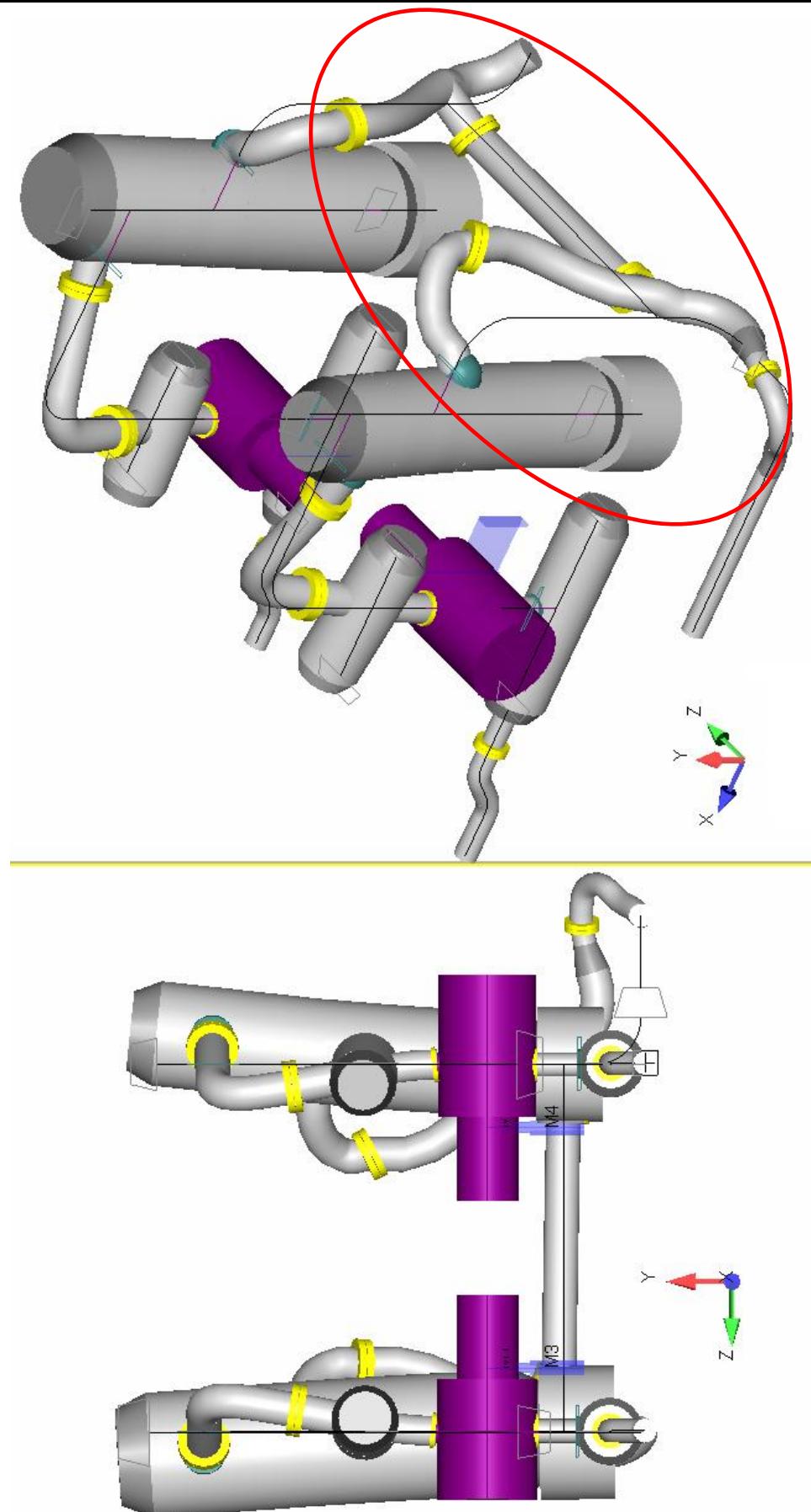


Figure 3.9 Mode Shape 8 – 118.5 Hz – Modified System

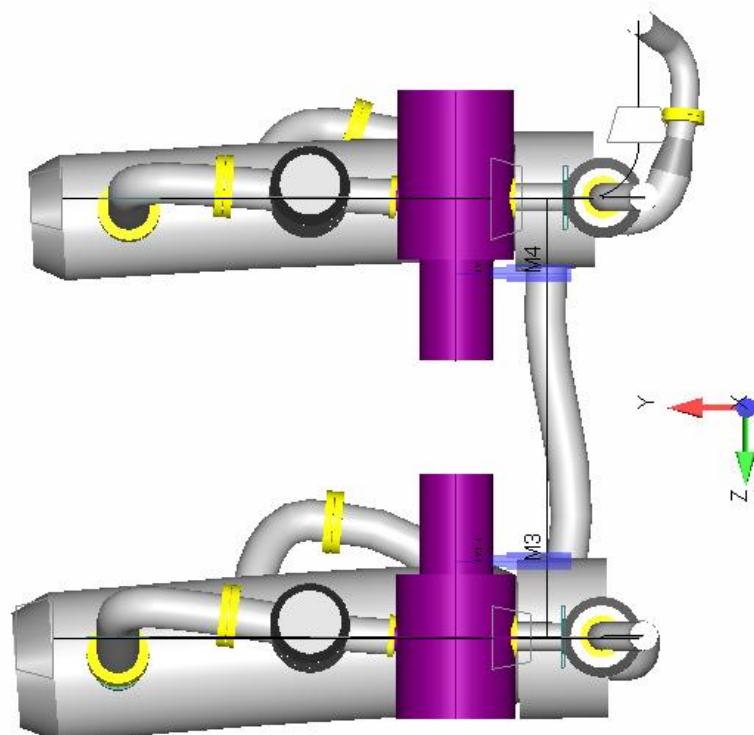
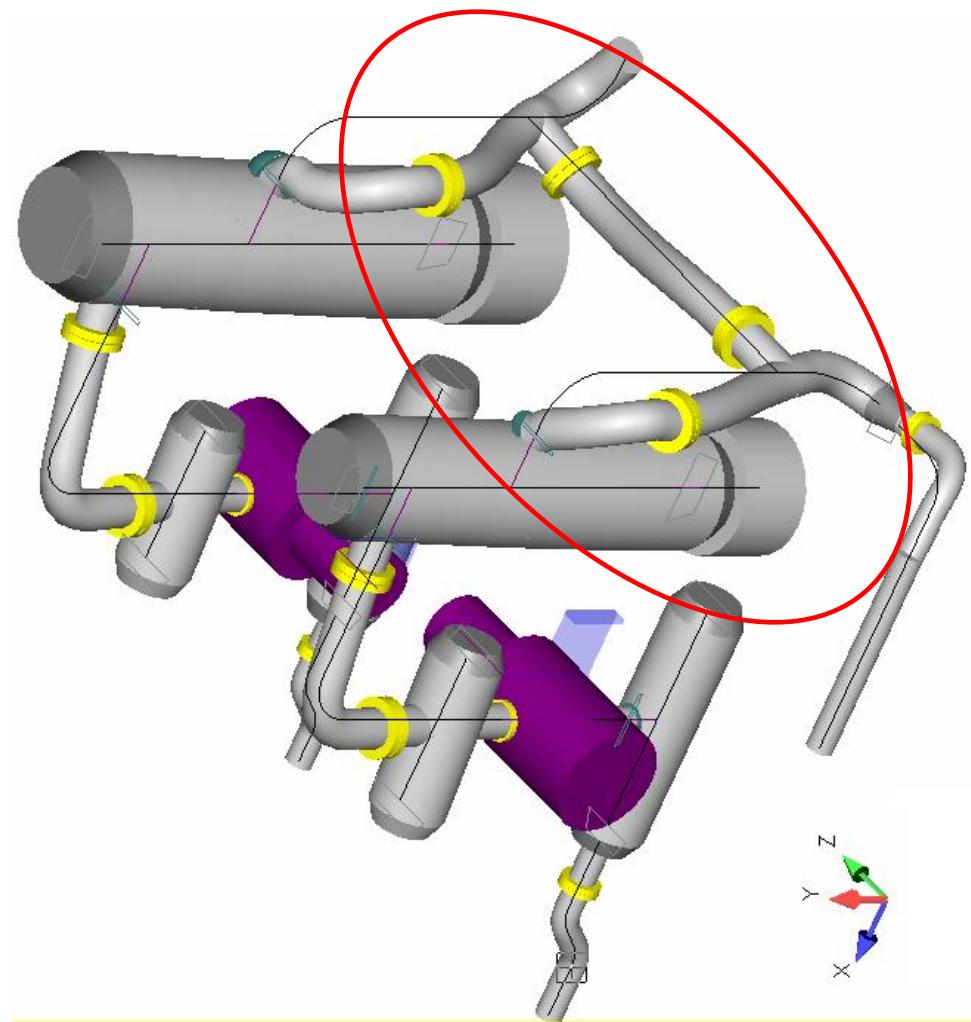


Figure 3.10 Mode Shape 9 – 126.3 Hz – Modified System

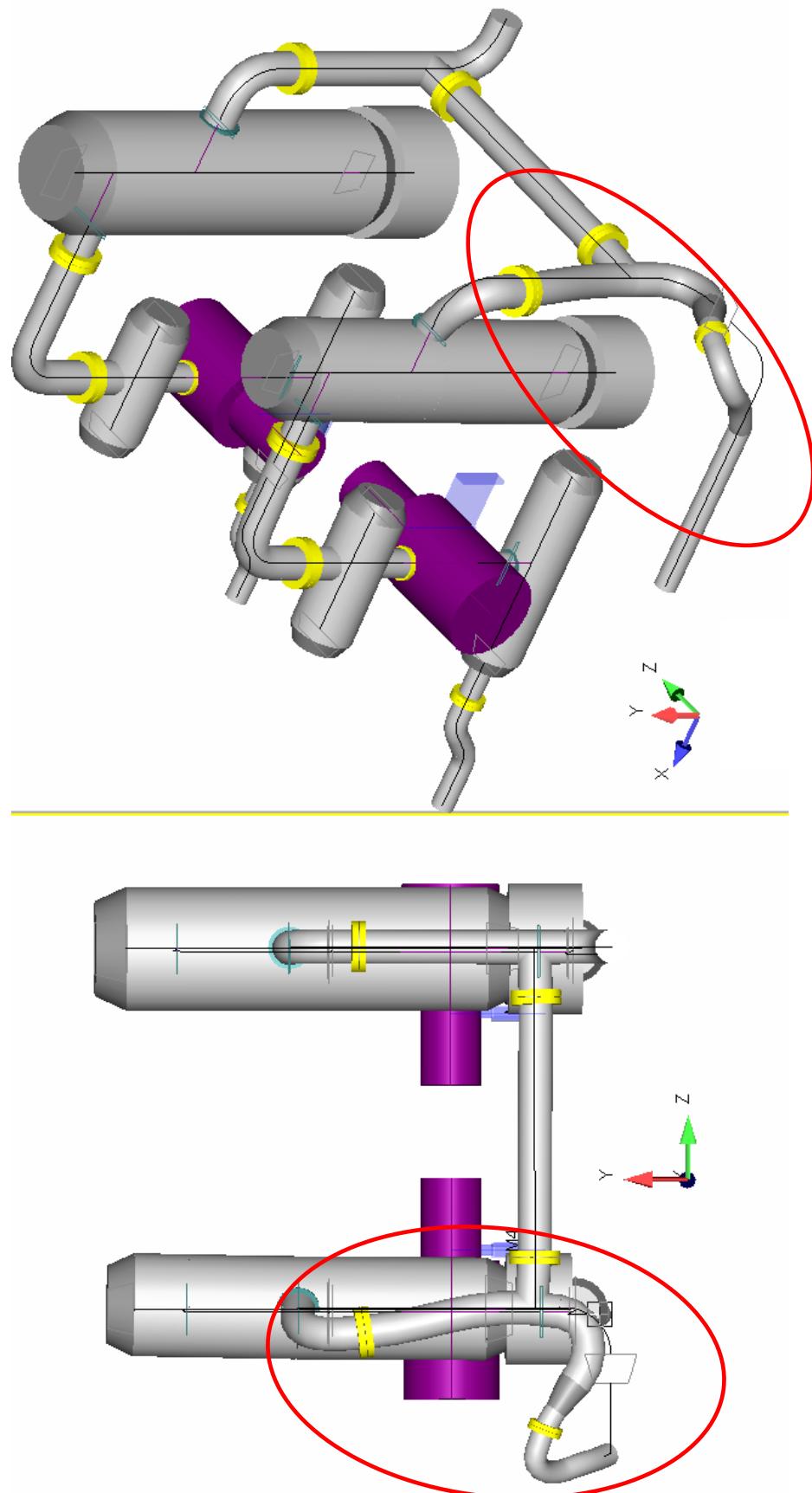


Figure 3.11 Mode Shape 10 – 149 Hz – Modified System

4. METHODOLOGY

4.1 ACOUSTICAL ANALYSIS

The analysis was performed using acoustic simulation software. The intent of the analysis is to adjust the pipeline and bottles designs of the compressor system to make it compliant with API 618 4th edition guidelines.

The software is based on the transfer matrix approach, which relates acoustic pressures and acoustic volume velocities across the all elements of the model. The elements of the model are lengths of pipe, expansions and contractions, acoustic volumes, orifice plates, Tee's, pulsation sources such as the compressor pistons and terminations, either open end, closed end, or anechoic. In essence, any configuration of piping system can be accurately modeled by combining these acoustical elements.

The program calculates the pulsation at the nodes entered as part of the model and displays the resulting pulsation spectrum on a frequency scale. These pulsations are translated into acoustically excited shaking forces. The software also provides the pressure drop across each element in the model, and if desired compares the pressure loss to the API 618 4TH EDITION allowable losses.

For reciprocating compressors, specialized boundary conditions are used to account for the losses across the valves and the intermittent sinusoidal excitation. The compressor boundary conditions can be unloaded to simulate the excitation introduced by removing the valves from a cylinder and changing it to single acting. If required, the modeling software can produce sinusoidal excitation at any node, sweeping in frequency to cover the band of interest to demonstrate the response of the system. These excitation sources can be set to be pressure sources or velocity sources.

4.2 API 618 4TH EDITION GUIDELINES

1. Pulsations in the piping system

As per §3.9.2.7 of API 618 4TH EDITION, the limit for pulsations allowed in the piping system is given by:

$$P_p = \frac{3 \times P_L}{\sqrt{P_L \times D \times f}}$$

where: P_L = line pressure (kPa)
 D = inside diameter (m)
 f = frequency (Hz)

2. Pulsations in the compressor cylinder and bottles

As per §3.9.2.2.3 of API 618 4TH EDITION, the limit for pulsations allowed at the cylinder flange is given by:

$$P_p = 0.03 \times P_L \times R \leq 0.07 \times P_L$$

where: P_L = line pressure (kPa)
 f = frequency (Hz)
 R = compression ratio for the stage

In the absence of any specific guideline for compressor nozzles and pulsation bottles, this guideline is typically used for these components of the system as well.

These guideline pulsation levels are automatically calculated by the software and compared to the pulsation present in the system. The pulsations are shown as a violation ratio, or as a ratio between the pulsation and the API 618 4TH EDITION allowable levels.

4.3 ACOUSTICAL ANALYSIS PROCEDURE

1. Data Gathering

- Piping Isometric Drawings.
- Vessel Design Drawings.
- Compressor information.
- Gas composition.
- Cooler information.
- System general arrangement.

2. Generate parameter model

- Define independent systems for analysis.
- Define model for bottles, scrubber and cooler.
- Build compressor model.

3. Troubleshooting and verification of the model

- The input data is verified.
- Model is run to assess initial 'as found' conditions for loading case.

4. Trial runs for several possible solutions are conducted.

5. Developing Solutions

- Several presented options are tried in the model, ranging from the simplest and most economical to the more elaborate.
- Optimization (de-tuning of resonant frequencies) is attempted for various solutions.

6. Generation of results files

- Result files are created and analyzed for all cases under study.
- Results are plotted for defining case only.
- Plots are created for various parts of the model, in particular for the components that need to be modified.

4.4 MECHANICAL ANALYSIS

The objective of the mechanical analysis is to assess the response of the system to the type of dynamic loading under which the systems should operate. These dynamic loads, in the form of harmonic forces are the product of the engine and compressor rotating masses. In addition, shaking forces are produced by pulsating pressures at pipeline locations where there are changes of section or direction, such as elbows, reductions, scrubbers and bottles.

For the mechanical analysis, the elements of the system are divided in two groups:

- a) large elements, including the cylinder heads, vessels and larger diameter pipeline; and
- b) smaller elements, including all small diameter pipes, valves, gauges and other ancillary elements attached to vessels and pipeline.

The analysis is conducted in two steps:

- 1.) The dynamic characteristics of the large elements are assessed using a computational model and recommendations for the attenuation of excessive vibrations are given based on the results of the computer analysis. This analysis is conducted using a combination of piping stress analysis software (AutoPIPE V8i) and finite element analysis software (FEMAP 9.3). Recommendations are the number type and location of pipe clamps. Also, notifications to the pulsation bottles result from the analysis.
- 2.) The small elements attached to the main system are surveyed and assessed for potential vibration problems and recommendations are given based on experience and the results of the simulation.

Harmonic forces can also excite vibration modes of the system if the frequencies of excitation coincide with the natural frequencies of the structure. The distribution of masses, the stiffness of the pipeline components, the stiffness of the compressor cross-head guides, and the stiffness of the pipeline and compressor supports determine the natural frequencies of the structure.

Particular attention should be given to the dynamic characteristics of the compressor manifold system. Supports should be designed to provide adequate stiffness in such a way that vibration modes of the manifold-cylinder assembly are above the range of frequencies corresponding to 2.4 times crankshaft rotation speed.

The most widely accepted method to determine the natural frequencies of the system consist of:

- a) building an approximate model with elastic beams, springs and masses; and
- b) assembling the stiffness and mass matrices and then calculating the eigenvalues and eigenvectors of the approximate system using numerical analysis techniques. The eigenvalues of the system represent the resonant frequencies, whilst the eigenvectors provide the deformed shape corresponding to each natural frequency.

The elastic and geometric properties for the beams in the model are derived from the properties of the piping. Suction and discharge bottles are modeled as pipe elements, as well as the cross-head guide and nozzles.

Precise calculation of frequencies and modal shapes depend on the accuracy with which the stiffness of the supports can be estimated. Pipeline supports are usually attached to crossbeams between the main skid beams. In this manner, the equivalent spring constant for displacements in every direction can be accurately calculated by using beam theory results.

The stiffness of the cross-guide assembly is determined from data available from the compressor manufacturer. The stiffness of the cross-guide support and crankcase are the most uncertain of the parameters. However, a combination of finite element analysis and sensitivity analysis can be used to assess the minimum stiffness necessary for adequate support.

The harmonic loading corresponding to the pressure pulsations can be accurately calculated using simulation software. If the system complies with API 618 4th edition guidelines for pulsation, the level of pressure pulsation shaking forces is normally too low to produce significant deformation, unless the frequency of the shaking force coincides with one of the natural frequencies of the system causing resonance.

Piping system natural frequencies are calculated and compared with pulsation excitation frequencies (corresponding to API 618 4th edition M.4). Recommendations on the position of supports are then made.

Finite element model of manifolds will be used to predict natural frequencies and modal shapes. The vibration characteristics will then be compared to the characteristic frequencies of excitation sources (mechanical and acoustical) in conformance to API M.5.

4.5 MECHANICAL ANALYSIS PROCEDURES

1. Data Gathering

- Piping Isometric Drawings.
- Vessel Design Drawings.
- Compressor information (mass and stiffness for different parts).
- System general arrangement.

2. Model Generation

- Select relevant operating cases.
- Select nodes on piping system to be represented in the model.
- Define model for cylinder heads, cross-head guides, bottles, scrubbers and main pipeline.
- Define forces in terms of the harmonic content of inertia loads, gas loads and pressure pulsations.

3. Troubleshooting and verification of the model

- The input data is verified.
- Model is run to assess initial 'as found' conditions for each loading case.

4. Developing Solutions

- Additional supports are implemented and tested in the model.
- Optimization (de-tuning of resonant frequencies) is attempted for various solutions.

5. Generation of Results Files

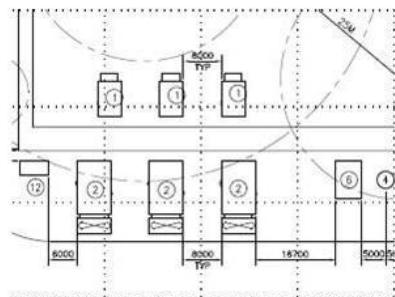
- Results are plotted for defining cases only.

5. REFERENCES

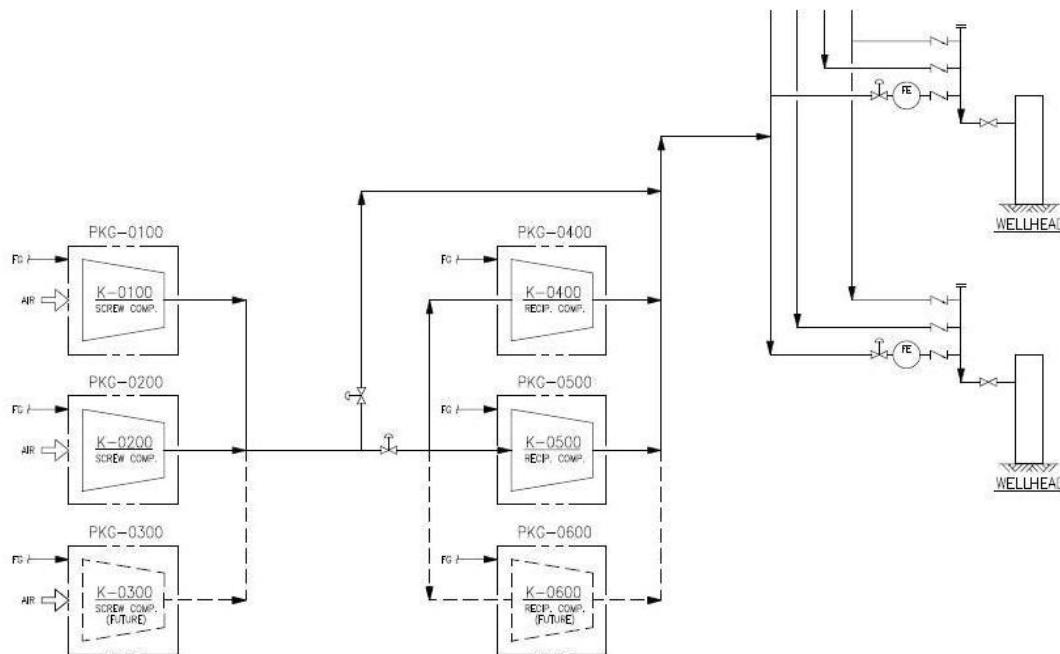
1. Alves, P. *Practical Vibrations Handbook*, Calgary: APD, 2001.
2. *API Standard 618 -- 4th ed.*, (June). Washington: American Petroleum Institute, 1995.
3. *ASME B31.3 Process Piping: Code for Pressure Piping*. New York: ASME, 1996.
4. *ASME Boiler & Pressure Vessel Code VIII*, Div. 2. Alternate Rules. New York: ASME, 2001.
5. DenHartog, J. *Mechanical Vibrations -- 4th ed.* New York: McGraw-Hill, 1956.
6. Farr, J. and M. Jawad. *Guidebook for the Design of ASME Section VIII Pressure Vessels -- 2nd ed.* New York: ASME, 2001.
7. Megyesy, E. *Pressure Vessel Handbook -- 13th ed.* Tulsa: Pressure Vessel Pub., 2004.
8. Rebis, *AutoPIPE User's Manual*, v. 6.3
9. Thomson, W.T. *Mechanical Vibrations -- 2nd ed.*, New Jersey: Prentice-Hall, 1953.

6. APPENDIX

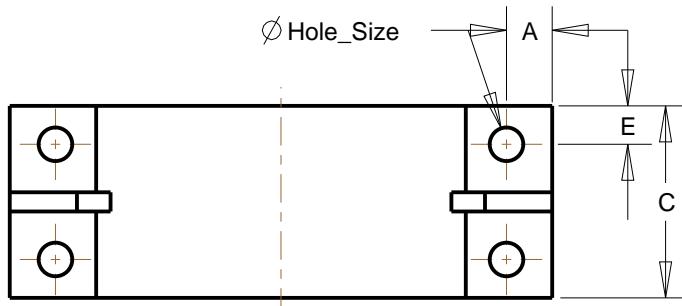
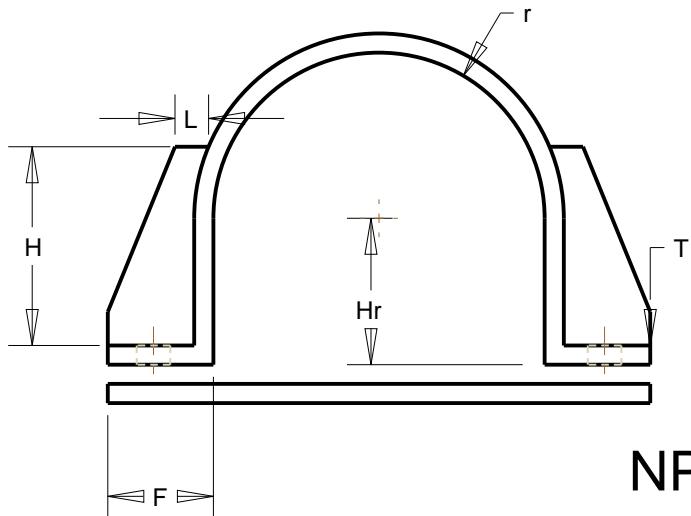
Part of the plot plan relevant to the air compressors is shown here. No. 1's are the Recip Compressor packages and No. 2's are the Screw compressor packages (s and between them is the pipe rack and No. 12 is the barrel dock. All dimensions shown are preliminary.



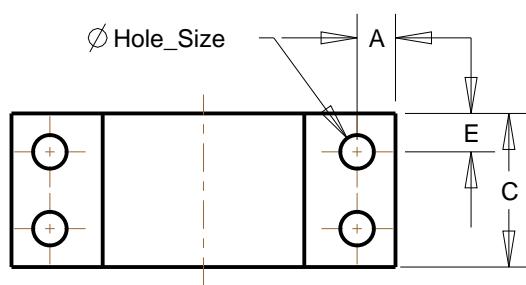
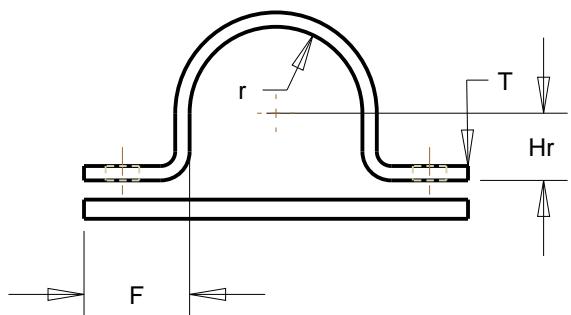
Part of the PFD relevant to the air compressors is shown here.



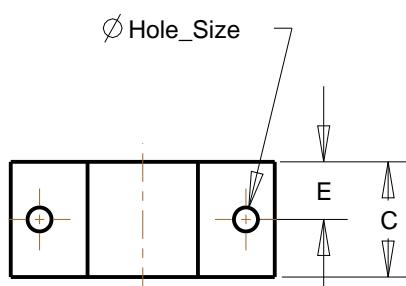
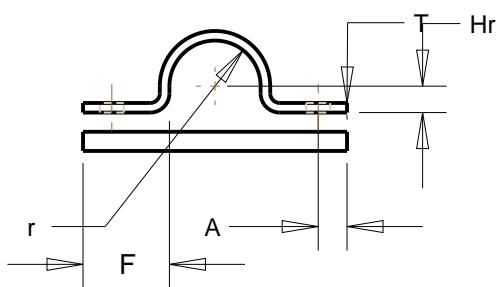
Name	r	A	C	E	F	T	Hole Size	Bolt_Size	Hr	H	L
NPS_2	1-3/16	3/4	3	1-1/2	2-1/4	1/4	5/8	0.500	11/16	-	-
NPS_3	1-3/4	1	4	2	2-3/4	1/4	7/8	0.750	1-1/4	-	-
NPS_4	2-1/4	1	4	1	2-3/4	3/8	7/8	0.750	1-3/4	-	-
NPS_6	3-5/16	1-1/8	5	1	2-3/4	3/8	7/8	0.750	2-13/16	4	11/16
NPS_8	4-5/16	1-3/16	5	1	2-3/4	1/2	7/8	0.750	3-13/16	5-3/16	7/8
NPS_10	5-3/8	1-1/2	6	1-3/16	3-1/2	1/2	1-1/8	1.000	4-7/8	6-1/2	1-1/8
NPS_12	6-3/8	1-11/16	6	1-3/16	3-3/4	1/2	1-1/8	1.000	5-7/8	7-11/16	1-5/16



NPS 6-12



NPS 4



NPS 2-3

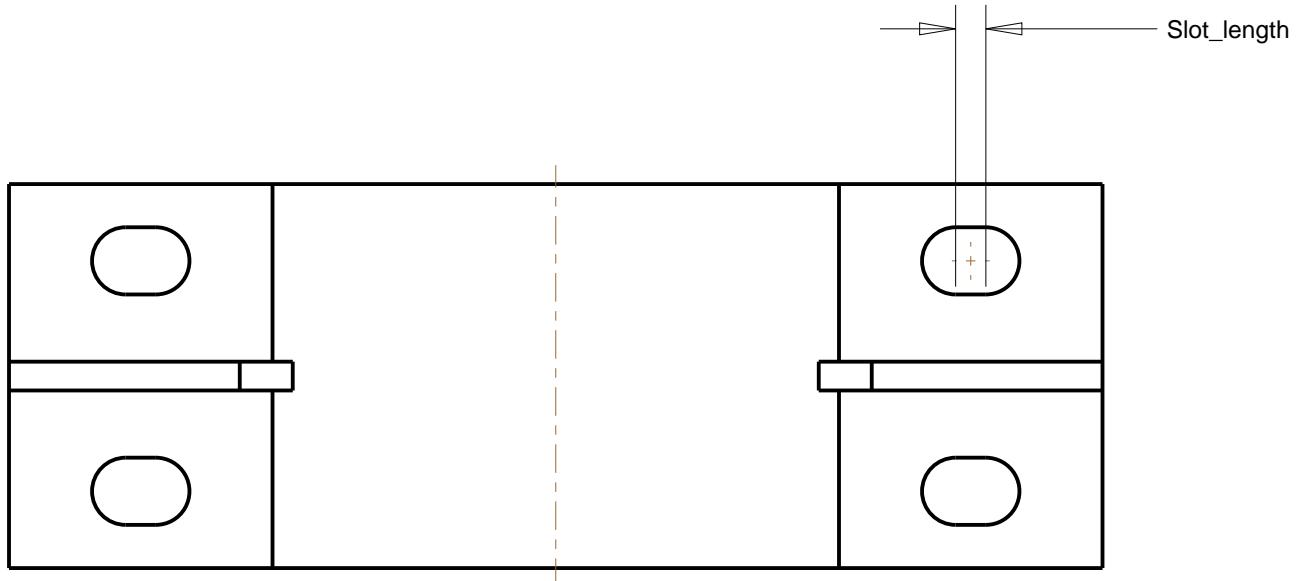
All Dimensions in inches unless otherwise noted

Note: Use double-nuts or lock nuts to prevent loosening of the clamp over time

AP Dynamics
Calgary Alberta - Houston Texas

Pipe clamps for
vibrating service

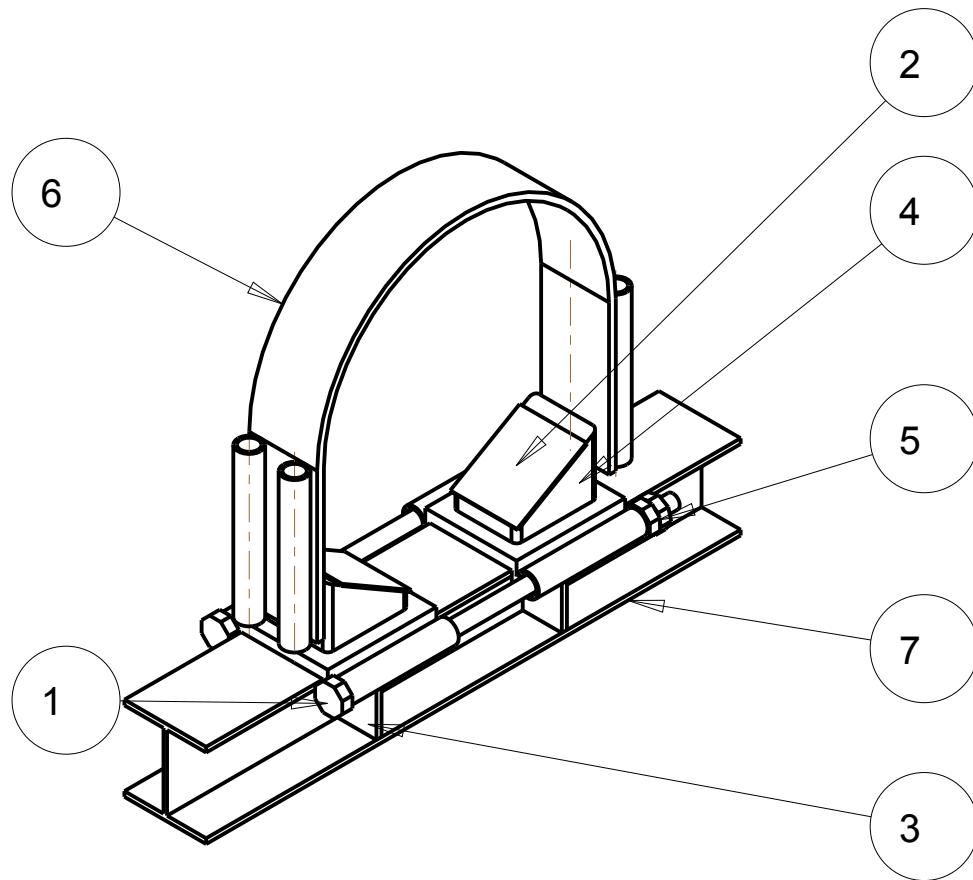
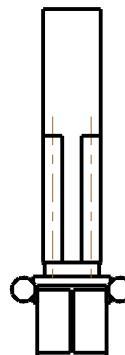
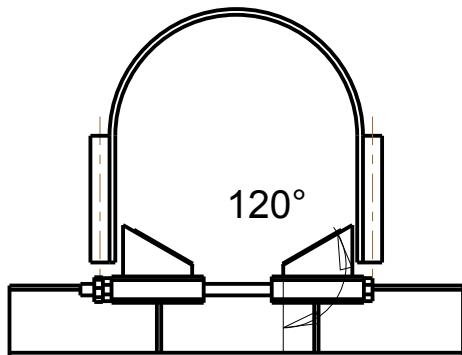
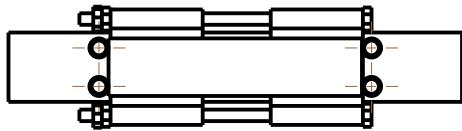
Slotted Clamp Variation



For lateral slots, increase clamp tab length "F" by slot length
For axial slots, increase clamp length "C" to accomodate

AP Dynamics
Calgary Alberta - Houston Texas

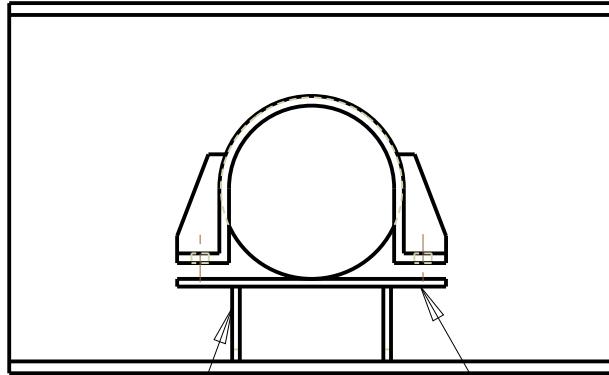
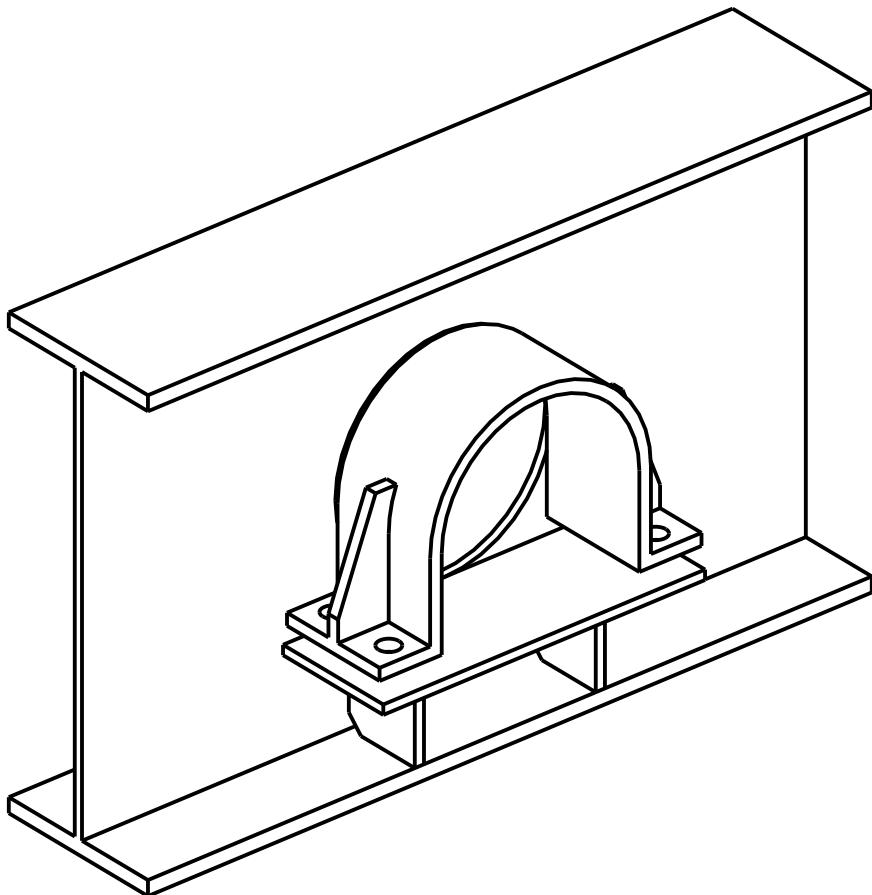
Pipe clamps for
vibrating service



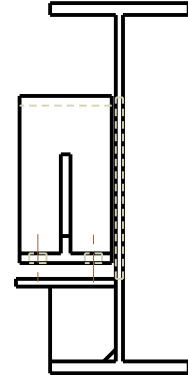
Item	Description	QTY
1	1" Bolt in 1-1/4 SCH 80 Pipe Collar	2
2	1/8" Fabreeka Pad	2
3	3/8" Full Depth Gusset	4
4	HSS cut to fit	2
5	Nut	4
6	Optional Strap, 1/2" Thick	1
7	Wide Flange or HSS Member	1

AP DYNAMICS
CALGARY ALBERTA - HOUSTON TEXAS

Bottle Wedge with
optional strap

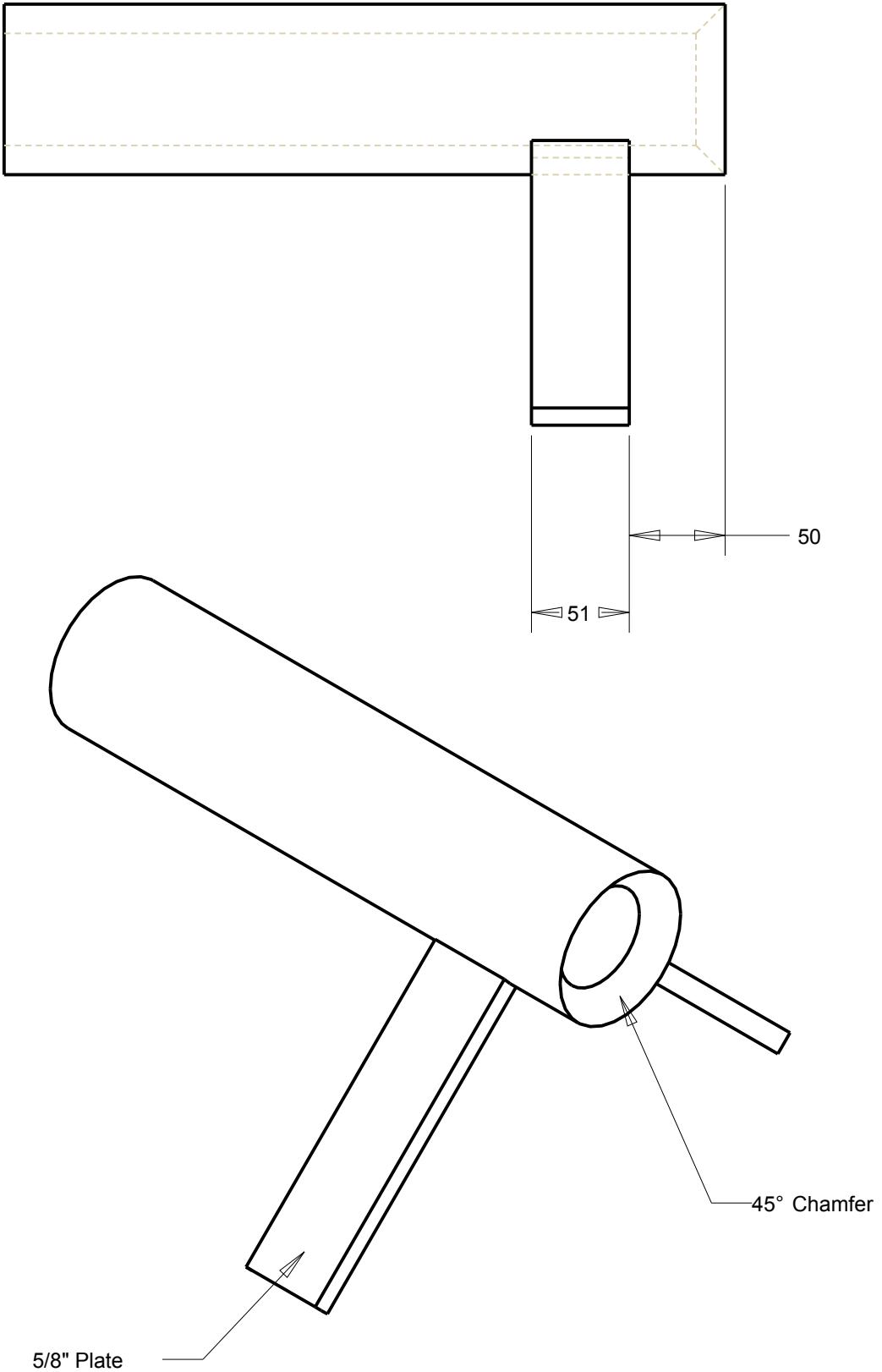


TYP → 3/8" 3/8" → 3/8"



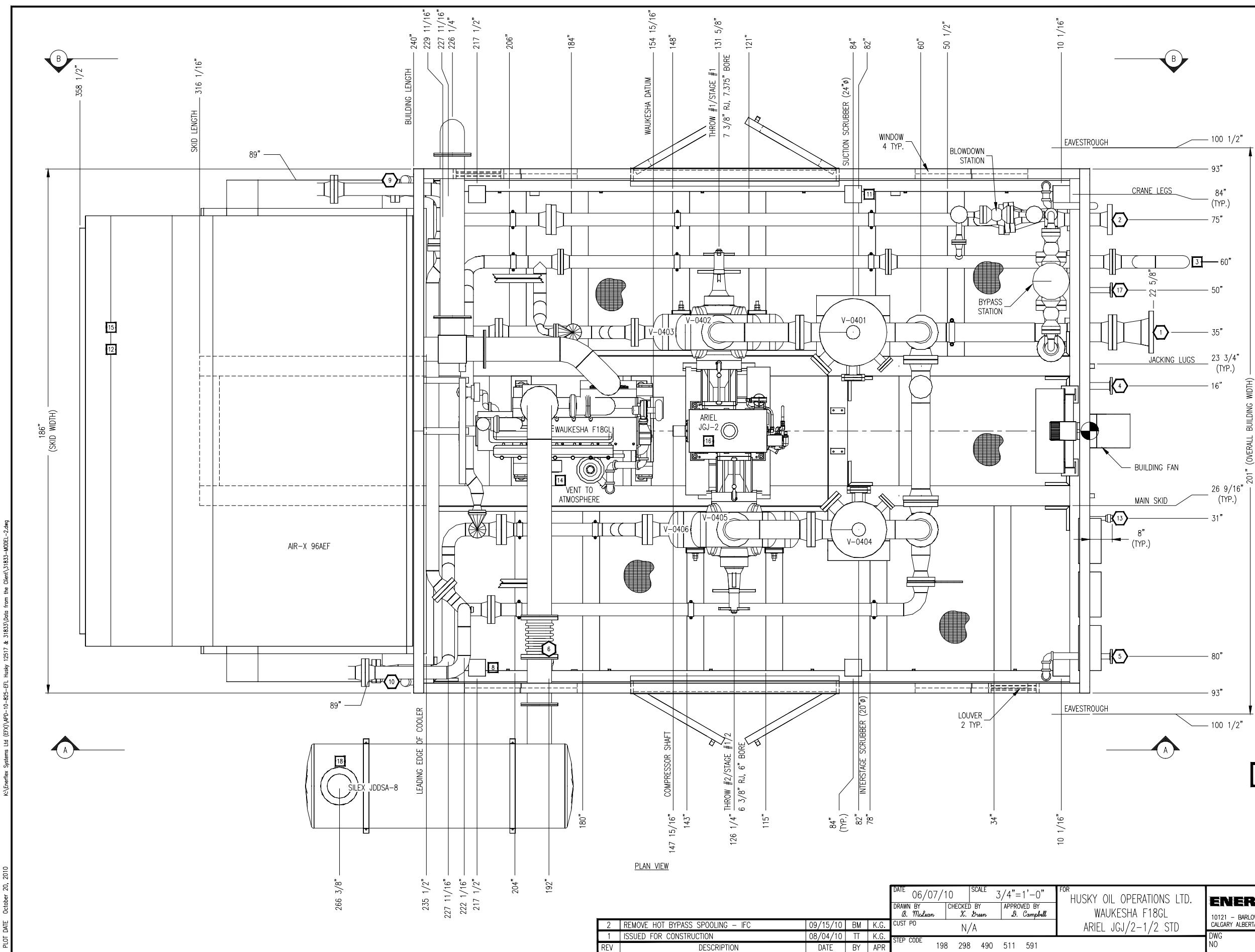
AP DYNAMICS
CALGARY ALBERTA - HOUSTON TEXAS

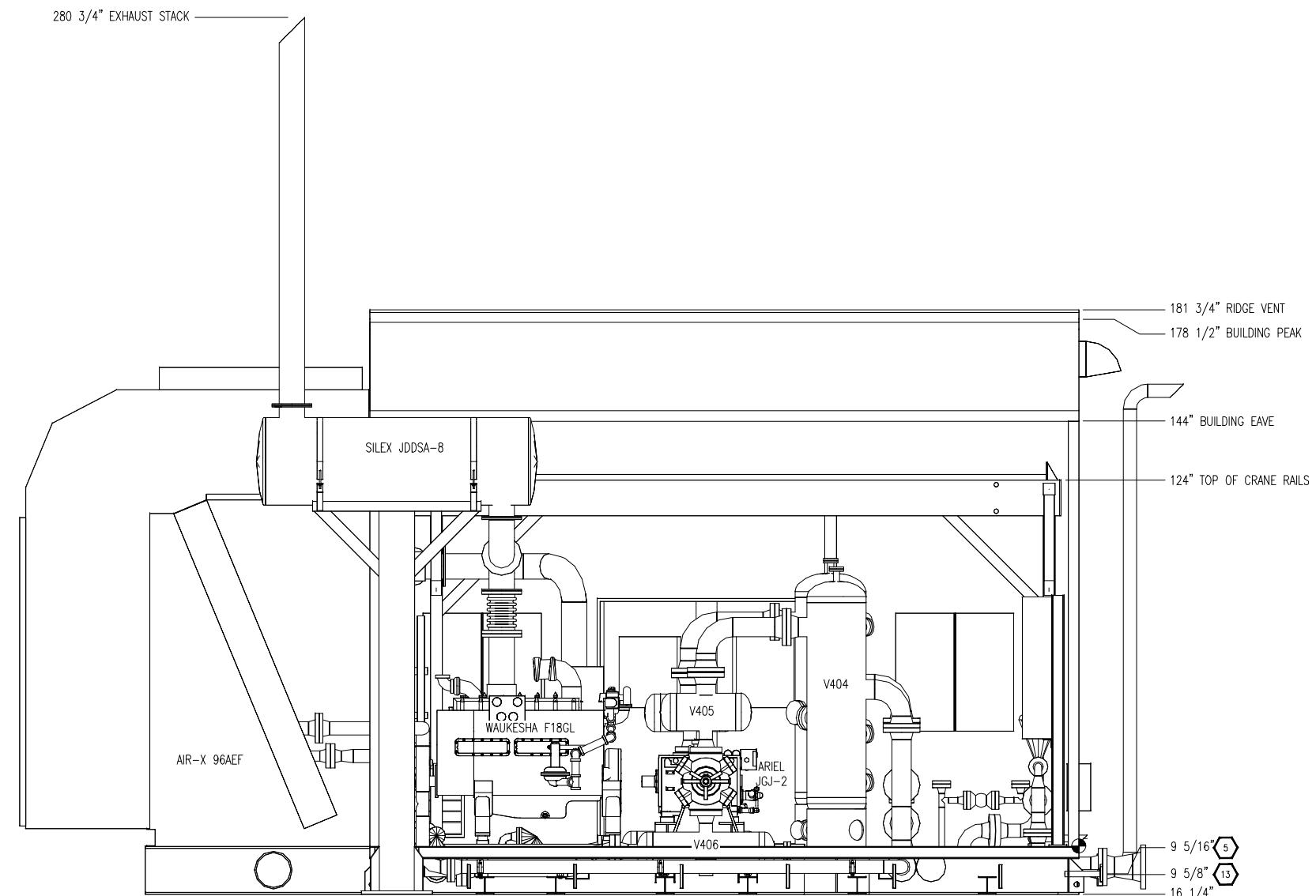
Typical Skid Edge Support



AP DYNAMICS
CALGARY ALBERTA - HOUSTON TEXAS

Typical choke tube end support





ELEVATION VIEW A-A

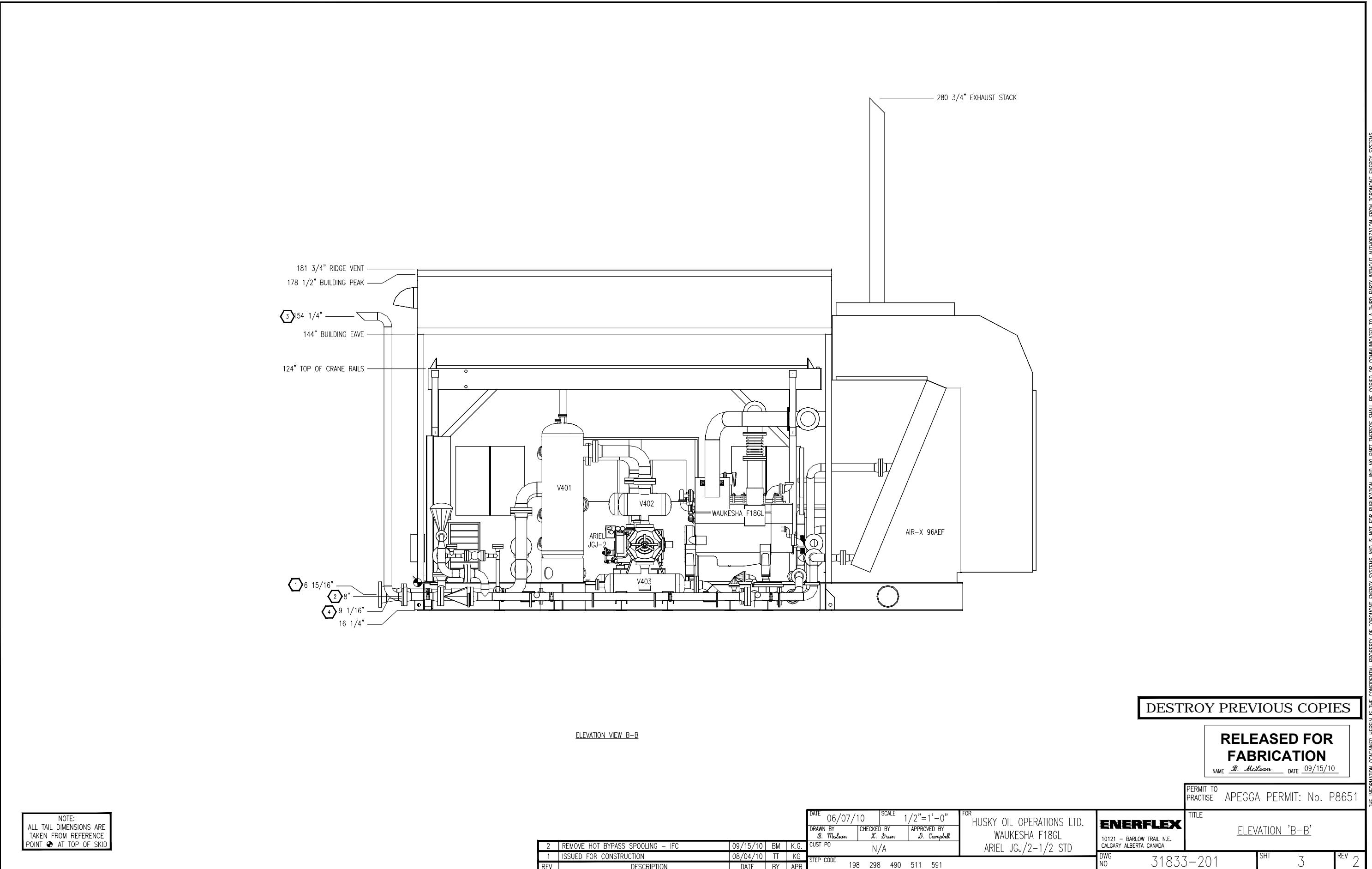
DESTROY PREVIOUS COPIES

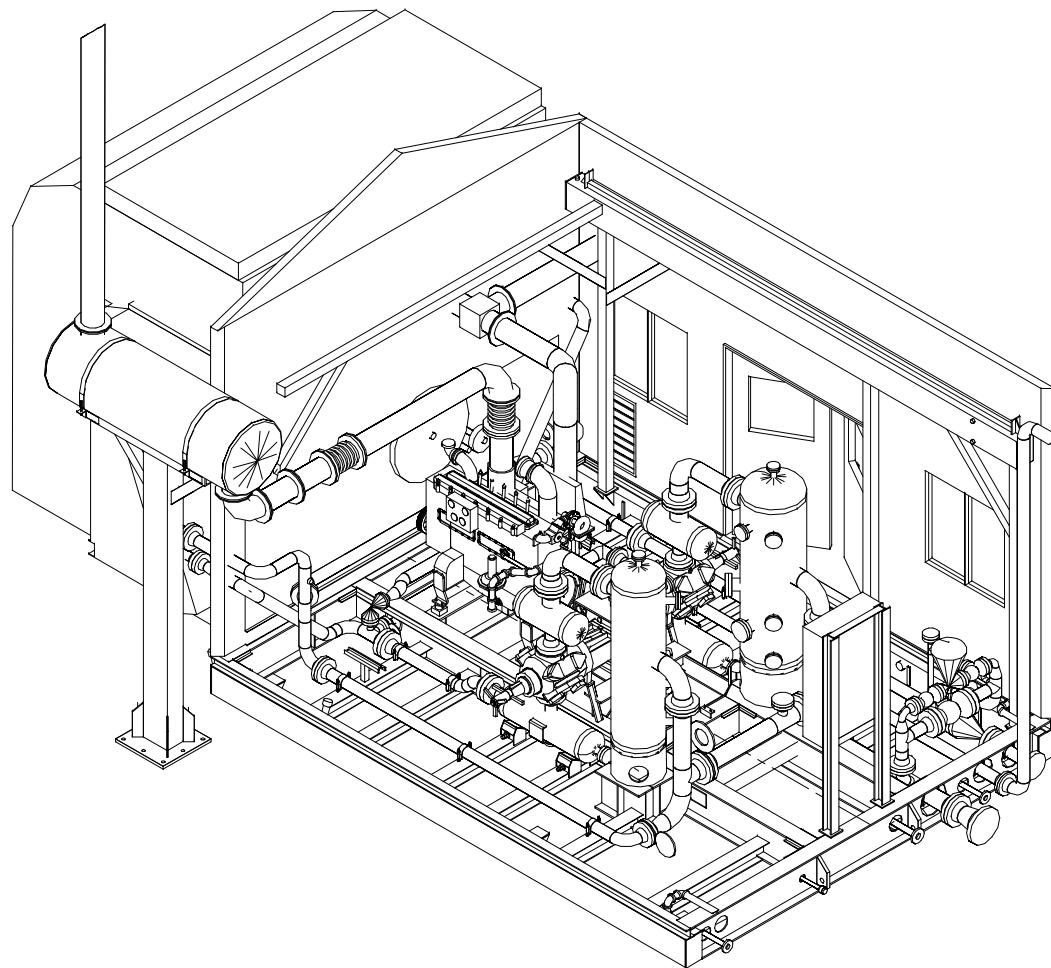
**RELEASED FOR
FABRICATION**
NAME B. McLean DATE 09/15/10

PERMIT TO PRACTISE APEGGA PERMIT: No. P8651	
TITLE <u>ELEVATION 'A-A'</u>	
DWG NO	31833-201
SHT	2
REV	2

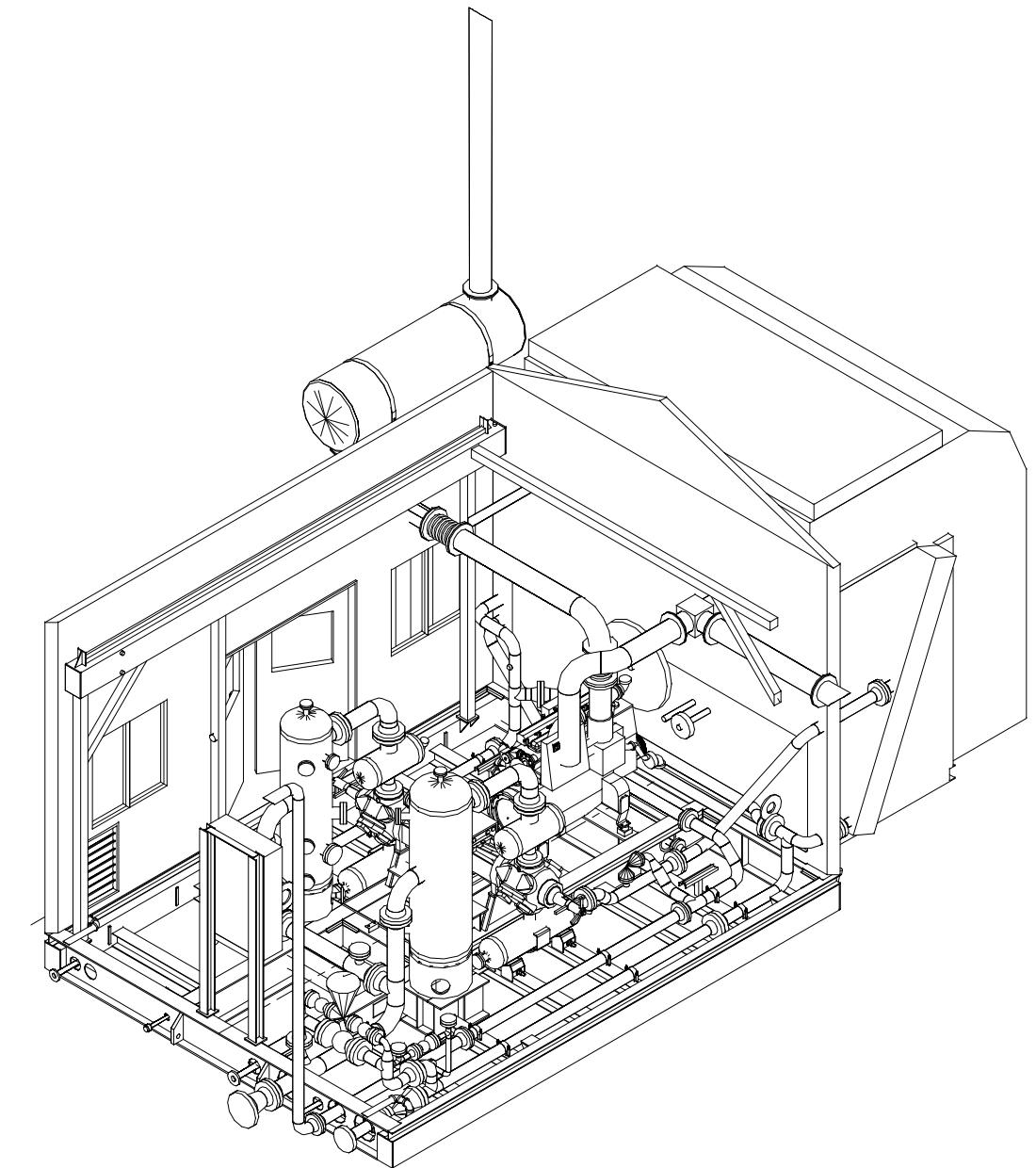
NOTE:
ALL TAIL DIMENSIONS ARE
TAKEN FROM REFERENCE
POINT AT TOP OF SKID

2 REMOVE HOT BYPASS SPOOLING - IFC		09/15/10	BM	K.G.	DATE 06/07/10	SCALE 1/2"=1'-0"	FOR HUSKY OIL OPERATIONS LTD.
1 ISSUED FOR CONSTRUCTION		08/04/10	TT	KG	DRAWN BY <u>B. McLean</u>	CHECKED BY <u>X. Green</u>	APPROVED BY <u>B. Campbell</u>
REV	DESCRIPTION	DATE	BY	APR	CUST PO	N/A	
		STEP CODE	198	298	490	511	591





3D VIEW



DESTROY PREVIOUS COPIES

**RELEASED FOR
FABRICATION**

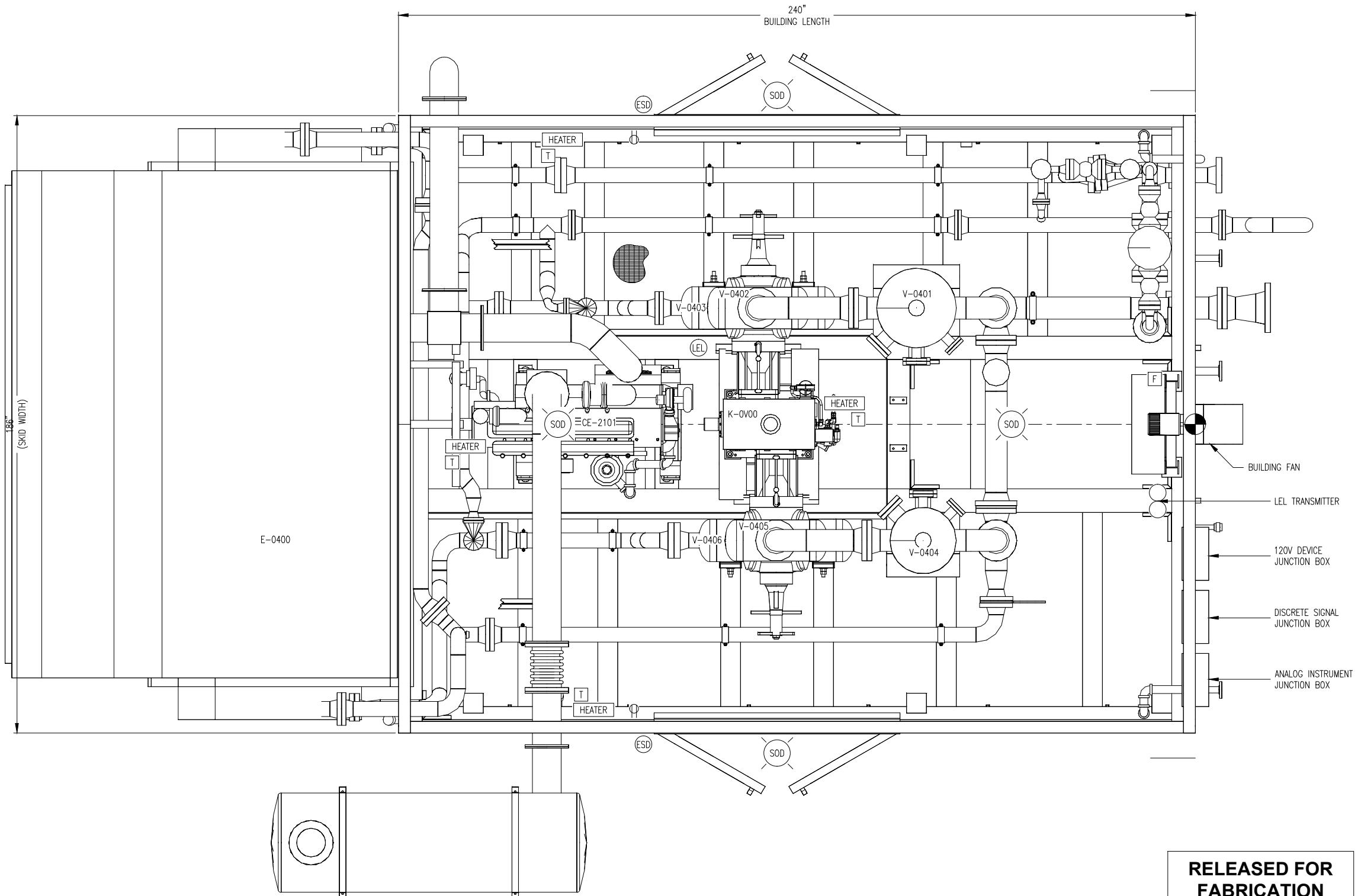
NAME B. McLean DATE 09/15/10

PERMIT TO
PRACTISE APEGGA PERMIT: No. P8651

ENERFLEX
10121 - BARLOW TRAIL N.E.
CALGARY ALBERTA CANADA

TITLE
3D VIEW

DATE	06/07/10	SCALE	3/8" = 1'-0"	FOR
DRAWN BY	<u>B. McLean</u>	CHECKED BY	<u>X. Green</u>	APPROVED BY
CUST PO		N/A		<u>S. Campbell</u>
STEP CODE	198	298	490	511 591
REV	DESCRIPTION	DATE	BY	APR
2	REMOVE HOT BYPASS SPOOLING - IFC	09/15/10	BM	K.G.
1	ISSUED FOR CONSTRUCTION	08/04/10	TT	KG



PLAN VIEW

1	ISSUED FOR CONSTRUCTION	08/04/10	BM	K.G.
REV	DESCRIPTION	DATE	BY	APR

DATE 06/17/10 SCALE 3/4" = 1'-0"
DRAWN BY B. McLean CHECKED BY X. Green APPROVED BY S. Campbell
CUST PO N/A STEP CODE 198 298 490 511 591

FOR HUSKY OIL OPERATIONS LTD.
WAUKESHA F18GL
ARIEL JGJ/2-1/2 STD

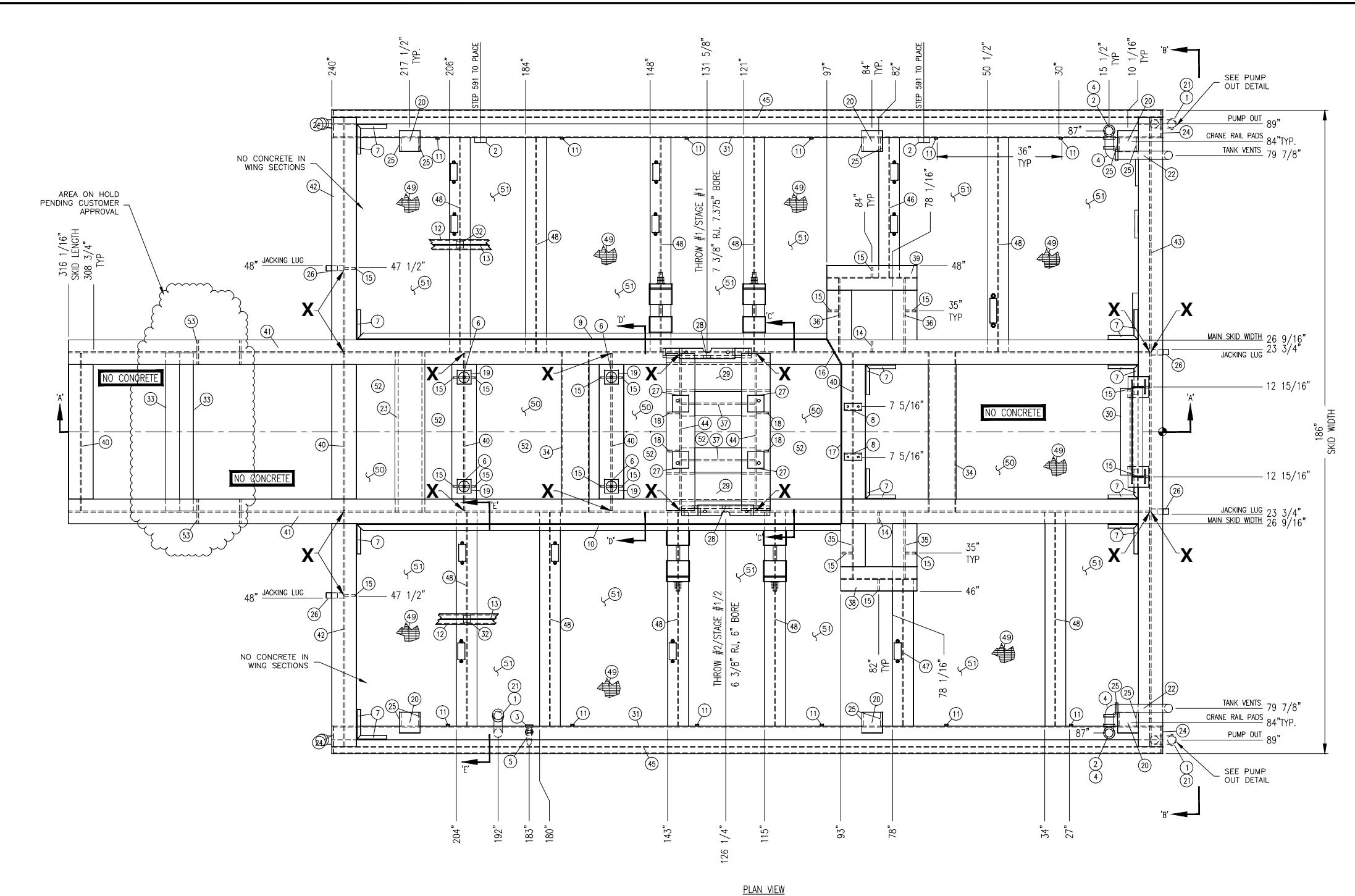
ENERFLEX
10121 - BARLOW TRAIL N.E.
CALGARY ALBERTA CANADA

PERMIT TO PRACTISE APEGGA PERMIT: No. P8651
TITLE ELECTRICAL LAYOUT
DWG NO 31833-201 SHT 5 REV 1

ELECTRICAL LEGEND	
SYMBOL	DESCRIPTION
(L)	LIGHT
(INC)	INCANDESCENT LIGHT
(SOD)	H.P. SODIUM LIGHT
(E)	BATTERY POWERED EMERGENCY LIGHT
(R)	RECEPTACLE
(T)	THERMOSTAT
(F)	FIRE EYE
(H) _F	FIRE PULL STATION
(R)	COMBUSTIBLE LEL GAS ALARM BEACON
(B)	TOXIC GAS H ₂ S ALARM BEACON
(C) _F	FIRE ALARM STROBE
(F)	ALARM HORN
(CO)	CO SENSOR
(LEL)	COMBUSTIBLE DETECTION
(H ₂ S)	H ₂ S SENSOR
(HEATER)	HEATER
(TIME)	TIMER SWITCH
(ESD)	EMERGENCY SHUT DOWN BUTTON
	-
	-
	-
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	-
	-
	-
	-

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NAME B. McLean DATE 08/04/10



PLAN VIEW

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NOTES

- GRIND ALL ROUGH EDGES & WELD SPATTER
- SANDBLAST INTERNALS OF GLYCOL STORAGE TANK PRIOR TO FINAL ASSEMBLY.
- AIR PRESSURE TEST GLYCOL & OIL TANKS TO 10 PSI.
- CONSTRUCT STRUCTURE FLUSH TO TOP UNLESS OTHERWISE NOTED.
- ALL TAN DIMENSIONS FROM REFERENCE POINT.
- WEIGHT OF SKID (w/o CONCRETE) 10,500 lbs.
- 1/4" IN LENGTH HAS BEEN REMOVED FROM ITEMS #40 & 44 FOR FIT UP.
- SEAL PAN IS FLAT.
- PRIMER - TPS-SP-001, FIN-1, RED OXIDE.
PAINT - TPS-SP-001, FIN-1, 20-F "WARM GREY".
- ALL WELDS TO BE 1/4" MIN. FILLETS ALL AROUND, UNLESS OTHERWISE NOTED.
- FULL PENETRATION WELD REQUIRED AT THESE LOCATIONS.
- X FULL STRENGTH WELD REQUIRED AT THESE LOCATIONS.

BILL OF MATERIAL

MK	QTY	DESCRIPTION	MAT.	LENGTH	PART NUMBER
1	3	CAP,2",3000#,NPT	SA-105	-	115-HRB-0200
2	4	COUPLING,2",3000#,NPT,HALF	SA-105	-	123-HRB-0200
3	1	ELBOW,1",3000#,45D,NPT	SA-105	-	142-HRB-0100
4	4	ELBOW,2",3000#,90D,NPT	SA-105	-	146-HRB-0200
5	1	PIPE,1",XS,SMLS,TOE&POE (CUT TO SUIT)	SA-106-B	4	600-HPA-0100
6	4	VIBRACON SM16 ADJUSTABLE CHOCK			NPN
7	1	ANGLE,1-1/4" X 1-1/4",3/16" THK	G40.2144W	900	005-HA-0125-0125
8	2	ANGLE,2" X 2",1/4" THK	G40.2144W	5	005-HA-0200-0200-0025
9	1	ANGLE,2" X 2",1/4" THK	G40.2144W	135 15/16	005-HA-0200-0200-0025
10	1	ANGLE,2" X 2",1/4" THK	G40.2144W	139 15/16	005-HA-0200-0200-0025
11	12	CHANNEL,1" X 0.83"	G40.2144W	8	018-HA-0100-0008
12	2	CHANNEL,3" X 4.1#	G40.2144W	225 7/8	018-HA-0300-0041
13	2	FLAT BAR,1" X 1/4" THK	G40.2144W	225 7/8	010-HA-0100-0025
14	2	FLAT BAR,3" X 1/2" THK	G40.2144W	4 1/2	010-HA-0300-0050
15	18	FLAT BAR,3" X 1/2" THK	G40.2144W	15	010-HA-0300-0050
16	1	FLAT BAR,3/4" X 1/4" THK	G40.2144W	8 1/8	010-HA-0075-0025
17	1	FLAT BAR,3/4" X 1/4" THK	G40.2144W	46	010-HA-0075-0025
18	4	FLAT BAR,3/12" X 1/2" THK	G40.2144W	19 3/4	010-HA-0350-0050
19	4	MOUNTING BLOCK 4" SQUARE,x 2", C1018 COLD ROLLED, FLAT BAR(SEE DWG. EMB-H24-F8 REV.0)	C1018	4	NPN
20	6	FLAT BAR,6" X 1/2" THK	G40.2144W	6	010-HA-0600-0050
21	3	PIPE,2",S/80(XS),ERW	SA-106-B	15	602-HPB-0200
22	1	PIPE,2",S/80(XS),ERW	SA-106-B	48	602-HPB-0200
23	1	PIPE,8",XS,SMLS	SA-106-B	46 3/8	600-PPA-0800
24	4	PLATE,1/2",SMOOTH X 1' X 8"	G40.2144W	12	040-HA-0050
25	12	PLATE,1/2",SMOOTH X 7 3/16" X 3 5/16"(ONE CUTS TWO)	G40.2144W	7 3/16	040-HA-0050
26	4	PLATE,1-1/2",SMOOTH X 1' X 5" (SEE DWG. # LIFT LUG REV.0)	G40.2144W	12	040-HA-0150
27	4	MOUNTING BLOCK 4 3/4"SQUARE,x 2", C1018 COLD ROLLED, FLAT BAR(SEE DWG. ARIEL-MB-JGRJ2 REV.0)	C1018	4 3/4	NPN
28	2	CYLINDER SUPPORT, SEE DWG. #JGJ-ALL CYLINDER SUPPORT	G40.2144W	-	NPN
29	2	PLATE,3/4",SMOOTH X 2'-6" X 10 3/4"	G40.2144W	30	040-HA-0075
30	1	PANEL STAND, 30" WIDE, SEE DWG. PANEL STAND REV.1	G40.2144W	-	NPN
31	2	TUBING,12"X8",1/4" THK	G40.21350W	239	020-HA-1200-0800-0025
32	12	TUBING,2"X2",3/16" THK	G40.21350W	9 1/8	020-HA-0200-0200-0018
33	2	TUBING,4"X4",1/4" THK	G40.21350W	47 1/4	020-HA-0400-0400-0025
34	2	TUBING,8"X6",1/4" THK	G40.21350W	46 3/8	020-HA-0800-0600-0025
35	2	WIDE FLANGE,16" X 50#	G40.2150W	19 1/16	034-HA-1600-0500
36	2	WIDE FLANGE,16" X 50#	G40.2150W	21 1/16	034-HA-1600-0500
37	2	WIDE FLANGE,16" X 50#	G40.2150W	21 5/16	034-HA-1600-0500
38	1	WIDE FLANGE,16" X 50#	G40.2150W	22	034-HA-1600-0500
39	1	WIDE FLANGE,16" X 50#	G40.2150W	26	034-HA-1600-0500
40	5	WIDE FLANGE,16" X 50#	G40.2150W	45 3/8	034-HA-1600-0500
41	2	WIDE FLANGE,16" X 50#	G40.2150W	312 5/16	034-HA-1600-0500
42	2	WIDE FLANGE,16" X 50#	G40.2150W	67 5/8	034-HA-1600-0500
43	1	WIDE FLANGE,16" X 50#	G40.2150W	181 11/16	034-HA-1600-0500
44	2	WIDE FLANGE,21" X 68#	G40.2150W	45 3/8	034-HA-2100-0680
45	2	WIDE FLANGE,4" X 13#	G40.2150W	240	034-HA-0400-0130
46	1	WIDE FLANGE,6" X 15#	G40.2150W	40 3/8	034-HA-0600-0150
47	1	WIDE FLANGE,6" X 15#	G40.2150W	42 3/8	034-HA-0600-0150
48	10	WIDE FLANGE,6" X 15#	G40.2150W	61 13/16	034-HA-0600-0150
49	-	BAR GRATING, 1" DEEP X 3/16" (SQ. FT.)		220	049-19W4-HAP-0018-0100
50	1	PLATE,12 GA,SMOOTH SHEAR TO 3"-6" (CTS) (SHOP TO ORDER)	G40.2144W	228	044-HA-0012
51	-	PLATE,12 GA,SMOOTH (CTS) (SQ. FT.) (SHOP TO ORDER)	G40.2144W	200	044-HA-0012
52	-	CONCRETE, MIX 20B5GN1, 20 MPa, STRENGTH C/W AIR, 2292 KG/M CU. (142LBS/FT3). (M CU.) (SEE SECTION A-A FOR LOCATIONS)		2.1	CONCRETE-20B5GN1-350
53	2	PIPE,12",XS,SMLS	SA-106-B	7 1/16	602-JPB-1200

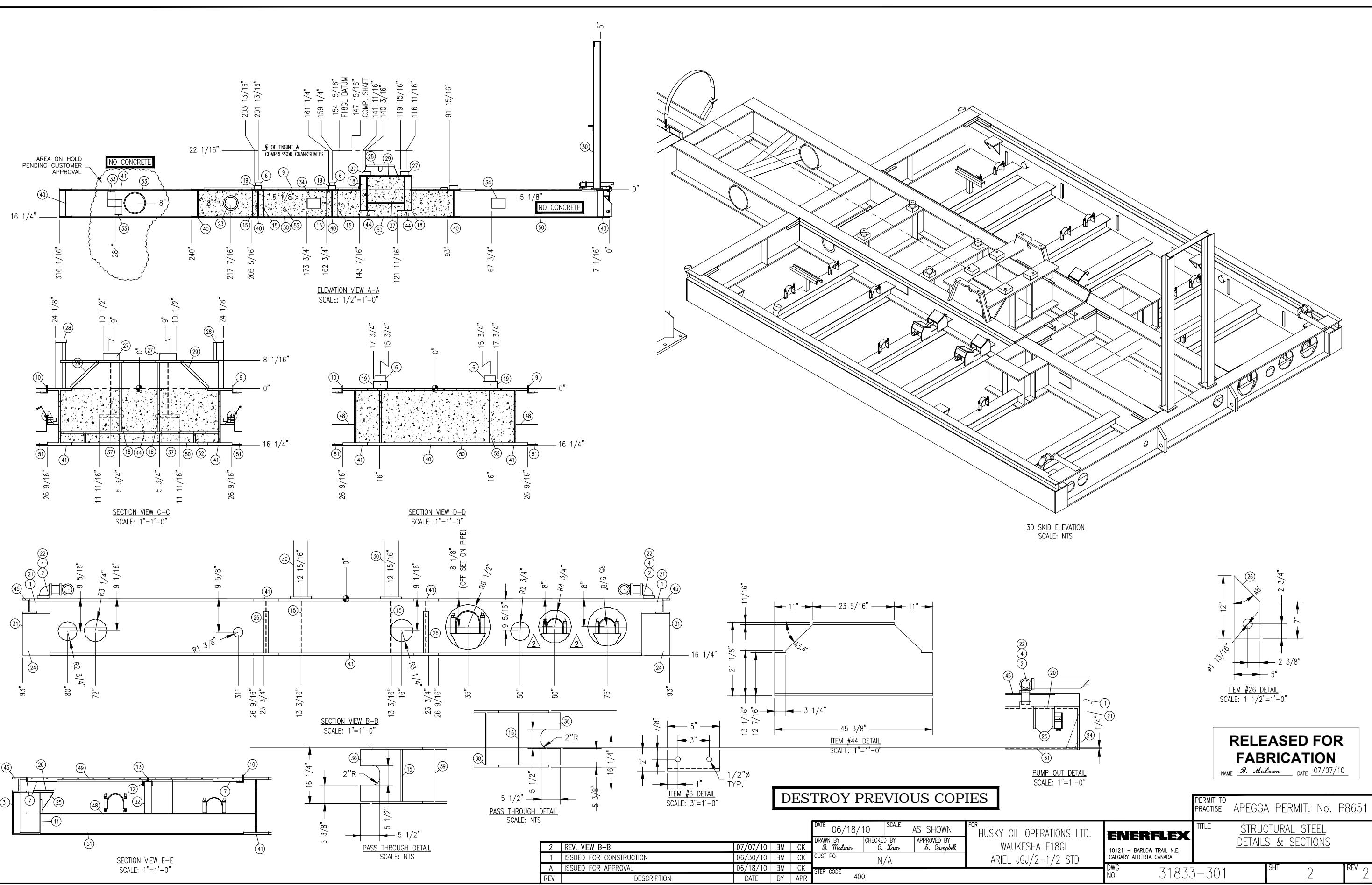
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FABRICATION**

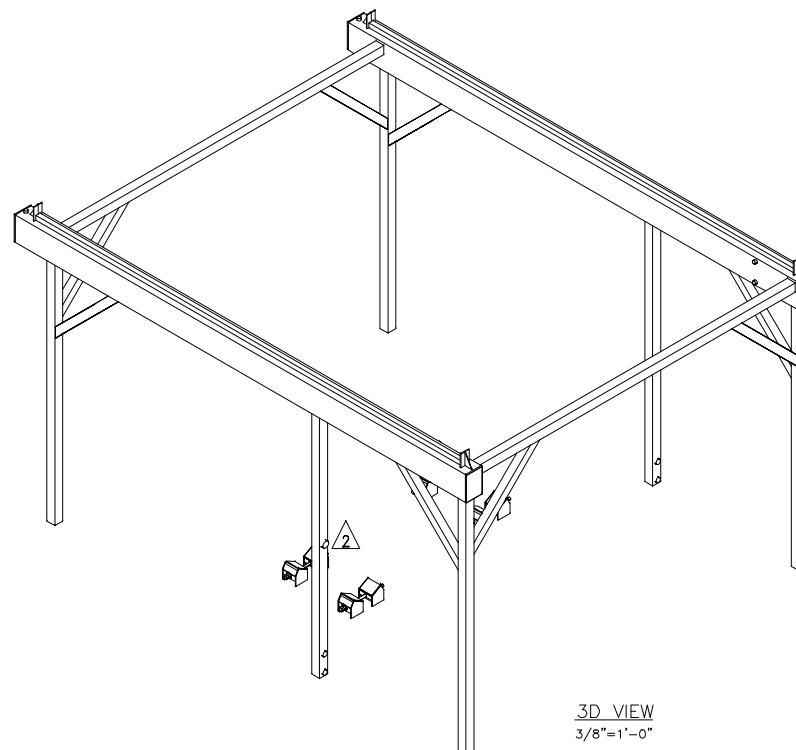
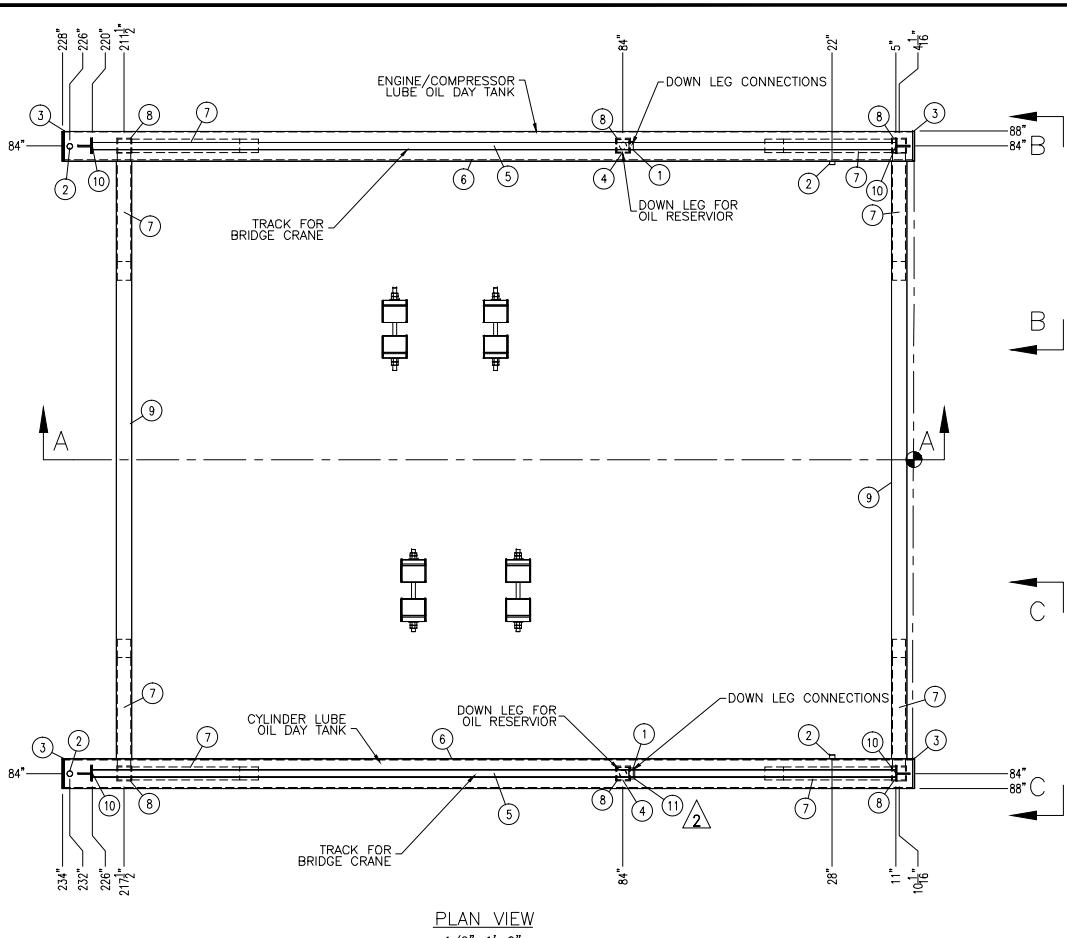
NAME B. McLean DATE 07/08/10

DESTROY PREVIOUS COPIES

DATE	06/18/10	SCALE	3/4"=1'-0"	FOR	HUSKY OIL OPERATIONS LTD.
DRAWN BY	<u>B. McLean</u>	CHECKED BY	<u>C. Yam</u>	APPROVED BY	<u>B. Campbell</u>
CUST PO	N/A				
STEP CODE	400				
REV		DESCRIPTION		DATE	BY APR

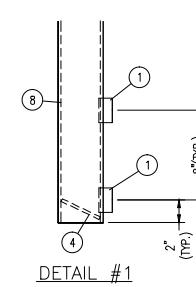
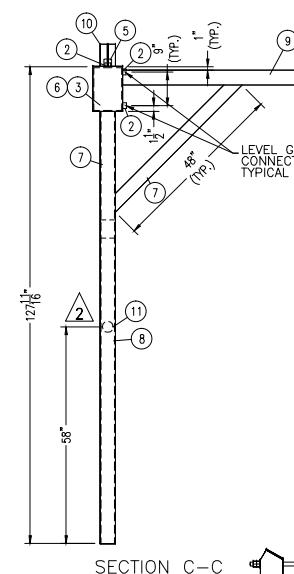
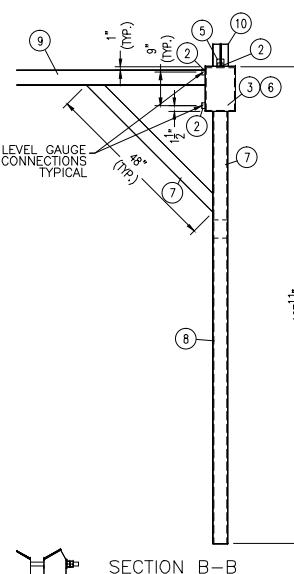
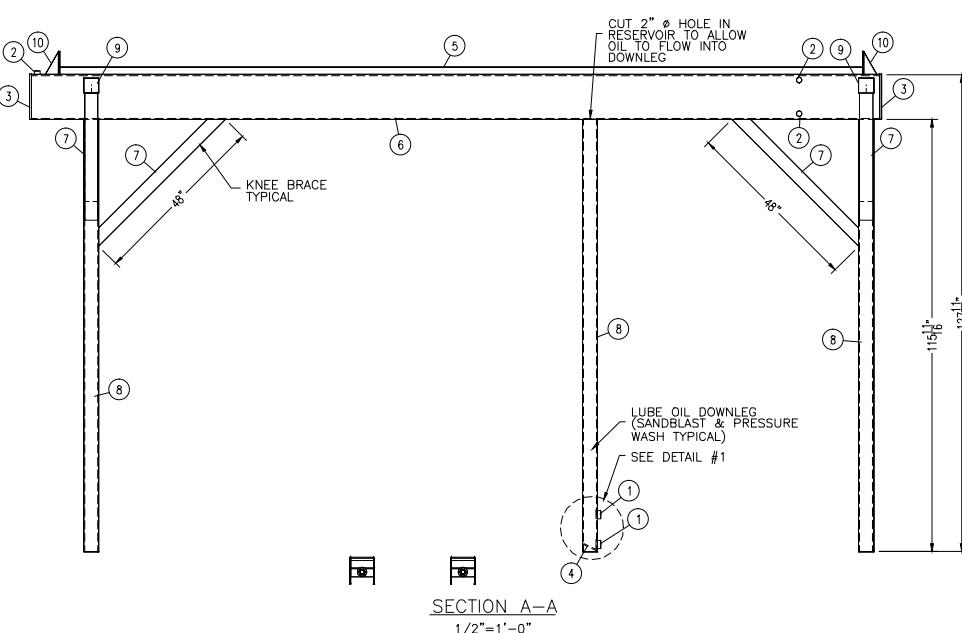
PERMIT TO PRACTISE	APEGGA PERMIT: No. P8651
TITLE	STRUCTURAL STEEL
DWG NO	31833-301
SHT	1
REV	2





BILL OF MATERIAL					
MK	QTY	DESCRIPTION	MAT.	LENGTH	PART_NUMBER
1	4	COUPLING,1",3000#,NPT,HALF	SA-105	—	123-HRA-0100
2	6	COUPLING,1/2",3000#,NPT,HALF	SA-105	—	123-HRA-0050
3	4	PLATE,1/2",SMOOTH X 1" X 8"	G40.2144W	12	040-HA-0050
4	2	PLATE,1/4",SMOOTH X 3 3/4" X 3 3/8"	G40.2144W	3 3/4	040-HA-0025
5	2	SQUARE BAR: 2	G40.2144W	215	014-HA-0200
6	2	TUBING,12"X8",1/4" THK	G40.21350W	227	020-HA-1200-0800-0025
7	8	TUBING,3-1/2"X3-1/2",1/4" THK	G40.21350W	48	020-HA-0350-0350-0225
8	6	TUBING,4"X4",1/4" THK	G40.21350W	115 11/16	020-HA-0400-0400-0025
9	2	TUBING,4"X4",1/4" THK	G40.21350W	160	020-HA-0400-0400-0025
10	4	WIDE FLANGE,4" X 1 1/2" (ONE CUTS TWO)	G40.2150W	6	034-HA-0400-0130
11	1	COUPLING,1 1/2",3000#,NPT,HALF	SA-105	7/8	123-HRA-0150

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DESTROY PREVIOUS COPIES

A STAMP

**RELEASED FOR
FABRICATION**

BY B. McLean DATE 09/01/10

NOTE:

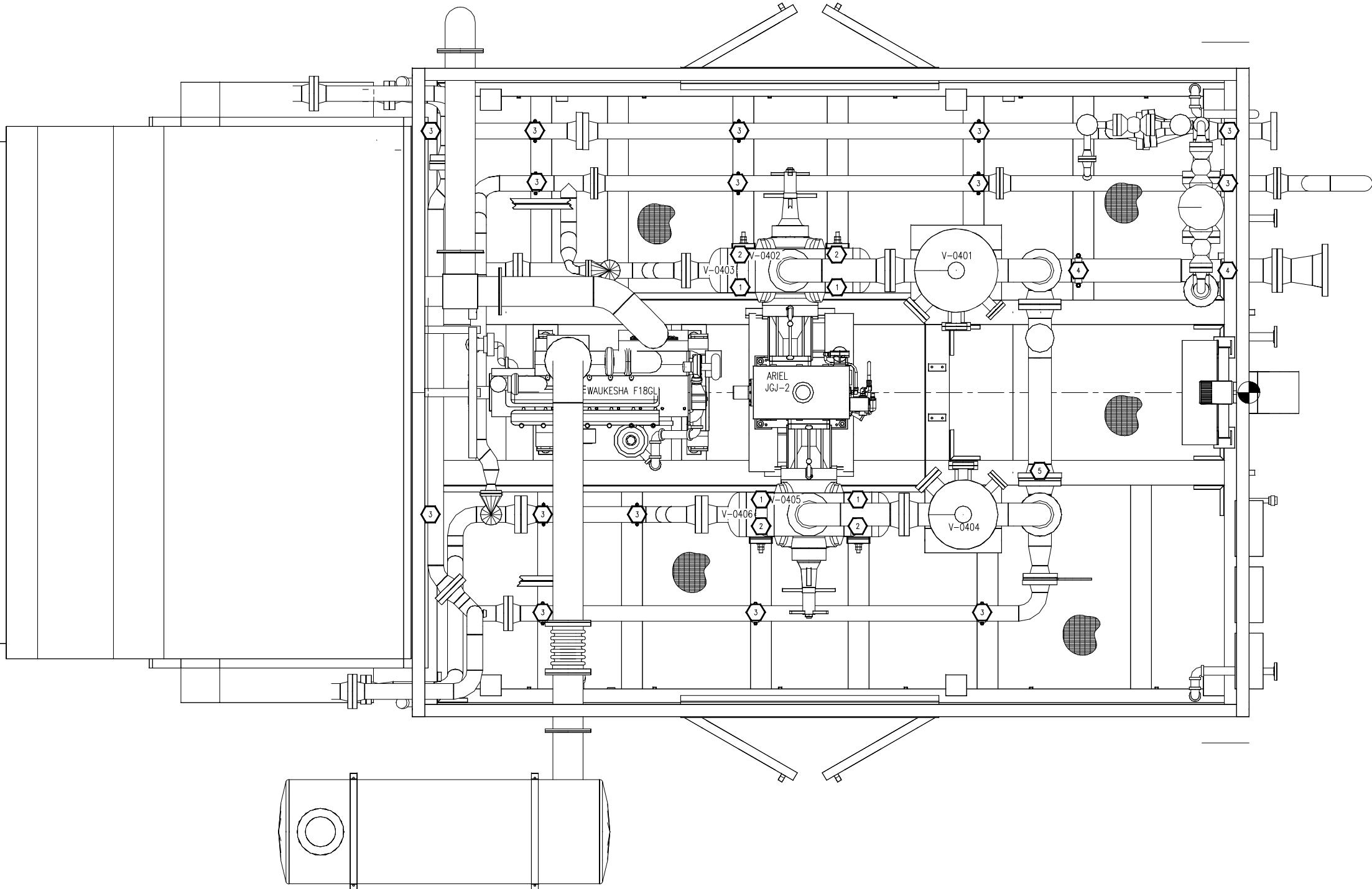
1. ALL TAIL DIMENSIONS ARE TAKEN FROM REFERENCE POINT C
2. ALL FABRICATION TO BE IN ACCORDANCE TO CSA W59-M.
3. ALL MATERIAL TO BE GRADE G40.21-M.
 - 300W FOR SECTIONS & PLATE
 - 350W FOR HSS SECTION
4. ALL BOLTS/CONNECTIONS - GRADE A325 UNC.

1

2	ADD LS CONNECTION
1	ISSUED FOR CONSTRUCTION
REV.	DESCRIPTION

DATE 09/01/10	SCALE AS SHOWN
DRAWN BY <i>B. McLean</i>	CHECKED BY <i>KB</i>
CUST PO	APPROVED BY —
N/A	
STEP CODE	

PERMIT TO PRACTISE		APEGGA PERMIT: No. P8651	
FOR HUSKY OIL OPERATIONS LTD. WAUKESHA F18GL ARIEL JGJ/2-1/2 STD		TITLE ENERFLEX 10121 - BARLOW TRAIL N.E. CALGARY ALBERTA CANADA <u>CRANE RAILS</u>	
DWG NO	31833-301	SHT 3	REV 2



PLAN VIEW

SCBW DETAIL LEGEND			
MARK	SERVICE	SIZE	QTY
1	BOTTLE CLAMPS	12 3/4"	4
2	LOW PRO BOTTLE WEDGES	STD	4
3	4"-PIPE CLAMPS	4"	15
4	6"-PIPE CLAMPS	6"	2
5	6"-FLANGE GRABBER	-	1

FOR APPROVAL
PLEASE SUBMIT ONE SET OF
APPROVED DRAWINGS BY
NAME J. Focher DATE 08/04/10

PERMIT TO
PRACTISE APEGGA PERMIT: No. P8651
TITLE PLAN VIEW
DWG NO 31833-301 SHT 4 REV 1

DATE 06/07/10 SCALE 3/4"=1'-0" FOR
DRAWN BY B. McLean CHECKED BY X. Green APPROVED BY B. Campbell
CUST PO N/A HUSKY OIL OPERATIONS LTD.
WAUKESHA F18GL
ARIEL JGJ/2-1/2 STD
10121 - BARLOW TRAIL N.E.
CALGARY ALBERTA CANADA

ENERFLEX

1	ISSUED FOR CONSTRUCTION	08/04/10	TT	K.G.
REV	DESCRIPTION	DATE	BY	APR

STEP CODE 198 298 490 511 591

air-x-limited

2230 East 49th Street Tulsa, Oklahoma 74105
 PHONE: (918) 743-6111 FAX: (918) 742-7039
 E-mail: jmcurley@airxlimited.com

Purchaser:	ENERFLEX (CGY)	From:	John Curley
To:	Abi Fatokun	Date:	May 11, 2010
Inquiry #:	Q190393	End User:	HUSKY
Proposal #:	105370	Destination:	
No. Pages:	2	Reference:	F18GL - JGJ/2 (formerly 95466)

PROPOSAL

In response to subject inquiry, we propose the following design in accordance with specification sheet **105370**

Dated: **5/11/2010**

Comments:

(1) Model 96AEF

Mounted enclosed slide idler

Extended lube lines

(2) Sets manual discharge shutters over water - tied together

(2) Sets automatic discharge shutters over gas with Garzo (or equal) / Kimray controls

Water Services – 15% excess over std duties

SA214 Tubing

1" NPT vent

Gas Services - 5% excess

350°F / -49°F MDMT

SA214 Tubing

1/16" Corrosion Allowance

1" Coupling opposite each discharge nozzle

Spot X-Ray

PWHT

AXH standard paint system tinted to grey

Discharge guard

Soft removable bugscreens

CRN Registration: AB, BC, SASK

Customer specifications with clarifications and exceptions attached

Fan(s):	(1) MOORE-10K S30VT	<i>Tip speed (fpm): 10,505</i>
	<i>Dia, In / # Blades: 96 / 6</i>	<i>Est. Noise Data: 85 dBA @1m, 64 dBA @ 15m</i>
Net price* (USD)	Shipment: FOB Calgary AB	\$47,435

**The quoted price is firm for 30 days based on shipment in accordance with our current production schedule.*

Shipment is: 8 weeks ARO and released for fabrication. Please confirm availability at time of order.



401 E. Lowry Road Phone (918) 283-9200
 Claremore, OK 74017 Fax (918) 283-9229
info@axh.com www.axh.com

Proposal / Job No.	105370
Date	5/11/2010
Page	1 OF 1

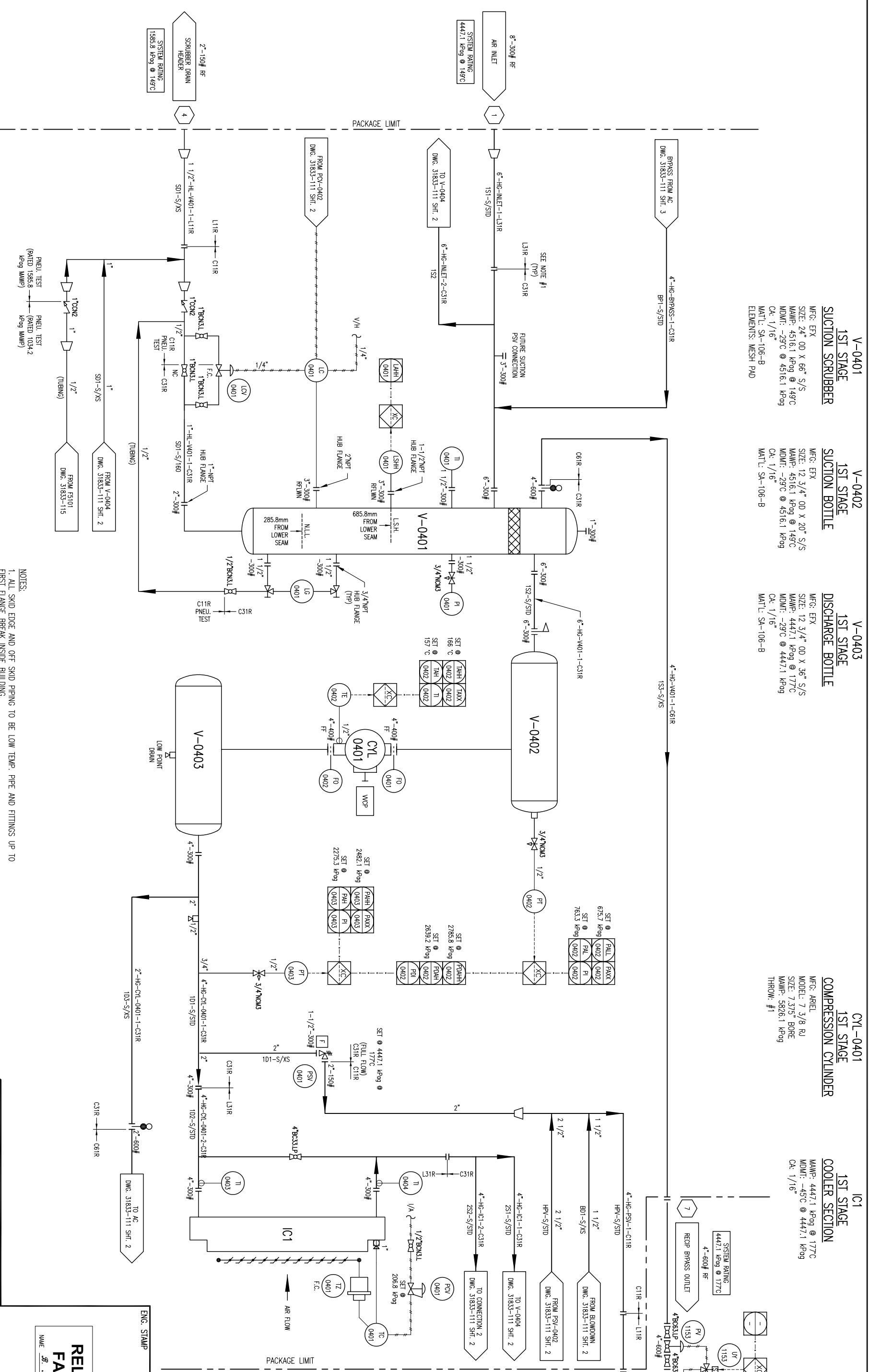
1	Purchaser	ENERFLEX (CGY)	Ultimate User	HUSKY
2	Inquiry / PO#	Q190393	Destination	
3	# Units	1	Model	96AEF
4	Assembly	PACKAGED	Draft	FORCED

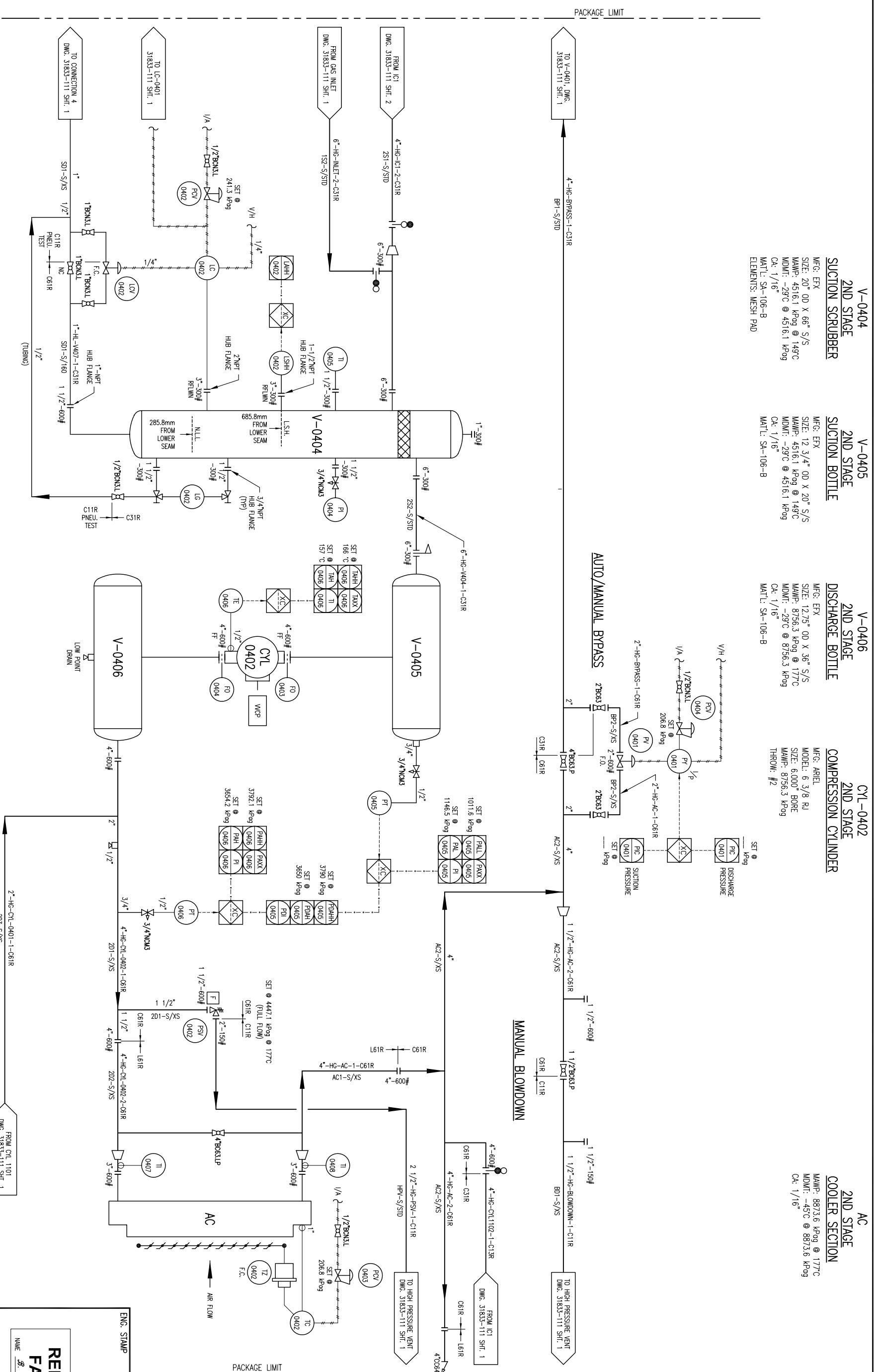
THERMAL & MECHANICAL DESIGN

5	Service	EJW	AUX-W	IC	AC
6	Flow	124GPM	35GPM	5.03MMSCFD	5.03MMSCFD
7	Fluid	50%GLY	50%GLY	AIR	AIR
8	Temp. In / Out, °F	180.0 / 165.0	156.8 / 130.0	235.0 / 115.7	249.0 / 122.0
9	Pressure, PSI			322PSIG	518PSIG
10	Pressure Drop, PSI	2.1	5.0	4.0	4.0
11	Heat Load, BTU/HR	864800	425735	475155	512019
12	True LMTD	48.5	28.8	39.5	47.6
13	Overall Rate, U	161.7	152.1	57.6	65.0
14	Fouling Factor	.0005	.0005	.0020	.0020
15	Surface, Tube / Total, Sq Ft	112 / 1789	99 / 1574	209 / 4468	166 / 3544
16	Sections, #	(1) - COMBINED	(1) - COMBINED	(1)	(1)
17	Design Temp, °F Max / Min	300 / -20	300 / -20	350 / -49	350 / -49
18	MWP / Test Press, PSIG	* 14 / 75	* 14 / 75	645 / 839	1287 / 1674
19	Pass Arrangement	CROSSFLOW	CROSSFLOW	CROSSFLOW	CROSSFLOW
20	# Tube Rows	3	3	4	4
21	# Tube Passes	1	3	3	3
22	Tubes, OD x BWG	5/8X16	5/8X16	1X16	1X16
23	Material	SA214 STEEL	SA214 STEEL	SA214 STEEL	SA214 STEEL
24	# Per Section / Length, Ft	50 / 14	44 / 14	58 / 14	46 / 14
25	Turbulators				
26	Accelerators				
27	Fins, Type	HI-EFF	HI-EFF	HI-EFF	HI-EFF
28	Material	AL	AL	AL	AL
29	Nozzles, Rating / Type	150RF	150RF	300RF	600RF
30	Material	SA105	SA105	SA350LF2	SA350LF2
31	#-Inlets / Size In	(1) 3	(1) 2	(1) 4	(1) 3
32	#-Oulets / Size In	(1) 3	(1) 2	(1) 4	(1) 3
33	Headers, Type	REC TUBE	REC TUBE	BOX	BOX
34	Material	A500	A500	SA516-70N	SA516-70N
35	Corrosion Allow, In			.0625	.0625
36	Grooved Tubesheet	SGL	SGL	DBL	DBL
37	Plugs, Type	TAPER	TAPER	SHOULDER	SHOULDER
38	Plugs Material	STEEL	STEEL	SA350LF2	SA350LF2
39	PWHT			YES	YES
40	ASME Code & Nat'l Board			YES	YES
41	CRN			AB, BC, SK	AB, BC, SK
42	Add'l Specs & Options				
43	API				
44	Louvers / Hail Screen	MAN /	MAN /	MAN /	MAN /
45	Inspection / NDT			SX	SX

FX= 100% X-Ray of all header seam, attachment & nozzle butt welds. **SX**= Spot X-Ray of 1 long seam & 1 end closure, per header
BX= 100% X-Ray of all nozzle butt welds. **UT** = 100% UT of all header seam, attachment & nozzle butt welds. **H** = Hardness testing.

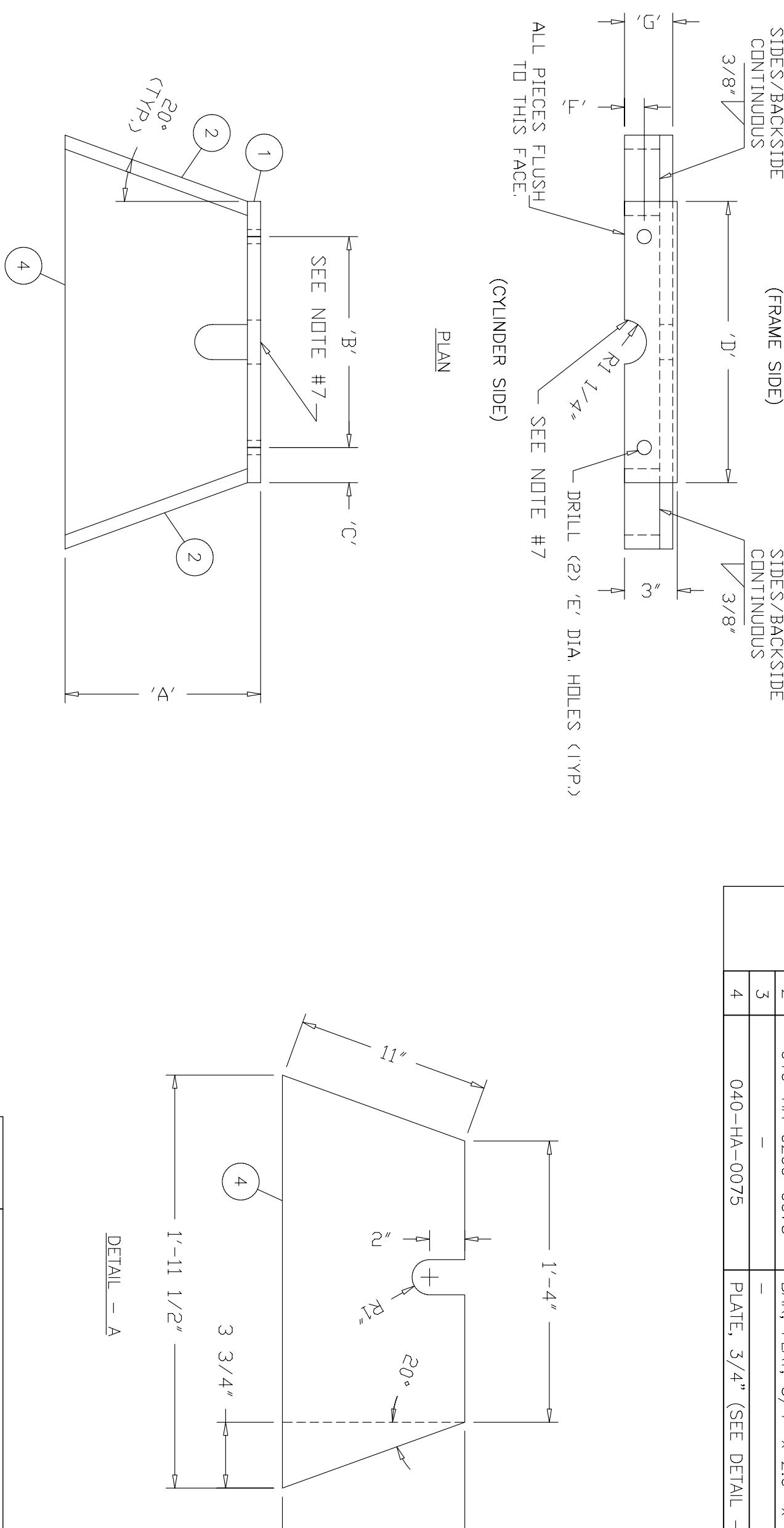
AIR-SIDE PERFORMANCE		FAN DATA		DRIVER DATA		STRUCTURAL	
46	Ambient Air Temp, In °F	104	Fan(s)	(1) MOORE-10K S30VT	Type	Guards	FAN
47	Elevation, Ft	1900	Blade Material	ALUMINUM	V-BELT BY OTHERS		DISCHARGE GUARD
48	Air Flow, SCFM	82,740	HP / Fan	21.3			SOFT BUG SCREENS
49	Outlet Air Temp, °F	128.9	Dia, In / # Blades	96 / 6			
50	Min Air Temp, °F	-20	RPM	418			
51			Tipspeed, FPM	10505			
52	Est. Noise Data:	85 dBA @1m, 64 dBA @ 15m	Pitch, Deg	19			
53	Additional Info:	*-BUILT TO STANDARD 150# DESIGN BUT DESIGNATED 14# TO AVOID THE NEED FOR PROVINCIAL REGISTRATION					
54							
55							
56							
57							
58							





BILL OF MATERIALS

MODEL	ITEM	PART#	DESCRIPTION	MATERIAL	QTY
JGJ	1	010-HA-0300-0075	BAR, FLAT, 3/4" x 3.0" x 16'L.G.	SA-36	1
	2	010-HA-0200-0075	BAR, FLAT, 3/4" x 2.0" x 11'L.G.	SA-36	2
	3	-	-	-	-
	4	040-HA-0075	PLATE, 3/4" (SEE DETAIL - A)	SA-36	1



NOTES:

- DIMENSION TOLERANCE $\pm 0.125"$
- ALL ASSEMBLIES TO BE TRIMMED TO FIT EACH INSTALL.
- USE GOOD WELDING PRACTICES – "JETROD" NOT ACCEPTABLE.
- LABEL EACH SUPPORT ACCORDINGLY.
- DIMENSIONS ALLOW FOR 1/4" SHIM.
- CUT TO FIT VARIOUS DRIVER CONFIGURATIONS.
- RADIUS CORNERS TO 1/8" TO REDUCE STRESS CONCENTRATIONS.

DIMENSION CHART (ALL DIMENSIONS SHOWN ARE IN INCHES)						
MODEL	A	B	C	D	E	F
JGJ	11.125"	12.0"	2.0"	16.0"	0.8125"	1.125"

PROPRIETARY MATERIAL: NOT TO BE REPRODUCED OR DISTRIBUTED WITHOUT THE PRIOR WRITTEN CONSENT OF TOROMONT ENERGY SYSTEMS INC.



10121 - BARLOW TRAIL N.E. CALGARY ALBERTA CANADA

TITLE:

CROSSHEAD SUPPORT STRUCTURE

FOR ARIEL JGJ, STD, S2C & L2C CROSSHEAD GUIDE

(FOR FRAMES WITH MAX 19 1/4" CENTERLINE TO TOS)

RELEASER BY: DATE RELEASED:

RELEASER BY: DRAWING NO.: JGJ-ALL CYLINDER SUPPORT REV.2

ELEVATION

DETAIL - A

Ariel Performance



Company: Toromont Energy Systems
 Quote: Q190393B
7.6.4.0 Case 1: Max

Customer: Husky Oil Operations Ltd
 Inquiry:
 Project: 31833 & 31834


Compressor Data:

Elevation,m:	575.00	Barmtr,bara:	0.945	Ambient,°C:	40.00	Type:	Nat. Gas
Frame:	JGJ/2	Stroke, mm:	88.90	Rod Dia, mm:	38.100	Mfg:	Waukesha
Max RL Tot, kN:	187	Max RL Tens, kN:	94	Max RL Comp, kN:	103	Model:	F18GL
Rated RPM:	1800	Rated kW:	462.3	Rated PS m/s:	5.3	kW:	298 (294)
Calc RPM:	1800.0	kW:	156	Calc PS m/s:	5.3	Avail:	279 (15)

Driver Data:

Services	1
Stage Data:	1
Flow Req'd, Nm3/h	0.000
Flow Calc, Nm3/h	5672.961
Cyl kW per Stage	151.5
Specific Gravity	0.9953
Ratio of Sp Ht (N)	1.3988
Comp Suct (Zs)	0.9947
Comp Disch (Zd)	0.9971
Pres Suct Line, barg	10.00
Pres Suct Flg, barg	9.66
Pres Disch Flg, barg	16.35
Pres Disch Line, barg	16.00
Pres Ratio F/F	1.631
Temp Suct, °C	48.89
Temp Clr Disch, °C	48.89
Cylinder Data:	Throw 1
Cyl Model	7-3/8RJ
Cyl Bore, mm	187.325
Cyl RDP (API), barg	53.0
Cyl MAWP, barg	58.3
Cyl Action	DBL
Cyl Disp, m3/h	518.3
Pres Suct Intl, barg	8.54
Temp Suct Intl, °C	51
Suct Zsph	0.9950
Pres Disch Intl, barg	18.12
Temp Disch Intl, °C	123
HE Suct Gas Vel, m/s	46
HE Disch Gas Vel, m/s	42
HE Spcrs Used/Max	0/2
HE Vol Pkt Avail, %	1.11+51.64
Vol Pkt Used, %	100.00 (V)
HE Min Clr, %	18.28
HE Total Clr, %	71.03
CE Suct Gas Vel, m/s	44
CE Disch Gas Vel, m/s	40
CE Spcrs Used/Max	0/2
CE Min Clr, %	19.21
CE Total Clr, %	19.21
Suct Vol Eff HE/CE, %	65.5/86.6
Disch Event HE/CE, ms	7.5/9.9
Suct Pseudo-Q HE/CE	11.4/10.5
Gas Rod Ld Comp, %	26.8 C
Gas Rod Ld Tens, %	26.0 T
Gas Rod Ld Total, %	27.7
Xhd Pin Deg/%Rvrsl kN	166/89.4
Flow Calc, Nm3/h	3478.571
Cyl kW	94.0
	57.5

Driver Data:

Ariel Performance



Company: Toromont Energy Systems
 Quote: Q190393B
7.6.4.0 Case 2: Min

Customer: Husky Oil Operations Ltd
 Inquiry:
 Project: 31833 & 31834


Compressor Data:

Elevation,m:	575.00	Barmtr,bara:	0.945	Ambient,°C:	40.00	Type:	Nat. Gas
Frame:	JGJ/2	Stroke, mm:	88.90	Rod Dia, mm:	38.100	Mfg:	Waukesha
Max RL Tot, kN:	187	Max RL Tens, kN:	94	Max RL Comp, kN:	103	Model:	F18GL
Rated RPM:	1800	Rated kW:	462.3	Rated PS m/s:	5.3	kW:	298 (294)
Calc RPM:	1800.0	kW:	144	Calc PS m/s:	5.3	Avail:	279 (15)

Driver Data:

Services	1
Stage Data:	1
Flow Req'd, Nm3/h	0.000
Flow Calc, Nm3/h	3900.374
Cyl kW per Stage	139.2
Specific Gravity	0.9944
Ratio of Sp Ht (N)	1.3973
Comp Suct (Zs)	0.9956
Comp Disch (Zd)	0.9986
Pres Suct Line, barg	8.00
Pres Suct Flg, barg	7.66
Pres Disch Flg, barg	16.35
Pres Disch Line, barg	16.00
Pres Ratio F/F	2.010
Temp Suct, °C	48.89
Temp Clr Disch, °C	48.89
Cylinder Data:	Throw 1
Cyl Model	7-3/8RJ
Cyl Bore, mm	187.325
Cyl RDP (API), barg	53.0
Cyl MAWP, barg	58.3
Cyl Action	DBL
Cyl Disp, m3/h	518.3
Pres Suct Intl, barg	6.75
Temp Suct Intl, °C	52
Suct Zsph	0.9958
Pres Disch Intl, barg	18.01
Temp Disch Intl, °C	147
HE Suct Gas Vel, m/s	46
HE Disch Gas Vel, m/s	42
HE Spcrs Used/Max	0/2
HE Vol Pkt Avail, %	1.11+51.64
Vol Pkt Used, %	100.00 (V)
HE Min Clr, %	18.28
HE Total Clr, %	71.03
CE Suct Gas Vel, m/s	44
CE Disch Gas Vel, m/s	40
CE Spcrs Used/Max	0/2
CE Min Clr, %	19.21
CE Total Clr, %	19.21
Suct Vol Eff HE/CE, %	49.3/82.0
Disch Event HE/CE, ms	5.7/8.8
Suct Pseudo-Q HE/CE	11.3/10.4
Gas Rod Ld Comp, %	31.2 C
Gas Rod Ld Tens, %	30.9 T
Gas Rod Ld Total, %	32.5
Xhd Pin Deg/%Rvrsl kN	171/89.7
Flow Calc, Nm3/h	2420.508
Cyl kW	87.2
	52.0

Driver Data:

Ariel Performance



Company: Toromont Energy Systems
 Quote: Q190393B
7.6.4.0 Case 3: Max

Customer: Husky Oil Operations Ltd
 Inquiry:
 Project: 31833 & 31834


Compressor Data:

Elevation,m:	575.00	Barmtr,bara:	0.945	Ambient,°C:	40.00	Type:	Nat. Gas
Frame:	JGJ/2	Stroke, mm:	88.90	Rod Dia, mm:	38.100	Mfg:	Waukesha
Max RL Tot, kN:	187	Max RL Tens, kN:	94	Max RL Comp, kN:	103	Model:	F18GL
Rated RPM:	1800	Rated kW:	462.3	Rated PS m/s:	5.3	kW:	298 (294)
Calc RPM:	1800.0	kW:	186	Calc PS m/s:	5.3	Avail:	279 (15)

Driver Data:

Services	1
Stage Data:	1
Flow Req'd, Nm3/h	0.000
Flow Calc, Nm3/h	5685.970
Cyl kW per Stage	181.2
Specific Gravity	0.9956
Ratio of Sp Ht (N)	1.3989
Comp Suct (Zs)	0.9943
Comp Disch (Zd)	0.9978
Pres Suct Line, barg	11.00
Pres Suct Flg, barg	10.66
Pres Disch Flg, barg	20.42
Pres Disch Line, barg	20.00
Pres Ratio F/F	1.842
Temp Suct, °C	48.89
Temp Clr Disch, °C	48.89
Cylinder Data:	Throw 1
Cyl Model	7-3/8RJ
Cyl Bore, mm	187.325
Cyl RDP (API), barg	53.0
Cyl MAWP, barg	58.3
Cyl Action	DBL
Cyl Disp, m3/h	518.3
Pres Suct Intl, barg	9.43
Temp Suct Intl, °C	52
Suct Zsph	0.9946
Pres Disch Intl, barg	22.54
Temp Disch Intl, °C	137
HE Suct Gas Vel, m/s	46
HE Disch Gas Vel, m/s	42
HE Spcrs Used/Max	0/2
HE Vol Pkt Avail, %	1.11+51.64
Vol Pkt Used, %	100.00 (V)
HE Min Clr, %	18.28
HE Total Clr, %	71.03
CE Suct Gas Vel, m/s	44
CE Disch Gas Vel, m/s	40
CE Spcrs Used/Max	0/2
CE Min Clr, %	19.21
CE Total Clr, %	19.21
Suct Vol Eff HE/CE, %	56.5/84.0
Disch Event HE/CE, ms	6.5/9.3
Suct Pseudo-Q HE/CE	11.4/10.5
Gas Rod Ld Comp, %	36.4 C
Gas Rod Ld Tens, %	35.8 T
Gas Rod Ld Total, %	37.9
Xhd Pin Deg/%Rvrsl kN	168/96.0
Flow Calc, Nm3/h	3508.167
Cyl kW	113.0
	68.2

Driver Data:

Ariel Performance



Company: Toromont Energy Systems
 Quote: Q190393B
7.6.4.0 Case 4: Min

Customer: Husky Oil Operations Ltd
 Inquiry:
 Project: 31833 & 31834



Compressor Data:

Elevation,m:	575.00	Barmtr,bara:	0.945	Ambient,°C:	40.00	Type:	Nat. Gas
Frame:	JGJ/2	Stroke, mm:	88.90	Rod Dia, mm:	38.100	Mfg:	Waukesha
Max RL Tot, kN:	187	Max RL Tens, kN:	94	Max RL Comp, kN:	103	Model:	F18GL
Rated RPM:	1800	Rated kW:	462.3	Rated PS m/s:	5.3	kW:	298 (294)
Calc RPM:	1800.0	kW:	170	Calc PS m/s:	5.3	Avail:	279 (15)

Disch Temp

Services 1

Stage Data: 1 ---

Flow Req'd, Nm3/h 0.000 ---

Flow Calc, Nm3/h 4371.736 ---

Cyl kW per Stage 166.0 ---

Specific Gravity 0.9951 ---

Ratio of Sp Ht (N) 1.3978 ---

Comp Suct (Zs) 0.9949 ---

Comp Disch (Zd) 0.9990 ---

Pres Suct Line, barg 9.50 ---

Pres Suct Flg, barg 9.16 ---

Pres Disch Flg, barg 20.42 ---

Pres Disch Line, barg 20.00 ---

Pres Ratio F/F 2.115 ---

Temp Suct, °C 48.89 ---

Temp Clr Disch, °C 48.89 ---

Cylinder Data: Throw 1 Throw 2

Cyl Model 7-3/8RJ 6-3/8RJ

Cyl Bore, mm 187.325 152.400

Cyl RDP (API), barg 53.0 79.6

Cyl MAWP, barg 58.3 87.6

Cyl Action DBL DBL

Cyl Disp, m3/h 518.3 339.3

Pres Suct Intl, barg 8.10 8.23

Temp Suct Intl, °C 53 52

Suct Zsph 0.9952 0.9952

Pres Disch Intl, barg 22.45 22.38

Temp Disch Intl, °C 153 151

HE Suct Gas Vel, m/s 46 43

HE Disch Gas Vel, m/s 42 41

HE Spcrs Used/Max 0/2 0/0

HE Vol Pkt Avail, % 1.11+51.64 1.29+64.48

Vol Pkt Used, % 100.00 (V) 100.00 (V)

HE Min Clr, % 18.28 18.00

HE Total Clr, % 71.03 83.77

CE Suct Gas Vel, m/s 44 40

CE Disch Gas Vel, m/s 40 39

CE Spcrs Used/Max 0/2 0/0

CE Min Clr, % 19.21 19.41

CE Total Clr, % 19.21 19.41

Suct Vol Eff HE/CE, % 45.1/80.8 36.3/80.6

Disch Event HE/CE, ms 5.4/8.5 4.7/8.5

Suct Pseudo-Q HE/CE 11.2/10.4 8.8/8.3

Gas Rod Ld Comp, % 39.7 C 26.3 C

Gas Rod Ld Tens, % 39.5 T 24.8 T

Gas Rod Ld Total, % 41.5 26.8

Xhd Pin Deg/%Rvrsl kN 172/95.3 170/82.2

Flow Calc, Nm3/h 2723.324 1648.413

Cyl kW 104.4 61.6

Driver Data:

Type:	Nat. Gas
Mfg:	Waukesha
Model:	F18GL
kW:	298 (294)
Avail:	279 (15)



Ariel Performance

7.6.4.0

Company: Toromont Energy Systems
 Quote: Q190393B
 Case 5:

Customer: Husky Oil Operations Ltd
 Inquiry:
 Project: 31833 & 31834


Compressor Data:

Elevation,m:	575.00	Barmtr,bara:	0.945	Ambient,°C:	40.00	Type:	Nat. Gas
Frame:	JGJ/2	Stroke, mm:	88.90	Rod Dia, mm:	38.100	Mfg:	Waukesha
Max RL Tot, kN:	187	Max RL Tens, kN:	94	Max RL Comp, kN:	103	Model:	F18GL
Rated RPM:	1800	Rated kW:	462.3	Rated PS m/s:	5.3	kW:	298 (294)
Calc RPM:	1800.0	kW:	179	Calc PS m/s:	5.3	Avail:	279 (15)

Driver Data:
Services
1

	1	2
Flow Req'd, Nm3/h	0.000	0.000
Flow Calc, Nm3/h	4440.256	4426.965
Cyl kW per Stage	104.1	70.3
Specific Gravity	0.9956	0.9967
Ratio of Sp Ht (N)	1.3995	1.4020
Comp Suct (Zs)	0.9943	0.9922
Comp Disch (Zd)	0.9963	0.9938
Pres Suct Line, barg	11.00	N/A
Pres Suct Flg, barg	10.66	16.04
Pres Disch Flg, barg	16.38	20.42
Pres Disch Line, barg	N/A	20.00
Pres Ratio F/F	1.494	1.258
Temp Suct, °C	48.89	48.89
Temp Clr Disch, °C	48.89	48.89
Cylinder Data:	Throw 1	Throw 2
Cyl Model	7-3/8RJ	6-3/8RJ
Cyl Bore, mm	187.325	152.400
Cyl RDP (API), barg	53.0	79.6
Cyl MAWP, barg	58.3	87.6
Cyl Action	DBL	DBL
Cyl Disp, m3/h	518.3	339.3
Pres Suct Intl, barg	9.43	14.47
Temp Suct Intl, °C	51	50
Suct Zsph	0.9945	0.9924
Pres Disch Intl, barg	18.21	22.71
Temp Disch Intl, °C	113	93
HE Suct Gas Vel, m/s	46	43
HE Disch Gas Vel, m/s	42	41
HE Spcrs Used/Max	0/2	0/0
HE Vol Pkt Avail, %	1.11+51.64	1.29+64.48
Vol Pkt Used, %	0.00 (V)	0.00 (V)
HE Min Clr, %	18.28	18.00
HE Total Clr, %	19.38	19.29
CE Suct Gas Vel, m/s	44	40
CE Disch Gas Vel, m/s	40	39
CE Spcrs Used/Max	0/2	0/0
CE Min Clr, %	19.21	19.41
CE Total Clr, %	19.21	19.41
Suct Vol Eff HE/CE, %	88.3/88.4	91.5/91.5
Disch Event HE/CE, ms	9.6/10.6	11.0/11.8
Suct Pseudo-Q HE/CE	11.4/10.5	9.5/8.4
Gas Rod Ld Comp, %	24.8 C	16.4 C
Gas Rod Ld Tens, %	23.6 T	13.3 T
Gas Rod Ld Total, %	25.4	15.6
Xhd Pin Deg/%Rvrsl kN	165/90.6	161/82.3
Flow Calc, Nm3/h	4440.256	4426.965
Cyl kW	104.1	70.3





Ariel Performance

7.6.4.0

Company: Toromont Energy Systems
 Quote: Q190393B
 Case 6:

Customer: Husky Oil Operations Ltd
 Inquiry:
 Project: 31833 & 31834


Compressor Data:

Elevation,m:	575.00	Barmtr,bara:	0.945	Ambient,°C:	40.00	Type:	Nat. Gas
Frame:	JGJ/2	Stroke, mm:	88.90	Rod Dia, mm:	38.100	Mfg:	Waukesha
Max RL Tot, kN:	187	Max RL Tens, kN:	94	Max RL Comp, kN:	103	Model:	F18GL
Rated RPM:	1800	Rated kW:	462.3	Rated PS m/s:	5.3	kW:	298 (294)
Calc RPM:	1800.0	kW:	175	Calc PS m/s:	5.3	Avail:	279 (15)

Driver Data:
Services
1
Stage Data:
1**2**

Flow Req'd, Nm3/h	0.000	0.000
Flow Calc, Nm3/h	3645.451	3631.626
Cyl kW per Stage	90.0	81.0
Specific Gravity	0.9949	0.9963
Ratio of Sp Ht (N)	1.3985	1.4006
Comp Suct (Zs)	0.9951	0.9932
Comp Disch (Zd)	0.9969	0.9957
Pres Suct Line, barg	9.00	N/A
Pres Suct Flg, barg	8.66	13.49
Pres Disch Flg, barg	13.83	20.42
Pres Disch Line, barg	N/A	20.00
Pres Ratio F/F	1.539	1.480
Temp Suct, °C	48.89	48.89
Temp Clr Disch, °C	48.89	48.89
Cylinder Data:	Throw 1	Throw 2
Cyl Model	7-3/8RJ	6-3/8RJ
Cyl Bore, mm	187.325	152.400
Cyl RDP (API), barg	53.0	79.6
Cyl MAWP, barg	58.3	87.6
Cyl Action	DBL	DBL
Cyl Disp, m3/h	518.3	339.3
Pres Suct Intl, barg	7.64	12.16
Temp Suct Intl, °C	51	51
Suct Zspf	0.9953	0.9934
Pres Disch Intl, barg	15.38	22.60
Temp Disch Intl, °C	116	110
HE Suct Gas Vel, m/s	46	43
HE Disch Gas Vel, m/s	42	41
HE Spcrs Used/Max	0/2	0/0
HE Vol Pkt Avail, %	1.11+51.64	1.29+64.48
Vol Pkt Used, %	0.00 (V)	0.00 (V)
HE Min Clr, %	18.28	18.00
HE Total Clr, %	19.38	19.29
CE Suct Gas Vel, m/s	44	40
CE Disch Gas Vel, m/s	40	39
CE Spcrs Used/Max	0/2	0/0
CE Min Clr, %	19.21	19.41
CE Total Clr, %	19.21	19.41
Suct Vol Eff HE/CE, %	87.7/87.8	88.5/88.5
Disch Event HE/CE, ms	9.4/10.3	9.7/10.6
Suct Pseudo-Q HE/CE	11.4/10.5	9.5/8.3
Gas Rod Ld Comp, %	21.7 C	20.1 C
Gas Rod Ld Tens, %	20.9 T	17.6 T
Gas Rod Ld Total, %	22.3	19.8
Xhd Pin Deg/%Rvrsl kN	164/85.4	162/85.1
Flow Calc, Nm3/h	3645.451	3631.626
Cyl kW	90.0	81.0

Ariel Performance



Company: Toromont Energy Systems
 Quote: Q190393B
7.6.4.0 Case 7: Max

Customer: Husky Oil Operations Ltd
 Inquiry:
 Project: 31833 & 31834


Compressor Data:

Elevation,m:	575.00	Barmtr,bara:	0.945	Ambient,°C:	40.00	Type:	Nat. Gas
Frame:	JGJ/2	Stroke, mm:	88.90	Rod Dia, mm:	38.100	Mfg:	Waukesha
Max RL Tot, kN:	187	Max RL Tens, kN:	94	Max RL Comp, kN:	103	Model:	F18GL
Rated RPM:	1800	Rated kW:	462.3	Rated PS m/s:	5.3	kW:	298 (294)
Calc RPM:	1800.0	kW:	212	Calc PS m/s:	5.3	Avail:	279 (15)

Driver Data:
Services
1

	1	2
Flow Req'd, Nm3/h	0.000	0.000
Flow Calc, Nm3/h	5209.182	5195.702
Cyl kW per Stage	122.4	85.1
Specific Gravity	0.9961	0.9971
Ratio of Sp Ht (N)	1.4003	1.4030
Comp Suct (Zs)	0.9935	0.9911
Comp Disch (Zd)	0.9960	0.9932
Pres Suct Line, barg	13.00	N/A
Pres Suct Flg, barg	12.66	19.01
Pres Disch Flg, barg	19.39	24.50
Pres Disch Line, barg	N/A	24.00
Pres Ratio F/F	1.495	1.275
Temp Suct, °C	48.89	48.89
Temp Clr Disch, °C	48.89	48.89
Cylinder Data:	Throw 1	Throw 2
Cyl Model	7-3/8RJ	6-3/8RJ
Cyl Bore, mm	187.325	152.400
Cyl RDP (API), barg	53.0	79.6
Cyl MAWP, barg	58.3	87.6
Cyl Action	DBL	DBL
Cyl Disp, m3/h	518.3	339.3
Pres Suct Intl, barg	11.22	17.17
Temp Suct Intl, °C	51	50
Suct Zsph	0.9938	0.9914
Pres Disch Intl, barg	21.54	27.22
Temp Disch Intl, °C	113	94
HE Suct Gas Vel, m/s	46	43
HE Disch Gas Vel, m/s	42	41
HE Spcrs Used/Max	0/2	0/0
HE Vol Pkt Avail, %	1.11+51.64	1.29+64.48
Vol Pkt Used, %	0.00 (V)	0.00 (V)
HE Min Clr, %	18.28	18.00
HE Total Clr, %	19.38	19.29
CE Suct Gas Vel, m/s	44	40
CE Disch Gas Vel, m/s	40	39
CE Spcrs Used/Max	0/2	0/0
CE Min Clr, %	19.21	19.41
CE Total Clr, %	19.21	19.41
Suct Vol Eff HE/CE, %	88.3/88.4	91.3/91.2
Disch Event HE/CE, ms	9.6/10.5	10.9/11.7
Suct Pseudo-Q HE/CE	11.4/10.5	9.5/8.4
Gas Rod Ld Comp, %	29.1 C	19.9 C
Gas Rod Ld Tens, %	27.7 T	16.2 T
Gas Rod Ld Total, %	29.8	19.0
Xhd Pin Deg/%Rvrsl kN	165/95.6	162/87.4
Flow Calc, Nm3/h	5209.182	5195.702
Cyl kW	122.4	85.1



Ariel Performance



Company: Toromont Energy Systems
 Quote: Q190393B
7.6.4.0 Case 8: Min

Customer: Husky Oil Operations Ltd
 Inquiry:
 Project: 31833 & 31834



Compressor Data:

Elevation,m:	575.00	Barmtr,bara:	0.945	Ambient,°C:	40.00	Type:	Nat. Gas
Frame:	JGJ/2	Stroke, mm:	88.90	Rod Dia, mm:	38.100	Mfg:	Waukesha
Max RL Tot, kN:	187	Max RL Tens, kN:	94	Max RL Comp, kN:	103	Model:	F18GL
Rated RPM:	1800	Rated kW:	462.3	Rated PS m/s:	5.3	kW:	298 (294)
Calc RPM:	1800.0	kW:	200	Calc PS m/s:	5.3	Avail:	279 (15)

Services

1

Stage Data:	1	2
Flow Req'd, Nm3/h	0.000	0.000
Flow Calc, Nm3/h	3621.139	3606.615
Cyl kW per Stage	93.5	102.3
Specific Gravity	0.9949	0.9964
Ratio of Sp Ht (N)	1.3985	1.4004
Comp Suct (Zs)	0.9951	0.9930
Comp Disch (Zd)	0.9971	0.9970
Pres Suct Line, barg	9.00	N/A
Pres Suct Flg, barg	8.66	13.91
Pres Disch Flg, barg	14.25	24.50
Pres Disch Line, barg	N/A	24.00
Pres Ratio F/F	1.583	1.713
Temp Suct, °C	48.89	48.89
Temp Clr Disch, °C	48.89	48.89

Cylinder Data:

Throw 1 **Throw 2**

Cyl Model	7-3/8RJ	6-3/8RJ
Cyl Bore, mm	187.325	152.400
Cyl RDP (API), barg	53.0	79.6
Cyl MAWP, barg	58.3	87.6
Cyl Action	DBL	DBL
Cyl Disp, m3/h	518.3	339.3
Pres Suct Intl, barg	7.64	12.54
Temp Suct Intl, °C	51	51
Suct Zsph	0.9953	0.9933
Pres Disch Intl, barg	15.82	26.99
Temp Disch Intl, °C	120	127
HE Suct Gas Vel, m/s	46	43
HE Disch Gas Vel, m/s	42	41
HE Spcrs Used/Max	0/2	0/0
HE Vol Pkt Avail, %	1.11+51.64	1.29+64.48
Vol Pkt Used, %	0.00 (V)	0.00 (V)
HE Min Clr, %	18.28	18.00
HE Total Clr, %	19.38	19.29
CE Suct Gas Vel, m/s	44	40
CE Disch Gas Vel, m/s	40	39
CE Spcrs Used/Max	0/2	0/0
CE Min Clr, %	19.21	19.41
CE Total Clr, %	19.21	19.41
Suct Vol Eff HE/CE, %	87.2/87.2	85.6/85.6
Disch Event HE/CE, ms	9.3/10.1	8.8/9.6
Suct Pseudo-Q HE/CE	11.4/10.5	9.5/8.3
Gas Rod Ld Comp, %	22.9 C	27.3 C
Gas Rod Ld Tens, %	22.1 T	24.8 T
Gas Rod Ld Total, %	23.6	27.3
Xhd Pin Deg/%Rvrsl kN	165/86.6	164/92.5
Flow Calc, Nm3/h	3621.139	3606.615
Cyl kW	93.5	102.3

Driver Data:

Type:	Nat. Gas
Mfg:	Waukesha
Model:	F18GL
kW:	298 (294)
Avail:	279 (15)

Ariel Performance



Company: Toromont Energy Systems
 Quote: Q190393B
7.6.4.0 Case 9: Max

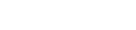
Customer: Husky Oil Operations Ltd
 Inquiry:
 Project: 31833 & 31834


Compressor Data:

Elevation,m:	575.00	Barmtr,bara:	0.945	Ambient,°C:	40.00	Type:	Nat. Gas
Frame:	JGJ/2	Stroke, mm:	88.90	Rod Dia, mm:	38.100	Mfg:	Waukesha
Max RL Tot, kN:	187	Max RL Tens, kN:	94	Max RL Comp, kN:	103	Model:	F18GL
Rated RPM:	1800	Rated kW:	462.3	Rated PS m/s:	5.3	kW:	298 (294)
Calc RPM:	1800.0	kW:	242	Calc PS m/s:	5.3	Avail:	279 (15)

Driver Data:
Services
1

	1	2
Flow Req'd, Nm3/h	0.000	0.000
Flow Calc, Nm3/h	5182.314	5168.261
Cyl kW per Stage	126.4	110.6
Specific Gravity	0.9961	0.9971
Ratio of Sp Ht (N)	1.4003	1.4028
Comp Suct (Zs)	0.9935	0.9910
Comp Disch (Zd)	0.9961	0.9945
Pres Suct Line, barg	13.00	N/A
Pres Suct Flg, barg	12.66	19.44
Pres Disch Flg, barg	19.85	28.58
Pres Disch Line, barg	N/A	28.00
Pres Ratio F/F	1.529	1.448
Temp Suct, °C	48.89	48.89
Temp Clr Disch, °C	48.89	48.89
Cylinder Data:	Throw 1	Throw 2
Cyl Model	7-3/8RJ	6-3/8RJ
Cyl Bore, mm	187.325	152.400
Cyl RDP (API), barg	53.0	79.6
Cyl MAWP, barg	58.3	87.6
Cyl Action	DBL	DBL
Cyl Disp, m3/h	518.3	339.3
Pres Suct Intl, barg	11.22	17.56
Temp Suct Intl, °C	51	51
Suct Zsph	0.9938	0.9913
Pres Disch Intl, barg	22.03	31.62
Temp Disch Intl, °C	116	108
HE Suct Gas Vel, m/s	46	43
HE Disch Gas Vel, m/s	42	41
HE Spcrs Used/Max	0/2	0/0
HE Vol Pkt Avail, %	1.11+51.64	1.29+64.48
Vol Pkt Used, %	0.00 (V)	0.00 (V)
HE Min Clr, %	18.28	18.00
HE Total Clr, %	19.38	19.29
CE Suct Gas Vel, m/s	44	40
CE Disch Gas Vel, m/s	40	39
CE Spcrs Used/Max	0/2	0/0
CE Min Clr, %	19.21	19.41
CE Total Clr, %	19.21	19.41
Suct Vol Eff HE/CE, %	87.9/87.9	89.0/89.0
Disch Event HE/CE, ms	9.4/10.4	9.9/10.7
Suct Pseudo-Q HE/CE	11.4/10.5	9.5/8.4
Gas Rod Ld Comp, %	30.4 C	27.1 C
Gas Rod Ld Tens, %	29.1 T	23.5 T
Gas Rod Ld Total, %	31.2	26.6
Xhd Pin Deg/%Rvrsl kN	167/96.7	163/95.7
Flow Calc, Nm3/h	5182.314	5168.261
Cyl kW	126.4	110.6





Ariel Performance

Company: Toromont Energy Systems
 Quote: Q190393B
7.6.4.0 Case 10: Min

Customer: Husky Oil Operations Ltd
 Inquiry:
 Project: 31833 & 31834


Compressor Data:

Elevation,m:	575.00	Barmtr,bara:	0.945	Ambient,°C:	40.00	Type:	Nat. Gas
Frame:	JGJ/2	Stroke, mm:	88.90	Rod Dia, mm:	38.100	Mfg:	Waukesha
Max RL Tot, kN:	187	Max RL Tens, kN:	94	Max RL Comp, kN:	103	Model:	F18GL
Rated RPM:	1800	Rated kW:	462.3	Rated PS m/s:	5.3	kW:	298 (294)
Calc RPM:	1800.0	kW:	222	Calc PS m/s:	5.3	Avail:	279 (15)

Driver Data:

Services	1	2
Stage Data:	1	2
Flow Req'd, Nm3/h	0.000	0.000
Flow Calc, Nm3/h	3597.825	3582.690
Cyl kW per Stage	96.9	120.5
Specific Gravity	0.9949	0.9964
Ratio of Sp Ht (N)	1.3984	1.4001
Comp Suct (Zs)	0.9951	0.9929
Comp Disch (Zd)	0.9972	0.9983
Pres Suct Line, barg	9.00	N/A
Pres Suct Flg, barg	8.66	14.31
Pres Disch Flg, barg	14.65	28.58
Pres Disch Line, barg	N/A	28.00
Pres Ratio F/F	1.625	1.936
Temp Suct, °C	48.89	48.89
Temp Clr Disch, °C	48.89	48.89
Cylinder Data:	Throw 1	Throw 2
Cyl Model	7-3/8RJ	6-3/8RJ
Cyl Bore, mm	187.325	152.400
Cyl RDP (API), barg	53.0	79.6
Cyl MAWP, barg	58.3	87.6
Cyl Action	DBL	DBL
Cyl Disp, m3/h	518.3	339.3
Pres Suct Intl, barg	7.65	12.91
Temp Suct Intl, °C	51	52
Suct Zsph	0.9953	0.9933
Pres Disch Intl, barg	16.26	31.37
Temp Disch Intl, °C	122	141
HE Suct Gas Vel, m/s	46	43
HE Disch Gas Vel, m/s	42	41
HE Spcrs Used/Max	0/2	0/0
HE Vol Pkt Avail, %	1.11+51.64	1.29+64.48
Vol Pkt Used, %	0.00 (V)	0.00 (V)
HE Min Clr, %	18.28	18.00
HE Total Clr, %	19.38	19.29
CE Suct Gas Vel, m/s	44	40
CE Disch Gas Vel, m/s	40	39
CE Spcrs Used/Max	0/2	0/0
CE Min Clr, %	19.21	19.41
CE Total Clr, %	19.21	19.41
Suct Vol Eff HE/CE, %	86.6/86.7	82.9/82.8
Disch Event HE/CE, ms	9.1/10.0	8.1/9.0
Suct Pseudo-Q HE/CE	11.4/10.5	9.5/8.3
Gas Rod Ld Comp, %	24.1 C	34.5 C
Gas Rod Ld Tens, %	23.4 T	32.1 T
Gas Rod Ld Total, %	24.9	34.9
Xhd Pin Deg/%Rvrsl kN	165/87.7	166/98.5
Flow Calc, Nm3/h	3597.825	3582.690
Cyl kW	96.9	120.5

Ariel Performance



Company: Toromont Energy Systems
 Quote: Q190393B
7.6.4.0 Case 11: Max

Customer: Husky Oil Operations Ltd
 Inquiry:
 Project: 31833 & 31834


Compressor Data:

Elevation,m:	575.00	Barmtr,bara:	0.945	Ambient,°C:	40.00	Type:	Nat. Gas
Frame:	JGJ/2	Stroke, mm:	88.90	Rod Dia, mm:	38.100	Mfg:	Waukesha
Max RL Tot, kN:	187	Max RL Tens, kN:	94	Max RL Comp, kN:	103	Model:	F18GL
Rated RPM:	1800	Rated kW:	462.3	Rated PS m/s:	5.3	kW:	298 (294)
Calc RPM:	1800.0	kW:	268	Calc PS m/s:	5.3	Avail:	279 (15)

Driver Data:
Services
1

	1	2
Flow Req'd, Nm3/h	0.000	0.000
Flow Calc, Nm3/h	5156.543	5141.968
Cyl kW per Stage	130.3	133.0
Specific Gravity	0.9961	0.9972
Ratio of Sp Ht (N)	1.4002	1.4026
Comp Suct (Zs)	0.9935	0.9908
Comp Disch (Zd)	0.9963	0.9959
Pres Suct Line, barg	13.00	N/A
Pres Suct Flg, barg	12.66	19.86
Pres Disch Flg, barg	20.29	32.66
Pres Disch Line, barg	N/A	32.00
Pres Ratio F/F	1.561	1.615
Temp Suct, °C	48.89	48.89
Temp Clr Disch, °C	48.89	48.89
Cylinder Data:	Throw 1	Throw 2
Cyl Model	7-3/8RJ	6-3/8RJ
Cyl Bore, mm	187.325	152.400
Cyl RDP (API), barg	53.0	79.6
Cyl MAWP, barg	58.3	87.6
Cyl Action	DBL	DBL
Cyl Disp, m3/h	518.3	339.3
Pres Suct Intl, barg	11.22	17.94
Temp Suct Intl, °C	51	51
Suct Zsph	0.9938	0.9912
Pres Disch Intl, barg	22.50	36.01
Temp Disch Intl, °C	118	120
HE Suct Gas Vel, m/s	46	43
HE Disch Gas Vel, m/s	42	41
HE Spcrs Used/Max	0/2	0/0
HE Vol Pkt Avail, %	1.11+51.64	1.29+64.48
Vol Pkt Used, %	0.00 (V)	0.00 (V)
HE Min Clr, %	18.28	18.00
HE Total Clr, %	19.38	19.29
CE Suct Gas Vel, m/s	44	40
CE Disch Gas Vel, m/s	40	39
CE Spcrs Used/Max	0/2	0/0
CE Min Clr, %	19.21	19.41
CE Total Clr, %	19.21	19.41
Suct Vol Eff HE/CE, %	87.5/87.5	86.9/86.8
Disch Event HE/CE, ms	9.4/10.2	9.2/10.0
Suct Pseudo-Q HE/CE	11.4/10.5	9.5/8.4
Gas Rod Ld Comp, %	31.7 C	34.4 C
Gas Rod Ld Tens, %	30.5 T	30.8 T
Gas Rod Ld Total, %	32.6	34.2
Xhd Pin Deg/%Rvrsl kN	167/97.7	165/98.7
Flow Calc, Nm3/h	5156.543	5141.968
Cyl kW	130.3	133.0



Ariel Performance

7.6.4.0

Company: Toromont Energy Systems
 Quote: Q190393B
 Case 12: Min

Customer: Husky Oil Operations Ltd
 Inquiry:
 Project: 31833 & 31834


Compressor Data:

Elevation,m:	575.00	Barmtr,bara:	0.945	Ambient,°C:	40.00	Type:	Nat. Gas
Frame:	JGJ/2	Stroke, mm:	88.90	Rod Dia, mm:	38.100	Mfg:	Waukesha
Max RL Tot, kN:	187	Max RL Tens, kN:	94	Max RL Comp, kN:	103	Model:	F18GL
Rated RPM:	1800	Rated kW:	462.3	Rated PS m/s:	5.3	kW:	298 (294)
Calc RPM:	1800.0	kW:	241	Calc PS m/s:	5.3	Avail:	279 (15)

Disch Temp

Services	1	2	Driver Data:
Stage Data:	1	2	
Flow Req'd, Nm3/h	0.000	0.000	
Flow Calc, Nm3/h	3574.283	3558.573	
Cyl kW per Stage	100.1	136.3	
Specific Gravity	0.9949	0.9965	
Ratio of Sp Ht (N)	1.3983	1.3999	
Comp Suct (Zs)	0.9951	0.9927	
Comp Disch (Zd)	0.9974	0.9996	
Pres Suct Line, barg	9.00	N/A	
Pres Suct Flg, barg	8.66	14.70	
Pres Disch Flg, barg	15.06	32.66	
Pres Disch Line, barg	N/A	32.00	
Pres Ratio F/F	1.667	2.149	
Temp Suct, °C	48.89	48.89	
Temp Clr Disch, °C	48.89	48.89	
Cylinder Data:	Throw 1	Throw 2	
Cyl Model	7-3/8RJ	6-3/8RJ	
Cyl Bore, mm	187.325	152.400	
Cyl RDP (API), barg	53.0	79.6	
Cyl MAWP, barg	58.3	87.6	
Cyl Action	DBL	DBL	
Cyl Disp, m3/h	518.3	339.3	
Pres Suct Intl, barg	7.65	13.26	
Temp Suct Intl, °C	52	52	
Suct Zsph	0.9954	0.9932	
Pres Disch Intl, barg	16.70	35.73	
Temp Disch Intl, °C	125	<u>154</u>	
HE Suct Gas Vel, m/s	46	43	
HE Disch Gas Vel, m/s	42	41	
HE Spcrs Used/Max	0/2	0/0	
HE Vol Pkt Avail, %	1.11+51.64	1.29+64.48	
Vol Pkt Used, %	0.00 (V)	0.00 (V)	
HE Min Clr, %	18.28	18.00	
HE Total Clr, %	19.38	19.29	
CE Suct Gas Vel, m/s	44	40	
CE Disch Gas Vel, m/s	40	39	
CE Spcrs Used/Max	0/2	0/0	
CE Min Clr, %	19.21	19.41	
CE Total Clr, %	19.21	19.41	
Suct Vol Eff HE/CE, %	86.1/86.2	80.4/80.3	
Disch Event HE/CE, ms	8.9/9.8	7.5/8.4	
Suct Pseudo-Q HE/CE	11.4/10.5	9.5/8.3	
Gas Rod Ld Comp, %	25.3 C	41.7 C	
Gas Rod Ld Tens, %	24.6 T	39.4 T	
Gas Rod Ld Total, %	26.2	42.5	
Xhd Pin Deg/%Rvrsl kN	165/88.8	168/95.9	
Flow Calc, Nm3/h	3574.283	3558.573	
Cyl kW	100.1	136.3	

Driver Data:

Ariel Performance



Company: Toromont Energy Systems
 Quote: Q190393B
7.6.4.0 Case 13: Max

Customer: Husky Oil Operations Ltd
 Inquiry:
 Project: 31833 & 31834


Compressor Data:

Elevation,m:	575.00	Barmtr,bara:	0.945	Ambient,°C:	40.00	Type:	Nat. Gas
Frame:	JGJ/2	Stroke, mm:	88.90	Rod Dia, mm:	38.100	Mfg:	Waukesha
Max RL Tot, kN:	187	Max RL Tens, kN:	94	Max RL Comp, kN:	103	Model:	F18GL
Rated RPM:	1800	Rated kW:	462.3	Rated PS m/s:	5.3	kW:	298 (294)
Calc RPM:	1800.0	kW:	279	Calc PS m/s:	5.3	Avail:	279 (15)

Driver Data:

Services	1	2
Stage Data:	1	2
Flow Req'd, Nm3/h	0.000	0.000
Flow Calc, Nm3/h	5024.525	5010.378
Cyl kW per Stage	126.5	148.6
Specific Gravity	0.9961	0.9972
Ratio of Sp Ht (N)	1.4002	1.4023
Comp Suct (Zs)	0.9935	0.9909
Comp Disch (Zd)	0.9963	0.9972
Pres Suct Line, barg	13.00	N/A
Pres Suct Flg, barg	12.66	19.79
Pres Disch Flg, barg	20.25	35.69
Pres Disch Line, barg	N/A	35.00
Pres Ratio F/F	1.558	1.767
Temp Suct, °C	48.89	48.89
Temp Clr Disch, °C	48.89	48.89
Cylinder Data:	Throw 1	Throw 2
Cyl Model	7-3/8RJ	6-3/8RJ
Cyl Bore, mm	187.325	152.400
Cyl RDP (API), barg	53.0	79.6
Cyl MAWP, barg	58.3	87.6
Cyl Action	DBL	DBL
Cyl Disp, m3/h	518.3	339.3
Pres Suct Intl, barg	11.22	17.88
Temp Suct Intl, °C	51	51
Suct Zsph	0.9938	0.9913
Pres Disch Intl, barg	22.45	39.25
Temp Disch Intl, °C	118	130
HE Suct Gas Vel, m/s	46	43
HE Disch Gas Vel, m/s	42	41
HE Spcrs Used/Max	0/2	0/0
HE Vol Pkt Avail, %	1.11+51.64	1.29+64.48
Vol Pkt Used, %	23.98 (V)	0.00 (V)
HE Min Clr, %	18.28	18.00
HE Total Clr, %	31.77	19.29
CE Suct Gas Vel, m/s	44	40
CE Disch Gas Vel, m/s	40	39
CE Spcrs Used/Max	0/2	0/0
CE Min Clr, %	19.21	19.41
CE Total Clr, %	19.21	19.41
Suct Vol Eff HE/CE, %	83.0/87.6	85.0/85.0
Disch Event HE/CE, ms	9.0/10.3	8.6/9.4
Suct Pseudo-Q HE/CE	11.4/10.5	9.5/8.4
Gas Rod Ld Comp, %	31.6 C	40.3 C
Gas Rod Ld Tens, %	30.3 T	36.8 T
Gas Rod Ld Total, %	32.5	40.4
Xhd Pin Deg/%Rvrsl kN	167/98.5	167/94.9
Flow Calc, Nm3/h	5024.525	5010.378
Cyl kW	126.5	148.6

Ariel Performance



Company: Toromont Energy Systems
 Quote: Q190393B
7.6.4.0
 Case 14: Min

Customer: Husky Oil Operations Ltd
 Inquiry:
 Project: 31833 & 31834



Compressor Data:

Elevation,m:	575.00	Barmtr,bara:	0.945	Ambient,°C:	40.00	Type:	Nat. Gas
Frame:	JGJ/2	Stroke, mm:	88.90	Rod Dia, mm:	38.100	Mfg:	Waukesha
Max RL Tot, kN:	187	Max RL Tens, kN:	94	Max RL Comp, kN:	103	Model:	F18GL
Rated RPM:	1800	Rated kW:	462.3	Rated PS m/s:	5.3	kW:	298 (294)
Calc RPM:	1800.0	kW:	264	Calc PS m/s:	5.3	Avail:	279 (15)

Disch Temp

Services	1	2
Stage Data:	1	2
Flow Req'd, Nm3/h	0.000	0.000
Flow Calc, Nm3/h	3951.352	3935.580
Cyl kW per Stage	110.2	148.9
Specific Gravity	0.9953	0.9968
Ratio of Sp Ht (N)	1.3987	1.4005
Comp Suct (Zs)	0.9947	0.9921
Comp Disch (Zd)	0.9972	0.9997
Pres Suct Line, barg	10.00	N/A
Pres Suct Flg, barg	9.66	16.28
Pres Disch Flg, barg	16.68	35.69
Pres Disch Line, barg	N/A	35.00
Pres Ratio F/F	1.663	2.127
Temp Suct, °C	48.89	48.89
Temp Clr Disch, °C	48.89	48.89
Cylinder Data:	Throw 1	Throw 2
Cyl Model	7-3/8RJ	6-3/8RJ
Cyl Bore, mm	187.325	152.400
Cyl RDP (API), barg	53.0	79.6
Cyl MAWP, barg	58.3	87.6
Cyl Action	DBL	DBL
Cyl Disp, m3/h	518.3	339.3
Pres Suct Intl, barg	8.54	14.70
Temp Suct Intl, °C	52	52
Suct Zsph	0.9950	0.9926
Pres Disch Intl, barg	18.48	39.05
Temp Disch Intl, °C	125	<u>152</u>
HE Suct Gas Vel, m/s	46	43
HE Disch Gas Vel, m/s	42	41
HE Spcrs Used/Max	0/2	0/0
HE Vol Pkt Avail, %	1.11+51.64	1.29+64.48
Vol Pkt Used, %	0.00 (V)	0.00 (V)
HE Min Clr, %	18.28	18.00
HE Total Clr, %	19.38	19.29
CE Suct Gas Vel, m/s	44	40
CE Disch Gas Vel, m/s	40	39
CE Spcrs Used/Max	0/2	0/0
CE Min Clr, %	19.21	19.41
CE Total Clr, %	19.21	19.41
Suct Vol Eff HE/CE, %	86.1/86.2	80.7/80.6
Disch Event HE/CE, ms	8.9/9.8	7.6/8.5
Suct Pseudo-Q HE/CE	11.4/10.5	9.5/8.3
Gas Rod Ld Comp, %	27.8 C	45.2 C
Gas Rod Ld Tens, %	27.0 T	42.6 T
Gas Rod Ld Total, %	28.7	46.1
Xhd Pin Deg/%Rvrsl kN	165/92.3	170/92.4
Flow Calc, Nm3/h	3951.351	3935.580
Cyl kW	110.2	148.9

Driver Data:

Type:	Nat. Gas
Mfg:	Waukesha
Model:	F18GL